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Monterey, California. Naval Postgraduate School

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Natural Convection Heat Transfer Studies of Simulated and Actual Electronic
Components Using Dielectric Liquids for Immersion Cooling

by

Ronald G. Thompson Jr.
Lieutenant Commander, United States Navy
B.S., United States Naval Academy, 1980

Submitted in partial fulfillment
of the requirements for the degree of

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ABSTRACT

Two experimental studies of the natural convection characteristics of heated protrusions immersed in dielectric liquids were conducted. The first study used a three by three array of simulated 20 pin dual-in-line chips which were made from aluminum blocks with foil heaters. The second set of experiments used a three by three array of thermal evaluation devices mounted on an alumina substrate. The devices were 8.9 mm square chips which contained resistors and a type of temperature sensing transistor. Both studies used an insulated Plexiglas enclosure with a top mounted heat exchanger maintained at a constant 10 °C. Each array was mounted on a Plexiglas substrate, and spacers were used to vary the horizontal distance from the components to the enclosure wall. Five separate enclosure widths were used, with a maximum spacing of 40 mm.

The vertically oriented aluminum blocks were tested with FC-71 and power levels ranging from 0.115 W/chip to 2.9 W/chip. The non-dimensional data obtained was used to develop an empirical correlation which predicts Nusselt number as a function of Rayleigh number and enclosure width. The correlation was accurate to within 4% of the array averaged data, and the maximum uncertainty in the Nusselt number was 7.4%.

The actual electronic components were tested with FC-71, FC-43, and FC-75. Power levels ranged from 0.34 W/chip to 1.48 W/chip. Again, the data obtained was used to develop a Nusselt number correlation. In this case a better correlation of the data was achieved using Grashof number and enclosure width. The correlation is accurate to within 2% of the array averaged data. The maximum Nusselt number uncertainty was 4.7%.

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NOMENCLATURE

Symbol	Description	Units
A_{tot}	Total surface area for convection	m^2
c_p	Dielectric liquid specific heat	$J/kg \text{ } ^\circ C$
g	Acceleration due to gravity	m/s^2
Gr	Grashof number	dimensionless
h	Average heat transfer coefficient	$W/m^2 \text{ } ^\circ C$
k	Dielectric fluid thermal conductivity	$W/m \text{ } ^\circ C$
L	Chip length in vertical direction	m
Nu	Nusselt number	dimensionless
Power	Calculated power supplied to a chip	W
Pr	Prandtl number	dimensionless
Q_{loss}	Average heat loss by conduction through circuit board assembly	W
Q_{net}	Net power dissipated by a chip	W
R_c	Thermal resistance for conduction loss	$^\circ C/W$
R_p	Precision resistor resistance	Ω
Ra	Rayleigh number	dimensionless
Ra_f	Flux based Rayleigh number	dimensionless
T_{avg}	Average temperature of the five chips	$^\circ C$
T_{chip}	Indicated chip temperature from transistor voltage measurement	$^\circ C$
T_{film}	Dielectric liquid film temperature	$^\circ C$
T_{lid}	Chip lid temperature	$^\circ C$
T_s	Circuit board assembly back temperature	$^\circ C$
T_{side}	Chip side temperature	$^\circ C$
T_{sink}	Average heat exchanger (sink) temperature	$^\circ C$
V_{htr}	Voltage drop across chip resistor	V
V_{rp}	Voltage drop across precision resistor	V
X	Non-dimensional enclosure width	dimensionless

α	Dielectric liquid thermal diffusivity	m^2/s
β	Dielectric liquid thermal expansion coefficient	$1/^\circ\text{C}$
δ	Uncertainty	various
ΔT	Area-based temperature difference between chip surface and sink	$^\circ\text{C}$
ΔT_c	Temperature difference for conduction loss	$^\circ\text{C}$
ν	Dielectric liquid kinematic viscosity	m^2/s
ρ	Dielectric liquid density	kg/m^3

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I. INTRODUCTION

A. THE ELECTRONICS COOLING PROBLEM

Computer silicon chips continue to become more powerful and smaller year after year. However, the surface heat flux produced continues to increase, and the removal of this heat plays a major role in large computer design. For example, Hitachi's recent M-880 general purpose computer uses a 100 cm² water cooled module which dissipates almost 850 W, and it is predicted that heat fluxes of nearly 10⁶ W/m² will be reached by the year 2000 (Bar-Cohen, 1991).

The challenge of removing this eleven-fold increase in surface heat flux is formidable. Forced convection air cooling methods are limited to heat fluxes of 10⁴ W/m² (Bergles, 1991). Conduction cooling with a cold plate and direct immersion dielectric liquid cooling are the cooling methods currently employed in large mainframe computers. However, with the exception of the Cray series of supercomputers, major computer companies have opted for various types of generally complex cold plate assemblies.

Extensive research into direct liquid cooling is being conducted for a number of reasons. First, potential cooling schemes using dielectric fluid would be much simpler than their conduction/cold plate counterparts. Second, the advantages and disadvantages of single phase natural convection, nucleate boiling, and forced convection methods for a wide variety of available dielectric liquids needs to be studied to determine their basic heat transfer characteristics in a simulated computer circuit board environment. Finally, the results of the research should point the way for the optimum dielectric cooling method, which can then be refined in order to make it a competitive means of cooling for the next generation of computers.

B. RELATED RESEARCH

Park and Bergles (1987) conducted natural convection experiments using foil heaters mounted both flush and protruding from a circuit board. The heat transfer coefficient was measured for a single flush heater with two heater heights of 5 mm and 10 mm. Heater widths varied from 2 mm to 70 mm. Additional experiments used a vertical array of two or three heaters with various distances between them. Combinations of heaters included two or three flush in-line, two flush staggered, and two protruding in-line. Distilled water and R-113 were the fluids used.

For the single flush heater, the heat transfer coefficient was found to increase as the heater width decreases, with this effect more pronounced in the R-113. For the in-line flush heaters, the heat transfer coefficient was higher for the bottom heater, while the opposite was true for the in-line protruding heaters.

Kelleher et al. (1987) conducted a natural convection study of a long horizontal protruding heater mounted on a vertical wall in a water filled enclosure. Heat exchangers on the bottom and top of the enclosure maintained a constant temperature. Results for three separate heater positions indicated that the Nusselt number decreased as the heater position was raised. Additionally, a flow visualization study revealed that the flow was divided into two regions. The more active upper buoyancy driven region accounted for most of the heat transfer, while the more sluggish lower region was driven by shear interaction with the upper region.

Joshi et al. (1990) performed a detailed natural convection study of a vertically mounted three by three array of heated protrusions in an enclosure filled with dielectric liquid FC-75. The protrusions were horizontally oriented rectangular aluminum blocks sized to simulate 20 pin dual-in-line (DIP) packages. The top and bottom enclosure boundaries were heat exchangers set to maintain a constant temperature for various heater power levels. Enclosure width was fixed at 30 mm.

Extensive flow visualization revealed three-dimensional transport which varied with power level. As the power level was raised, the upward flow increased in intensity and complexity. Flow away from the components varied with time, and this was confirmed by time history temperature measurement. Embedded thermocouples were also used to calculate heat transfer characteristics. A correlation was developed to compute component temperature from the dissipated power.

A follow-on investigation by Joshi et al. (1991) utilized a vertically mounted three by three array of vertically oriented heated protrusions in an enclosure filled with three different fluorinert type dielectric liquids. Again, the upper and lower boundaries were constant temperature heat exchangers. Enclosure widths of 13 mm and 30 mm were used with varying power levels in this study.

It was found that the top and bottom enclosure conditions affected the component temperatures to a greater degree for the lower power levels. The effect of enclosure width was minimal on the resultant calculations of Nusselt and modified Rayleigh numbers. These non-dimensional heat transfer characteristics were correlated in a similar manner to the previous investigation.

A similar set of experiments using ethylene glycol as the fluid was conducted by Keyhani et al. (1991). Five heated protrusions were uniformly spaced on a vertical wall inside an enclosure equipped with a top mounted heat exchanger. Six different enclosure widths varying from 13.5 mm to 45 mm were tested at power levels ranging from 2 W to 12 W per heater.

Flow visualization of the experiments revealed primary flows along the vertical walls separated by a narrow core flow consisting of secondary flow cells. The heat transfer coefficient of the top and bottom heaters was influenced markedly by the power level and enclosure width. A single correlation for Nusselt number versus modified Rayleigh number was developed that was independent of heater location and enclosure width.

Thesis experiments accomplished by Aytar (1991) and Matthews (1991) further established the heat transfer abilities of dielectric liquids on simulated electronic components. They both used a three by three array of 20 pin DIP sized aluminum blocks mounted vertically in an enclosure with a top mounted heat exchanger. Aytar studied the effects of enclosure width, power level, and Prandtl number on horizontally oriented protrusions. Matthews performed similar experiments with the protrusions oriented vertically.

C. OBJECTIVES

The objectives of this thesis were as follows:

1. Complete collecting data on Matthews' vertically oriented experimental setup. The component power level and enclosure width were varied using FC-71 as the dielectric liquid.
2. Reduce the above data into useful dimensional and non-dimensional parameters.
3. Utilize the non-dimensional data in developing an empirical correlation for the Nusselt number which takes into account the effects of enclosure width and Rayleigh number.
4. Using a circuit board assembly provided by NSWC, Crane, fabricate an actual three by three electronic component array experiment.
5. Collect natural convection heat transfer data on this circuit board assembly using dielectric liquids FC-71, FC-43, and FC-75. The power level and enclosure width were varied for each liquid similar to the previous study.
6. Reduce the above data into useful dimensional and non-dimensional parameters.
7. Utilize the non-dimensional data in developing a single correlation for the Nusselt number which takes into account the effects of enclosure width and Grashof number.
8. Based on the above findings, make a recommendation for the best dielectric liquid to use for natural convection cooling. Additionally, recommend additional areas for future research using the selected liquid.

II. EXPERIMENTAL APPARATUS

A. TEST CHAMBER ASSEMBLY

The test chamber assembly consisted of the rectangular enclosure and heat exchanger used by Matthews. The enclosure was constructed of 25.4 mm and 12.7 mm thick Plexiglas. A 3.2 mm O-ring sealed the boundary between the enclosure walls and the bottom of the heat exchanger. The two were assembled together with 12 threaded studs, washers, and nuts. Plexiglas inserts of various widths were used in order to vary the spacing between the circuit board assembly and the wall. Details of the enclosure are shown in Figure 1.

The top mounted heat exchanger was a Plexiglas and aluminum single-pass type with five rectangular channels for cooling water flow. Heat transferred from the dielectric liquid was conducted to the coolant across the 3 mm thick aluminum plate which formed the bottom of the heat exchanger. Three thermocouples embedded in the aluminum plate were used for temperature measurement. A drawing of the heat exchanger is shown in Figure 2.

B. SIMULATED CIRCUIT BOARD

The first set of experiments utilized the same component board used by Matthews. Nine aluminum blocks arranged in a three-by-three array were mounted on a 12.7 mm thick Plexiglas substrate. The numbering system for the components was unchanged: bottom to top, right column to left column. Small $10.6\ \Omega$ foil heaters were located between the blocks and the substrate. Temperatures were measured using six thermocouples per block. Nine thermocouples were also mounted on the back of the substrate for conduction loss calculations. Details of the heater, thermocouple, and block mounting procedures are described by Aytar (1991). A drawing of the circuit board is shown in Figure 3.

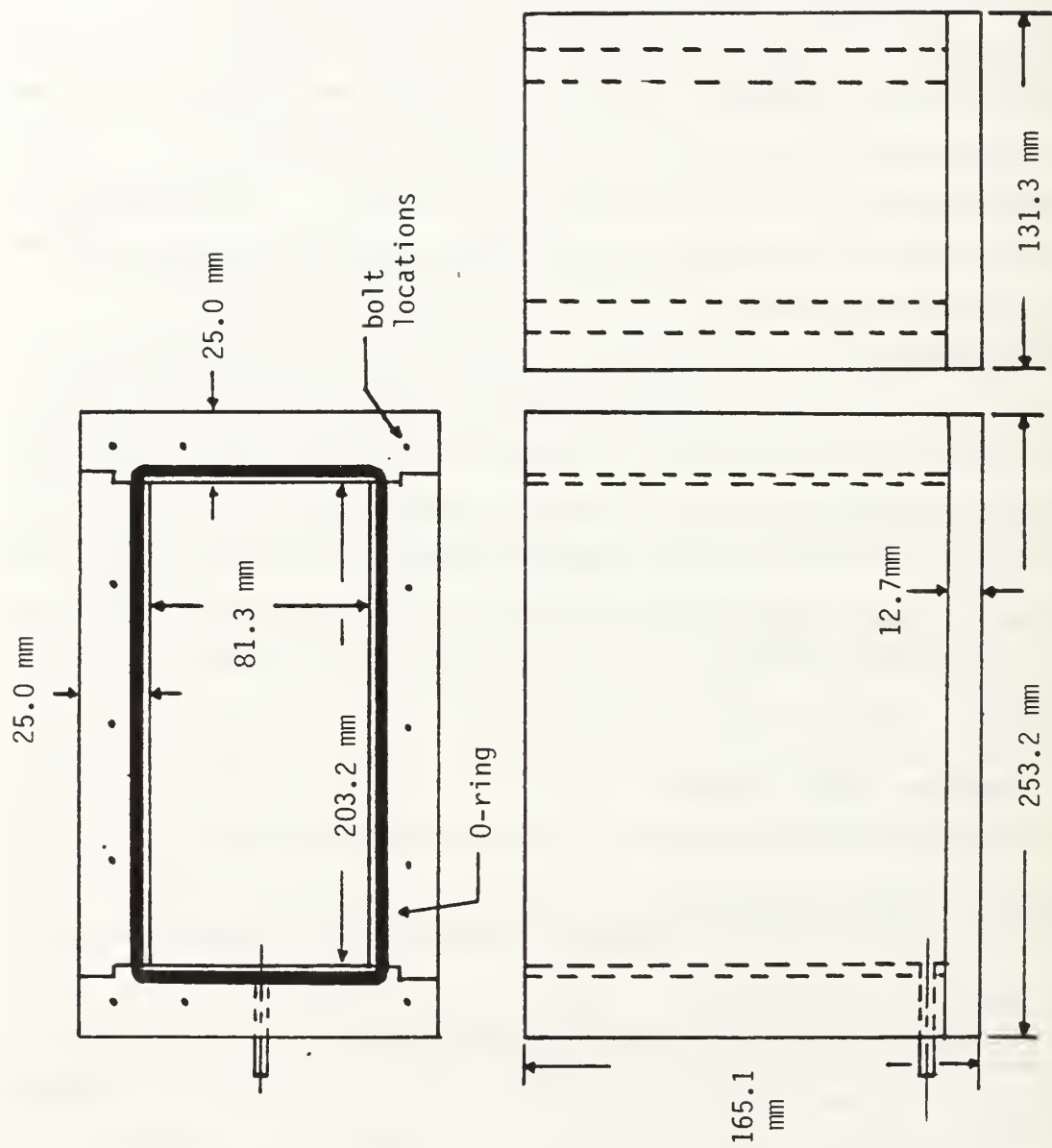


Figure 1. Enclosure

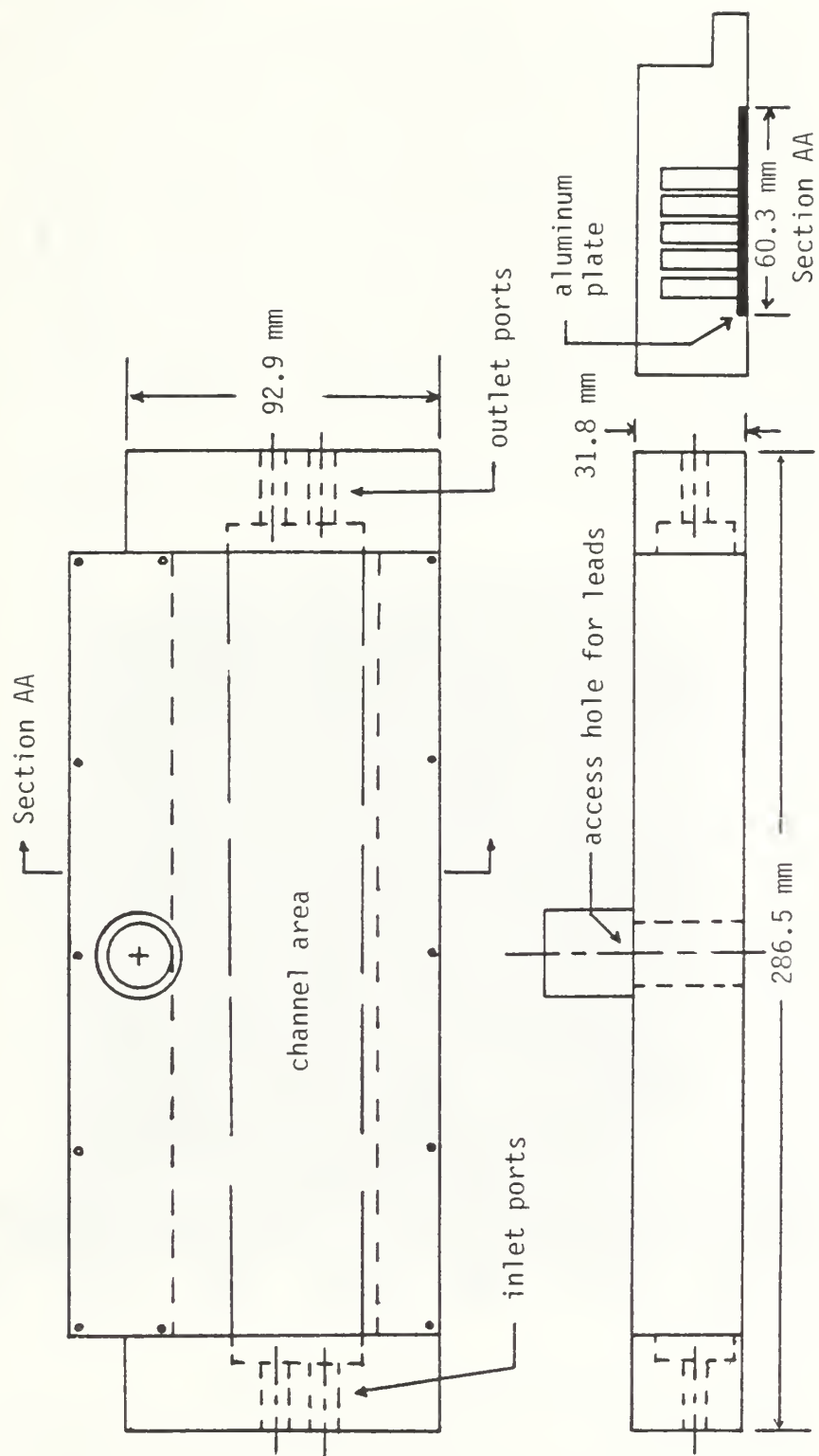


Figure 2. Heat Exchanger

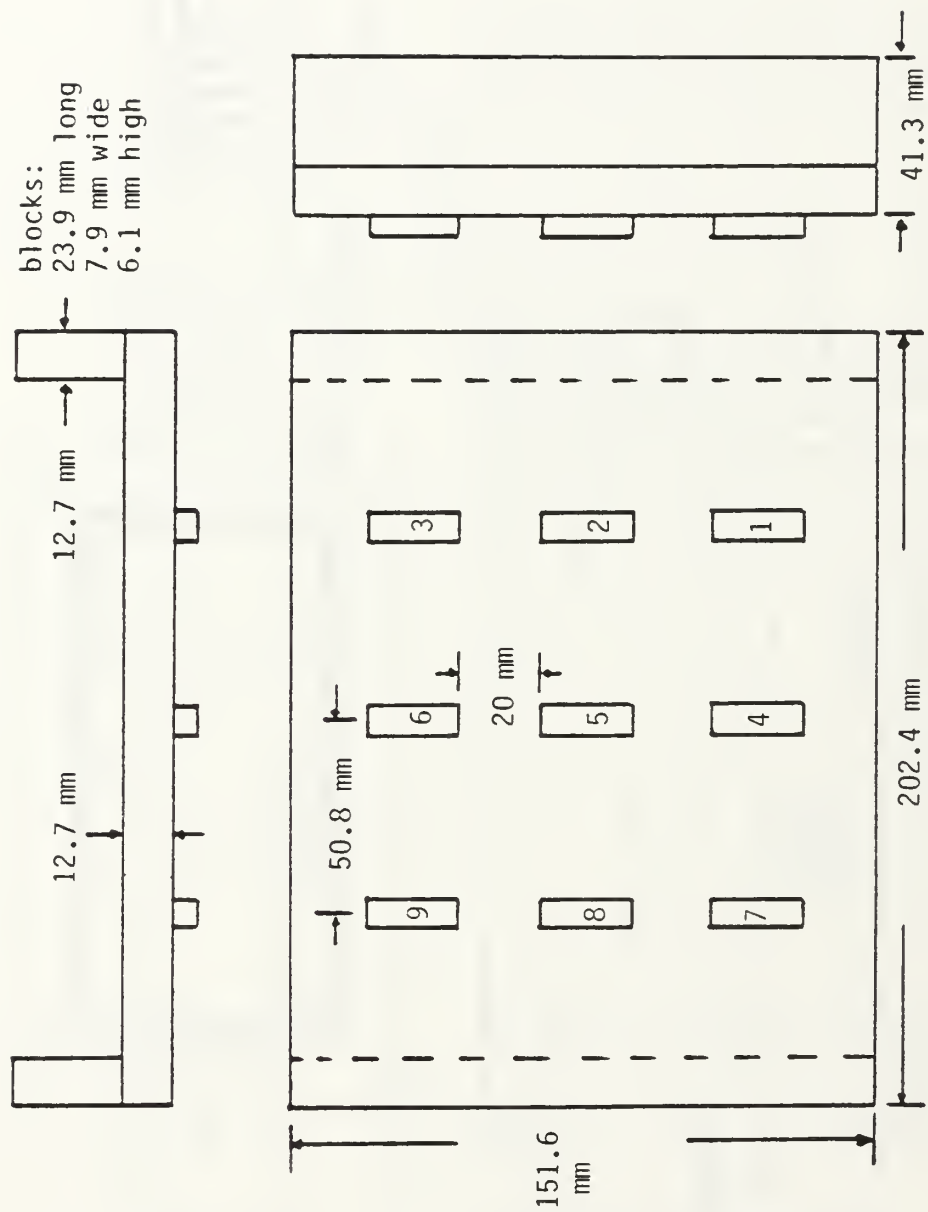


Figure 3. Circuit Board

C. NSWSC CIRCUIT BOARD

The remainder of the experiments used a three by three array of Texas Instruments thermal evaluation devices which were assembled on a 50.8 mm square alumina substrate board by Naval Surface Warfare Center (NSWC), Crane, Indiana. Each device is an 8.9 mm square chip which contains four resistors and a Temperature Sensing Element (TSE). Again, the same component numbering system is used for consistency. The resistors, when connected in series, have a resistance of approximately 165 ohms. The TSE is a solid state temperature measuring device, or a type of transistor. Figure 4 is a photograph of the circuit board, and a schematic diagram of the chip internals is shown in Figure 5.

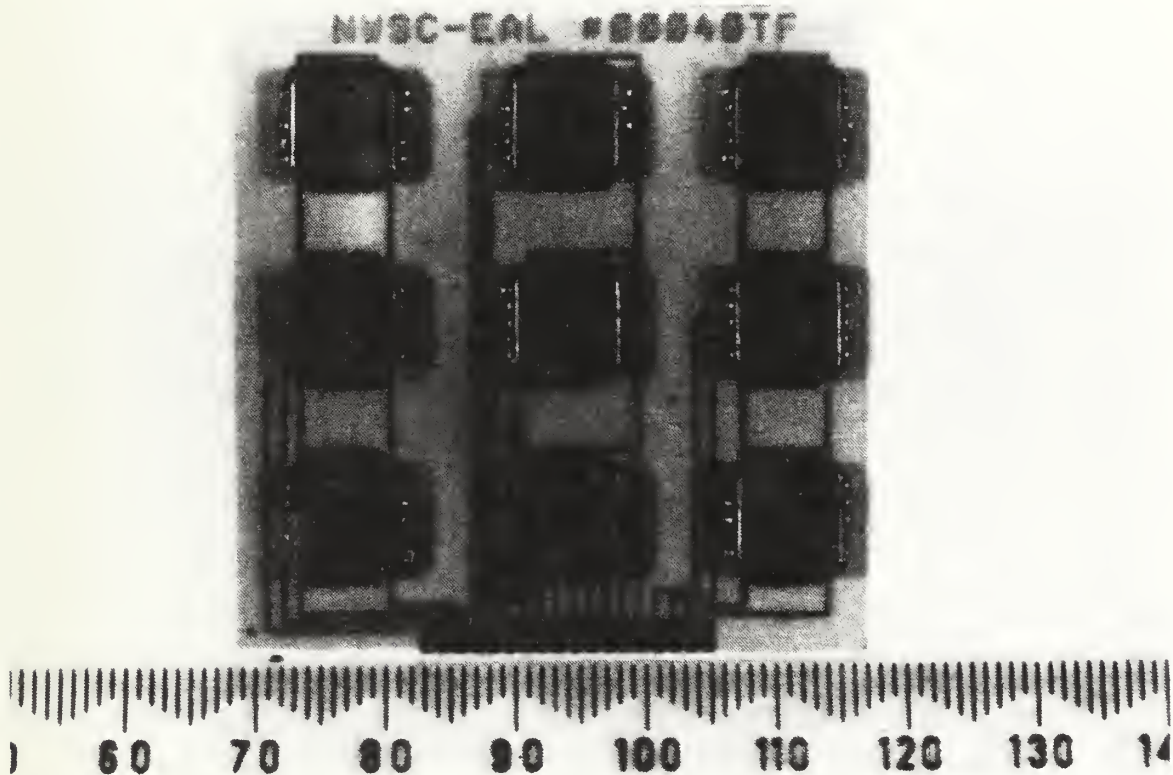


Figure 4. NSWSC Circuit Board

To simulate actual electronic chips, power is provided to the resistors for heating purposes. Using a constant 1 mA current source, the

voltage across the transistor base to emitter, V_{BE} , is measured. Component temperature is then obtained from a previously plotted calibration curve of temperature versus V_{BE} . The complete circuit board is mounted in the center of a Plexiglas substrate board with essentially the same dimensions as in Figure 3. 18 thermocouples were added as follows:

- Five on the chip lid surfaces. Located on chip #2, 4, 5, 6 and 9.
- Four on the substrate face. Located diagonally between chip #1 and 5, #3 and 5, #7 and 5, and #9 and 5.
- Nine on the circuit board assembly back. Located directly behind the chips.

The complete circuit board is shown in Figure 6.

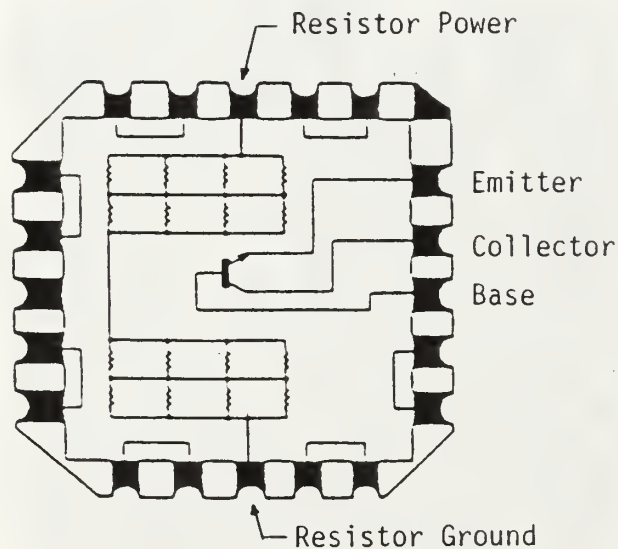


Figure 5. Thermal Evaluation Device Circuitry

D. SYSTEM HARDWARE

1. Simulated Circuit Board

Copper-constantan thermocouples, 0.010 inch diameter, were used for temperature measurement on the simulated circuit board assembly. Each

foil heater was connected in series to a $2.0 \Omega \pm 2.5\%$ resistor. These resistors, in turn, were connected in parallel to a 0-100 V, 0-5 A direct current power supply. This arrangement allowed for a simple calculation of heater power from measured voltages. Details of this simple calculation can be found in Matthews' thesis. The thermocouples and heaters were connected to a Hewlett-Packard HP-3497A Data Acquisition System (DAS). The input and output to the DAS was via an HP-9826 microcomputer. The data channels were unchanged from Matthews' experiments, and they are repeated below:

- Channels 0-53 Aluminum block temperatures
- Channels 54-56 Heat exchanger temperatures
- Channels 57-60 Back of board assembly temperatures
- Channel 61 DC power supply voltage
- Channels 62-70 Foil heater voltages
- Channels 71-75 Back of board assembly temperatures
- Channel 76 Ambient temperature

2. NSWC Circuit Board

As before, identically sized copper-constantan thermocouples were used with two HP-3497A units. One HP-3497A with two, twenty channel cards was used to measure thermocouple and voltage data. The other HP-3497A was used only as a current source. It had one, twenty channel card modified to supply the constant 1 mA current to the TSEs. Power supplied to each chip was easily calculated by multiplying the chip resistor voltage by its current. The current was equal to the precision resistor voltage divided by its resistance.

The chip resistors were powered from the same direct current power supply. However, all nine chips could not be powered individually. This was due to the circuit board layout employed by NSWC. The center column, chip #4, 5, and 6, could be powered individually, but the chip resistors in the two outer columns were each wired in parallel.

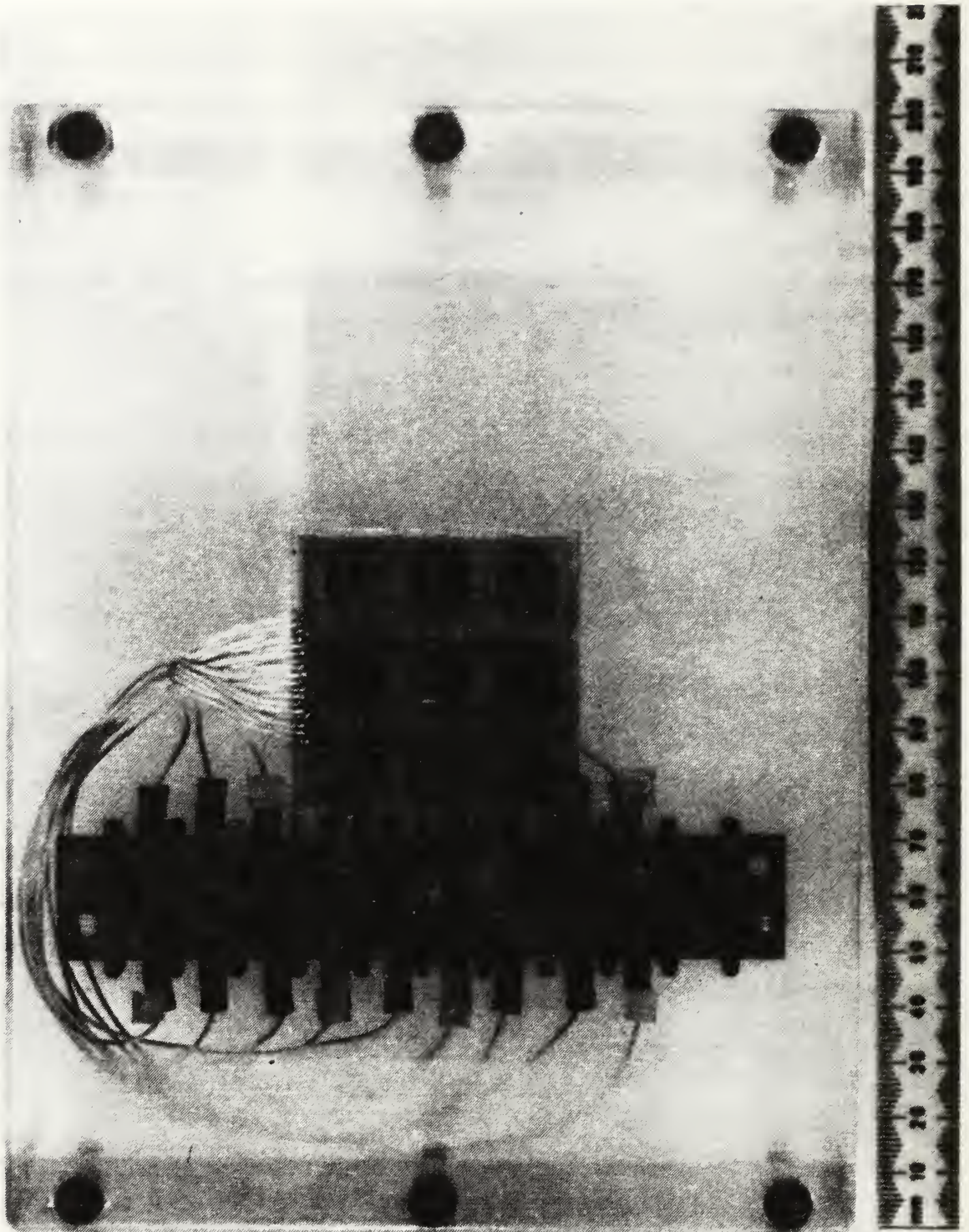


Figure 6. NSWC Circuit Board Assembly

Additionally, due to circuit board constraints, TSE junction temperatures could not be measured for the four corner chips. The channels were numbered as follows:

- Channels 0-4 Chip resistor voltages
- Channels 5-9 Precision resistor voltages
- Channel 10 DC power supply voltage
- Channels 11-13 Heat exchanger temperatures
- Channels 41-44 Substrate surface temperatures
- Channels 45-49 Chip lid temperatures
- Channel 50 Ambient temperature
- Channels 51-59 Circuit board assembly back temperatures
- Channels 60-64 Chip V_{BE} voltages

III. EXPERIMENTAL PROCEDURE

A. HARDWARE PREPARATION

Similar preparations were made for both sets of experiments. Once the component assembly was in place, the proper sized spacer was inserted for the particular run. Careful measurements of assembly to enclosure wall spacing were performed to ensure accuracy. Frequently a small amount of silicone RTV was applied to the corners of both the component assembly and the spacer to correct for any small warpage. The remaining steps were as follows:

1. The enclosure was filled nearly to the top with the proper dielectric liquid.
2. The heat exchanger was bolted to the enclosure, with the O-ring providing a seal.
3. The circulating bath supply and return lines were attached to the heat exchanger. The bath unit was set to 6-9 °C (indicated) and energized. It was found that this temperature setting was required in order for the heat exchanger temperature to be 10 °C during the runs. The system was checked for leaks.
4. The foil heaters or chip resistors were energized with the DC power supply.
5. The HP-3497A was energized. All channels were scanned to insure continuity.
6. Foam insulation was attached to the enclosure walls and bottom.
7. Additional dielectric liquid was siphoned into the enclosure via the vent hole. The air bubbles trapped directly under the heat exchanger were manipulated to the lead access hole via a small slot in the tops of the component assemblies.

B. EXPERIMENTAL PROCEDURE

After the above preparations were completed, the proper voltage was set on the DC power supply. Data was taken after steady state conditions were reached. For the FC-71, this took at least eight hours from ambient conditions. Subsequent runs took two to six hours to reach steady state, depending on power level and spacing. Steady state was achieved when two

to three data runs taken approximately 10 minutes apart indicated a ± 0.1 °C random temperature difference between the thermocouple readings. This criterion was changed to conform with Matthews' work for other dielectric liquids. It was ± 1 °C and ± 0.4 °C, respectively, for FC-75 and FC-43. As a rule, steady state conditions were achieved sooner with FC-75 and FC-43.

For the simulated circuit board, data acquisition was accomplished with the software program ACQUIRE. Calculations were then performed with the program CALCDIEL. Both of these programs were originally written and modified by Pamuk, Benedict, Torres, Powell, Aytar, and Matthews. They are included in Appendices A and B.

For the NSWC circuit board, two new programs were written. The program ACQ2 was used for data acquisition, and calculations were performed by the program CALC2. Major improvements in the new software are as follows:

- Since the TSEs had to be calibrated, the thermocouples were calibrated at the same time. Calibration was performed using a constant temperature oven and a platinum resistance thermometer. The equations for the resulting calibration curves were used in the programs to convert voltages to temperature. Temperature uncertainty was calculated to be 0.275 °C for the TSEs, and it was 0.3 °C for the thermocouples.
- To promote understanding and facilitate future modifications, extensive documentation and explanations are included in the programs, where appropriate.

The programs ACQ2 and CALC2 are included in Appendices C and D.

IV. DATA ANALYSIS

A. SIMULATED CIRCUIT BOARD

The experiments using the FC-71 were virtually identical in nature to the experiments previously completed with FC-75 and FC-43. A detailed explanation of the methodology used for data analysis can be found in Matthews' thesis.

B. NSWC CIRCUIT BOARD

The program CALC2 was used to obtain the Nusselt, Rayleigh, and Grashof numbers for the NSWC circuit board. These non-dimensional parameters were calculated on a single chip, horizontal row, and array basis for three dielectric liquids (FC-75, FC-43, and FC-71) and five enclosure spacings. Various power levels ranging from approximately 0.34 W/chip to 1.48 W/chip were tested. Assumptions used in the CALC2 program were as follows:

- The chip was modeled as a square wafer.
- Chip and lid temperatures of the four corner chips were assumed identical to the horizontally adjacent chip in their respective row.
- Chip side temperature was the average of the TSE and the lid temperatures.
- Conduction was assumed one dimensional from the chip to the back of the Plexiglas substrate. The heat was conducted from the chip, through the alumina circuit board and a very thin layer of silicone rubber, to the Plexiglas.
- Thermophysical properties of the above materials were assumed constant at a reference temperature.
- Thermophysical properties of the dielectric liquids were assumed constant. They were evaluated at T_{film} .
- The temperature difference used for the calculation of the heat transfer coefficient was area weighted. The lid accounted for about 55% of the convection area, so the surface temperature was 55% of T_{lid} plus 45% of T_{side} .
- All contact resistances were assumed negligible.

Calibration curve equations for the thermocouples and the TSEs were entered into the program. All thermocouple temperatures were obtained from the following equation:

$$T\text{ (}^{\circ}\text{C)} = 0.24977483 + 24.896088V - 0.079219169V^2$$

where V is the thermocouple voltage in millivolts. For accuracy, each TSE had its own calibration curve. A representative equation for chip #2 is as follows:

$$T\text{ (}^{\circ}\text{C)} = 577.58074 - 575.54353V_{\text{EF}}$$

Similar equations were obtained for the other TSEs. The calibration curve for the thermocouples is shown in Figure 7, and a TSE calibration curve is shown in Figure 8. The calibration data for the five TSEs is included in

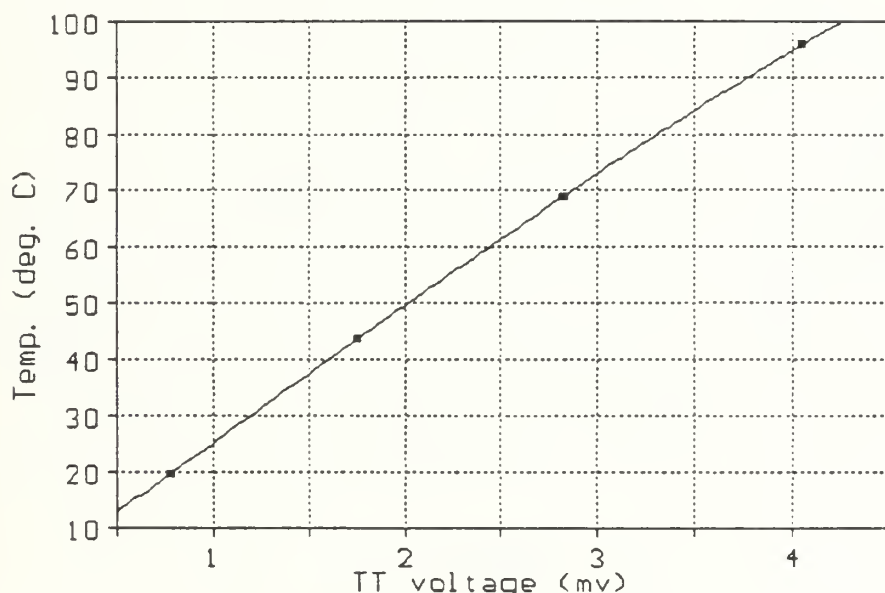


Figure 7. Thermocouple Calibration Curve

Appendix F.

Due to the wiring scheme of the NSWC circuit board, the power supplied to the TSEs had to be calculated two different ways. The power for the

individually wired TSEs in the center column (chip #4, 5, and 6) was defined as follows:

$$Power = \frac{V_{htr} V_{rp}}{R_p}$$

where

V_{htr} = voltage drop across chip resistor

V_{rp} = voltage drop across precision resistor

R_p = precision resistor resistance

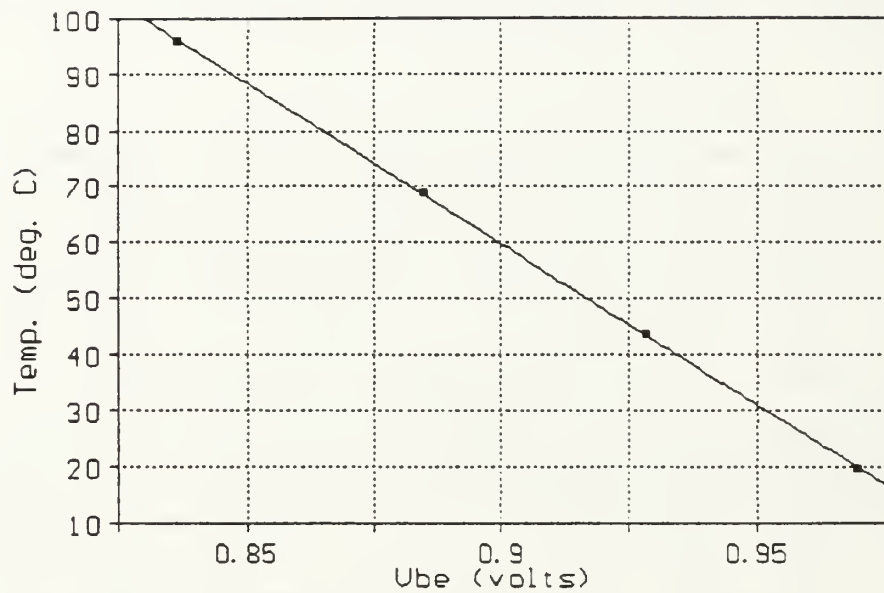


Figure 8. TSE #2 Calibration Curve

The remainder of the TSEs were wired in two parallel sets (chip #1, 2, and 3 and chip #7, 8, and 9), but only chip #2 and 8 could be read directly. Therefore, the power was modified as follows:

$$Power = \frac{V_{htr} V_{rp}}{3R_p}$$

with the correct V_{htr} (chip #2 or 8) substituted in the equation.

The heat loss by conduction involved several materials. The thermal resistance for conduction loss, R_c , was represented as:

$$R_c = \frac{1}{A} \sum \frac{L_i}{k_i}$$

where

A = cross-sectional area for conduction

L_i = material thickness

k_i = material thermal conductivity

The equation for Q_{loss} , calculated for each chip, was therefore:

$$Q_{loss} = \frac{\Delta T_c}{R_c}$$

where ΔT_c was the difference in temperature between the TSE derived temperature, T_{chip} , and the circuit board assembly back temperature, T_s .

The net heat transferred from the chip to the dielectric liquid could then be calculated from the following equation:

$$Q_{net} = Power - Q_{loss}$$

The average heat transfer coefficient, h , was calculated from:

$$h = \frac{Q_{net}}{A_{tot} \Delta T}$$

where

A_{tot} = total surface area for convection

ΔT = area based temperature difference between the chip surface and the heat exchanger, or sink

In equation form,

$$\Delta T = (0.55T_{lid} + 0.45T_{side}) - T_{sink}$$

T_{sink} is the average of the three heat exchanger temperatures.

The program then calculates the thermophysical properties for the particular dielectric liquid under investigation. The properties were evaluated at the film temperature, T_{film} , which was:

$$T_{film} = \frac{T_{avg} + T_{sink}}{2}$$

where T_{avg} was the average of the five chip TSE temperatures. The corresponding equations for the properties are outlined below:

Thermal conductivity, k (W/m °C)

$$FC-75: k = \frac{(0.65 - 7.89474 \times 10^{-4} \times T_{film})}{10}$$

$$FC-43: k = 0.0666 - 9.864 \times 10^{-6} \times T_{film}$$

$$FC-71: k = 0.071$$

Density, ρ (kg/m³)

$$FC-75: \rho = (1.825 - 0.00246 \times T_{film}) \times 1000$$

$$FC-43: \rho = (1.913 - 0.00218 \times T_{film}) \times 1000$$

$$FC-71: \rho = (2.002 - 0.00224 \times T_{film}) \times 1000$$

Specific heat, c_p (J/kg °C)

$$FC-75, 43, 71: c_p = (0.241111 + 3.7037 \times 10^{-4} \times T_{film}) \times 4187$$

Kinematic viscosity, ν (m²/s)

$$FC-75: \nu = (1.4074 - 2.964 \times 10^{-2} \times T_{film} + 3.8018 \times 10^{-4} \times T_{film}^2 - 2.7308 \times 10^{-6} \times T_{film}^3 + 8.1679 \times 10^{-9} \times T_{film}^4) \times 10^{-6}$$

$$FC-43: \nu = (8.875 - 0.47007 \times T_{film} + 1.387 \times 10^{-2} \times T_{film}^2 - 2.1469 \times 10^{-4} \times T_{film}^3 + 1.3139 \times 10^{-6} \times T_{film}^4) \times 10^{-6}$$

$$FC-71: \nu = 10^{-6} \times \exp(6.8976 - 0.1388 \times T_{film} + 1.331 \times 10^{-3} \times T_{film}^2 - 7.041 \times 10^{-6} \times T_{film}^3 + 1.523 \times 10^{-8} \times T_{film}^4)$$

Coefficient of thermal expansion, β ($1/^\circ\text{C}$)

$$FC-75: \beta = \frac{0.00246}{1.825 - 0.00246 \times T_{film}}$$

$$FC-43: \beta = \frac{0.00218}{1.913 - 0.00218 \times T_{film}}$$

$$FC-71: \beta = \frac{0.00224}{2.002 - 0.00224 \times T_{film}}$$

Now the various dimensionless numbers which characterize the heat transfer can be calculated. First, the ratio of thermal energy conduction to storage, or thermal diffusivity, was found from:

$$\alpha = \frac{k}{\rho C_p}$$

Then the Prandtl number could be calculated from:

$$Pr = \frac{\nu}{\alpha}$$

A primary measure of convective heat transfer, the Nusselt number, was defined as:

$$Nu = \frac{hL}{k}$$

where L was the vertical length of an individual chip. Natural convection effectiveness is measured by the Grashof number, which can be calculated from:

$$Gr = \frac{g\beta L^3 (T_{avg} - T_{sink})}{\nu^2}$$

Finally, the Rayleigh number is defined as:

$$Ra = GrPr$$

V. SIMULATED CIRCUIT BOARD RESULTS

A. GENERAL

The natural convection heat transfer characteristics of a vertically oriented array of simulated electronic components were studied using the dielectric liquid FC-71 as the coolant. The experiments were performed on the same equipment used by Matthews. The enclosure widths used, after careful measurement, were determined to be 7, 9, 16, 28, and 40 mm. The 2 mm difference between these widths and those reported by Matthews is due to the actual enclosure width, with no spacers, being 40 mm wide instead of 42 mm. The same Plexiglas spacers were used for the FC-71 runs. Additionally, the same approximate power levels of 0.115, 0.34, 0.8, 1.3, 1.7, 2.25, and 2.9 W/component were used for the FC-71 study.

The non-dimensional data obtained with the FC-71 was combined with Matthews' results with the FC-75 and FC-43. An empirical correlation for the Nusselt number, Nu , was then derived in a similar manner for the third liquid, FC-71. As defined in Matthews' thesis, this correlation accounted for variations in Rayleigh number and chamber width. Conspicuously absent is the variation due to Prandtl number, Pr . After reviewing Matthews' results, it was determined that the effect of Pr was accounted for in the Rayleigh number. The correlation is of the form:

$$Nu = a Ra^{b_1} X^{b_2}$$

where Nu is based on the component dimension in the direction of gravity, X is a non-dimensional enclosure width, and a , b_1 , and b_2 are constants.

B. DIMENSIONAL RESULTS

The array average temperature, $T_{avg} - T_{sink}$, is plotted against net power, Q_{net} , in Figure 9. Figure 10 through Figure 14 show the same plot for all three liquids, with each figure representing a different spacing.

The general shape of the curves in Figure 9 are identical to similar graphs of FC-43 and FC-75 data taken by Matthews (vertical orientation) and Aytar (horizontal orientation). However, the aluminum block temperatures are much hotter when using FC-71. Matthews took the maximum increase in the array average temperatures, which occurred at 2.86 W, and calculated the average of the five spacings. He reported this temperature to be 34.9 °C for FC-75 and 48.3 °C for FC-43. For FC-71, this temperature is 67.2 °C.

The maximum component temperature for FC-71 cooling was 76.5 °C, and it occurred at a power level of 2.9 W and an enclosure width of 7 mm. Corresponding values for FC-75 and FC-43 were 52 °C and 68 °C, respectively.

To be consistent with previous work, all FC-71 component temperature data was averaged. The temperatures of the three blocks on any row were averaged to facilitate a row-by-row comparison. For all power levels and spacings, the order of average row temperatures were always top > middle > bottom. This pattern indicated that the buoyancy forces overcame the viscous forces in the fluid. The boundary layers emerging from each component were definitely affected by the natural convection flow below them. This pattern for FC-71 compares to middle > top > bottom for FC-75 and top > middle > bottom for FC-43.

Specific component and row temperature extremes for the FC-71 data were as follows:

- Maximum block temperature occurred on chip #3 60% of the time and chip #6 or 9 34% of the time.
- Minimum block temperature occurred on chip #7 97% of the time.
- Maximum temperature difference between the top and middle rows was 1.7 °C. For the middle and bottom rows, it was 6.0 °C. These differences were noted at a power level of 2.9 W and a spacing of 7 mm.

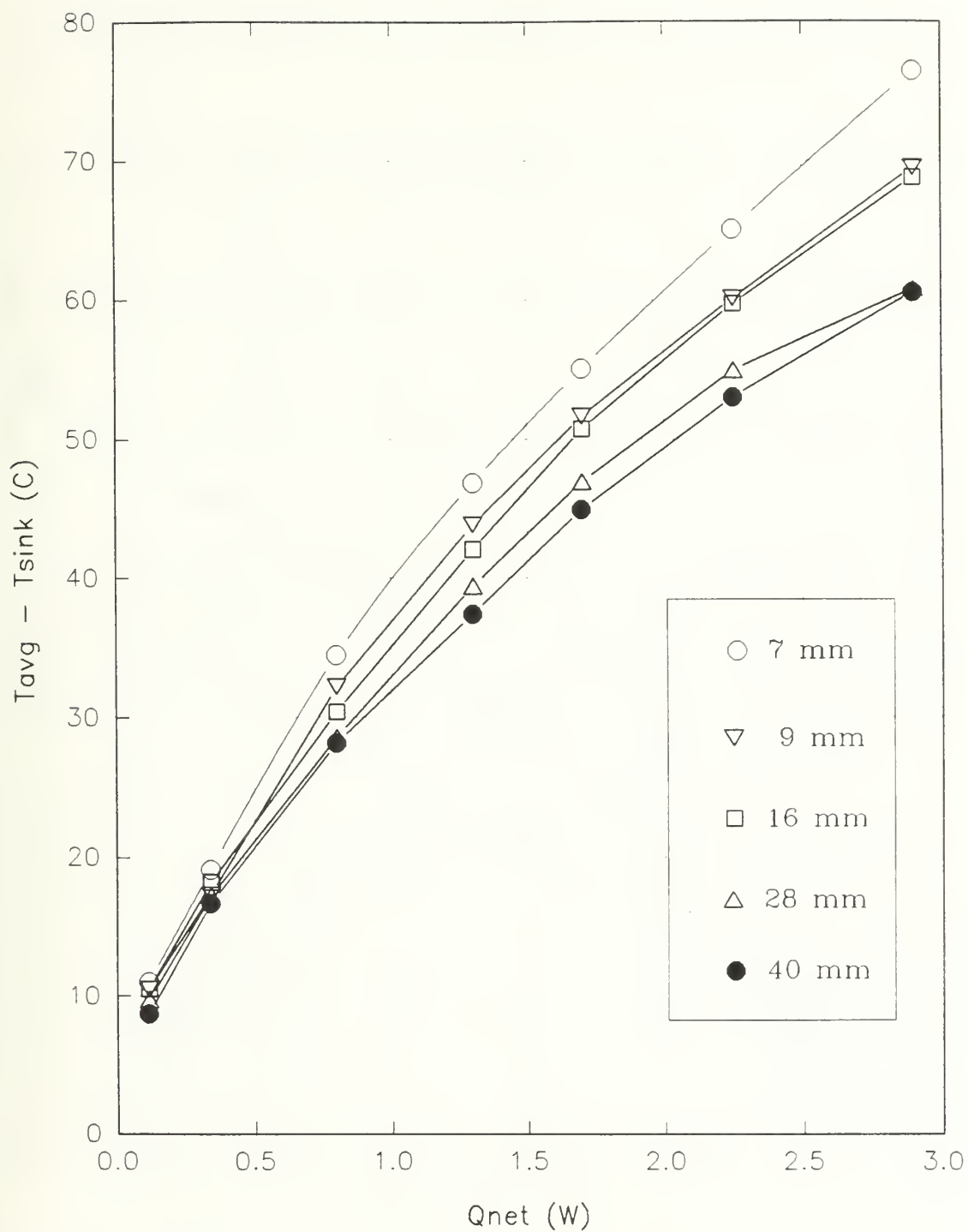


Figure 9. Array Average Temperature vs. Net Power for FC-71, Vertical Orientation

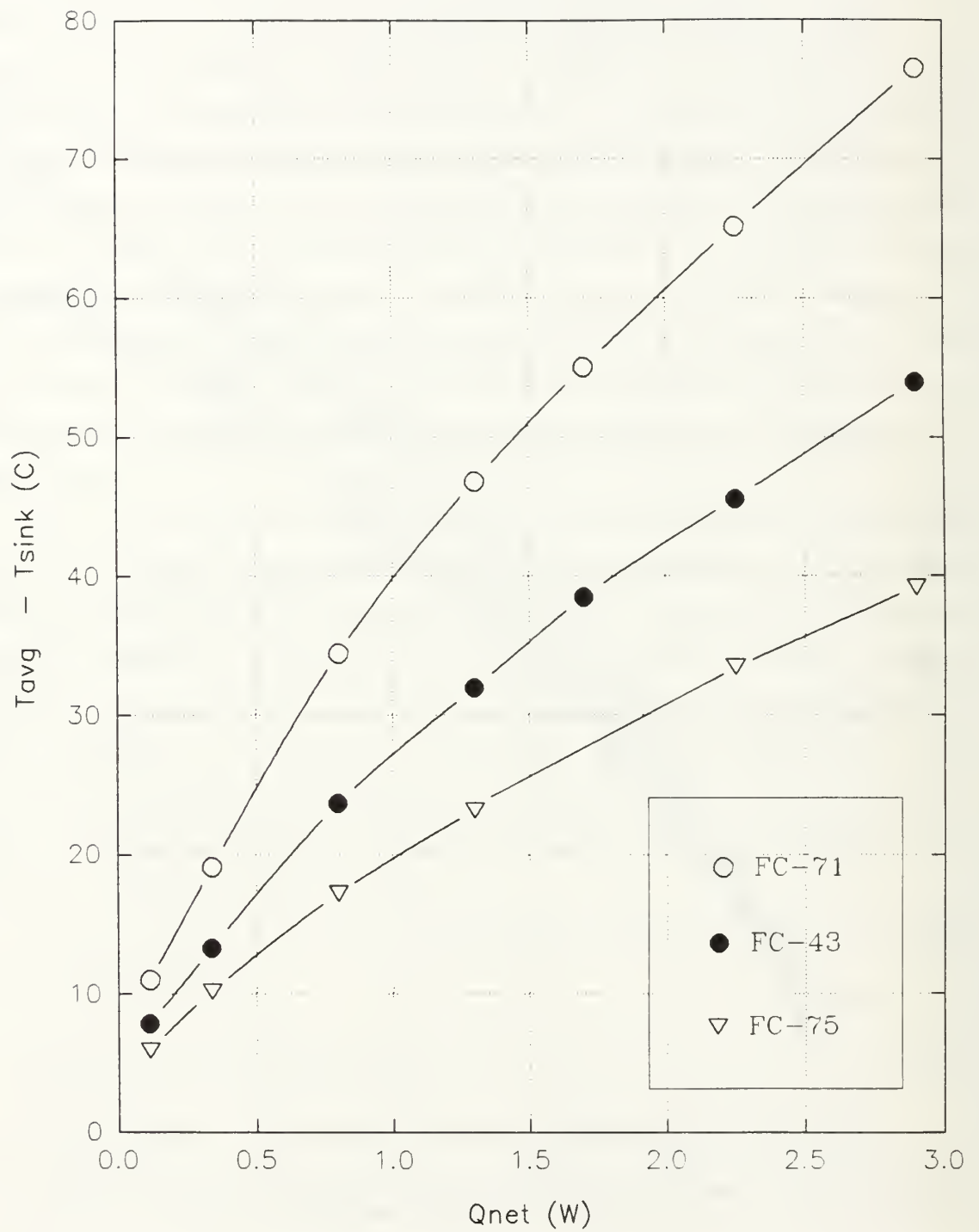


Figure 10. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 7 mm Spacing, Vertical Orientation

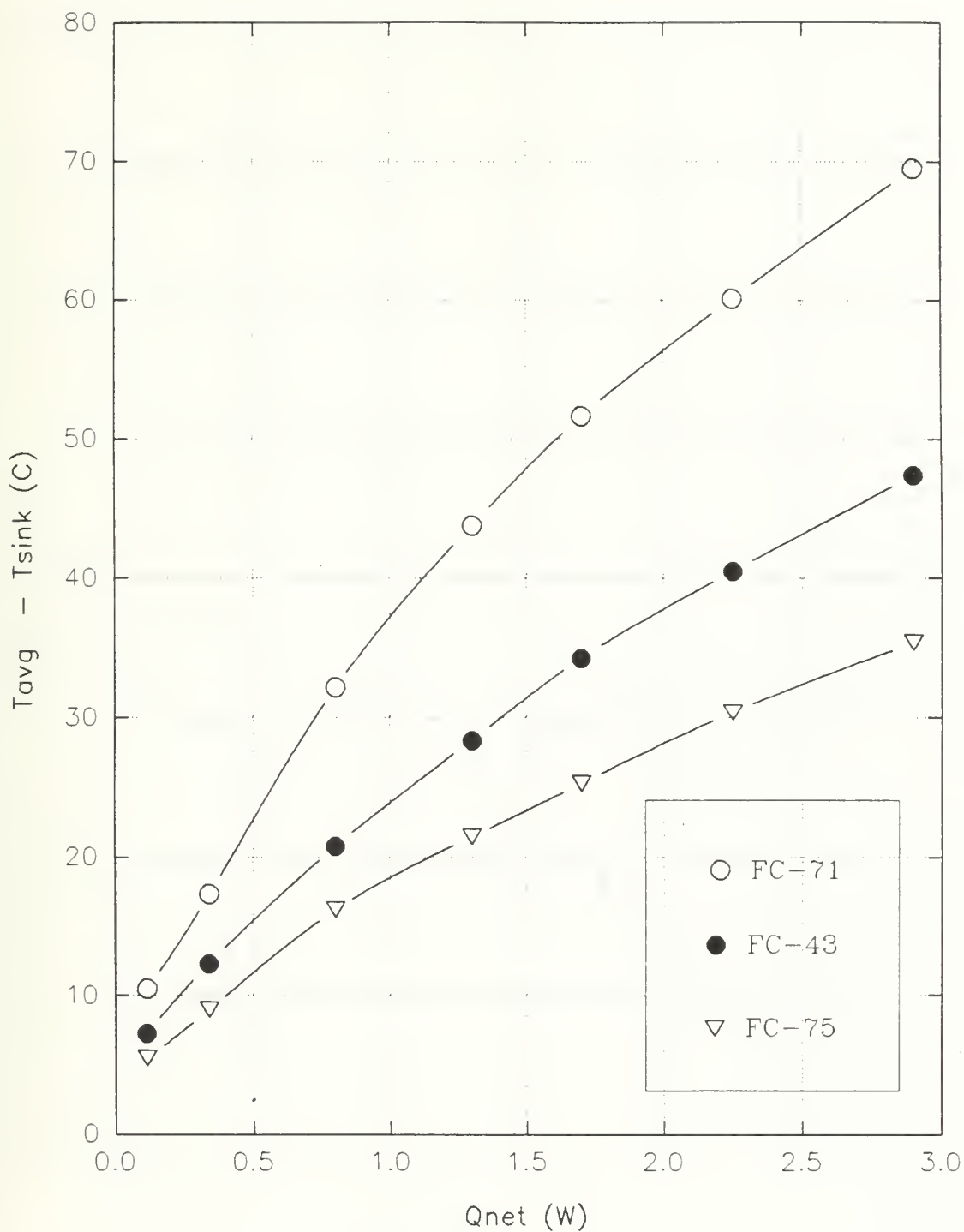


Figure 11. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 9 mm Spacing, Vertical Orientation

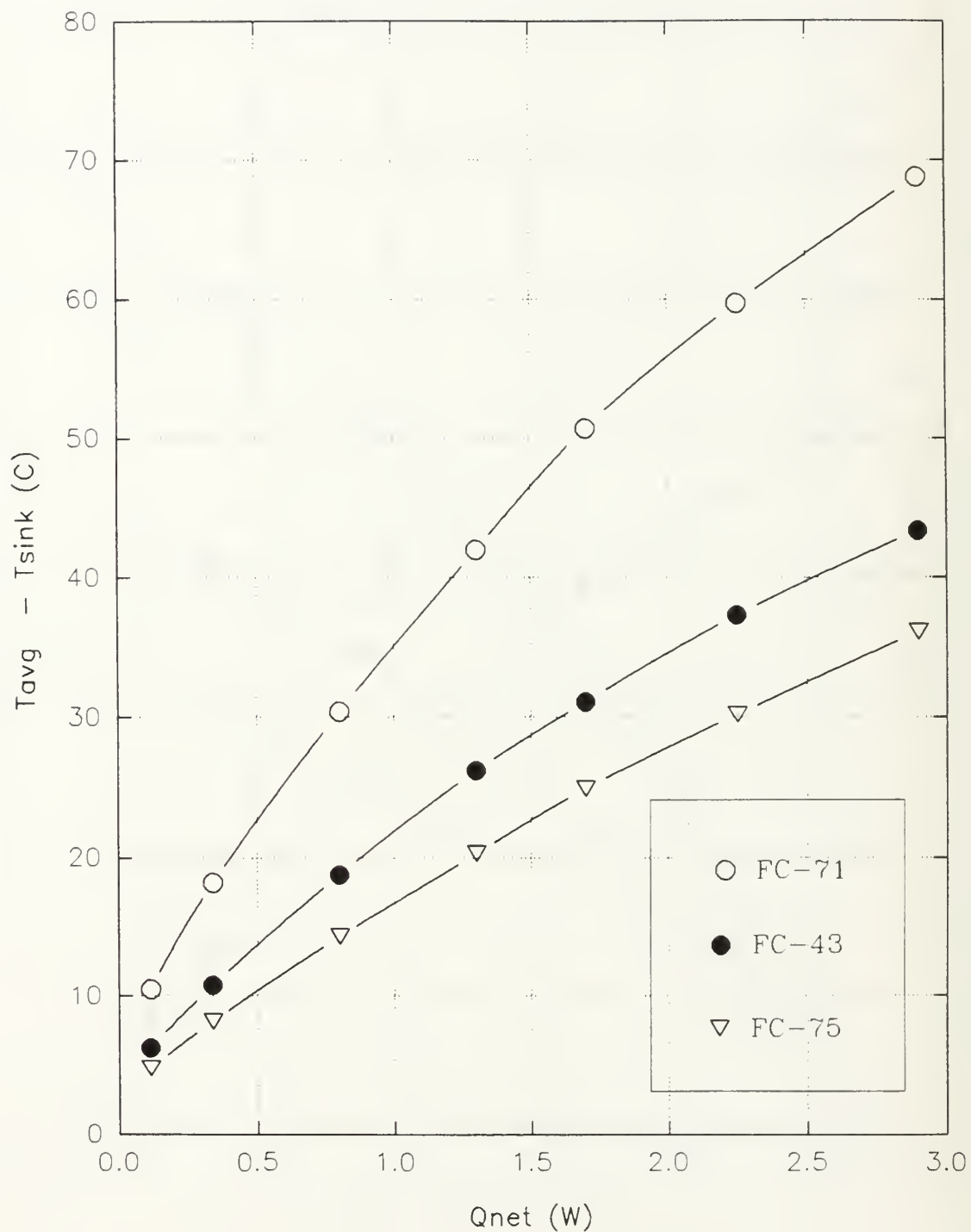


Figure 12. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 16 mm Spacing, Vertical Orientation

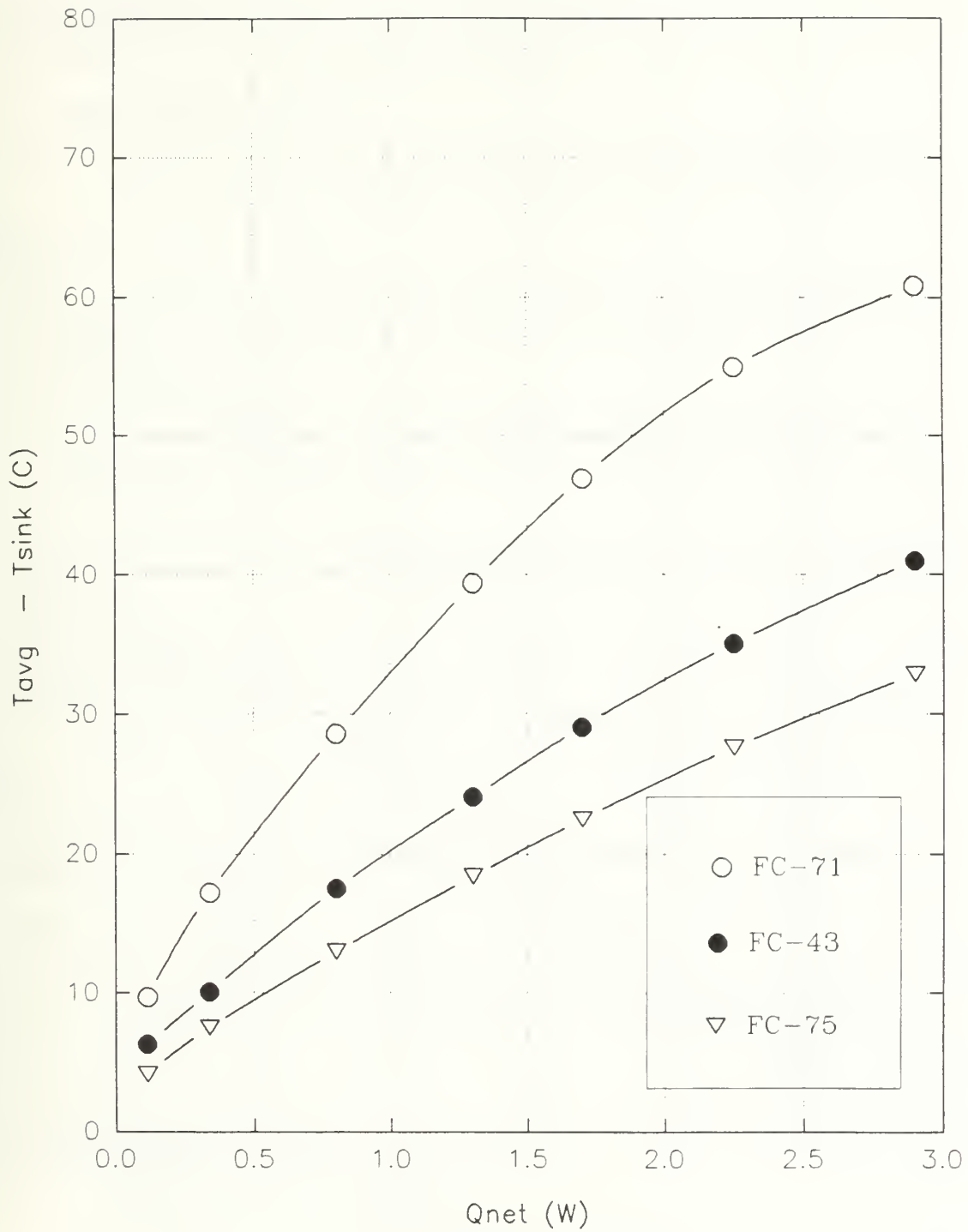


Figure 13. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 28 mm Spacing, Vertical Orientation

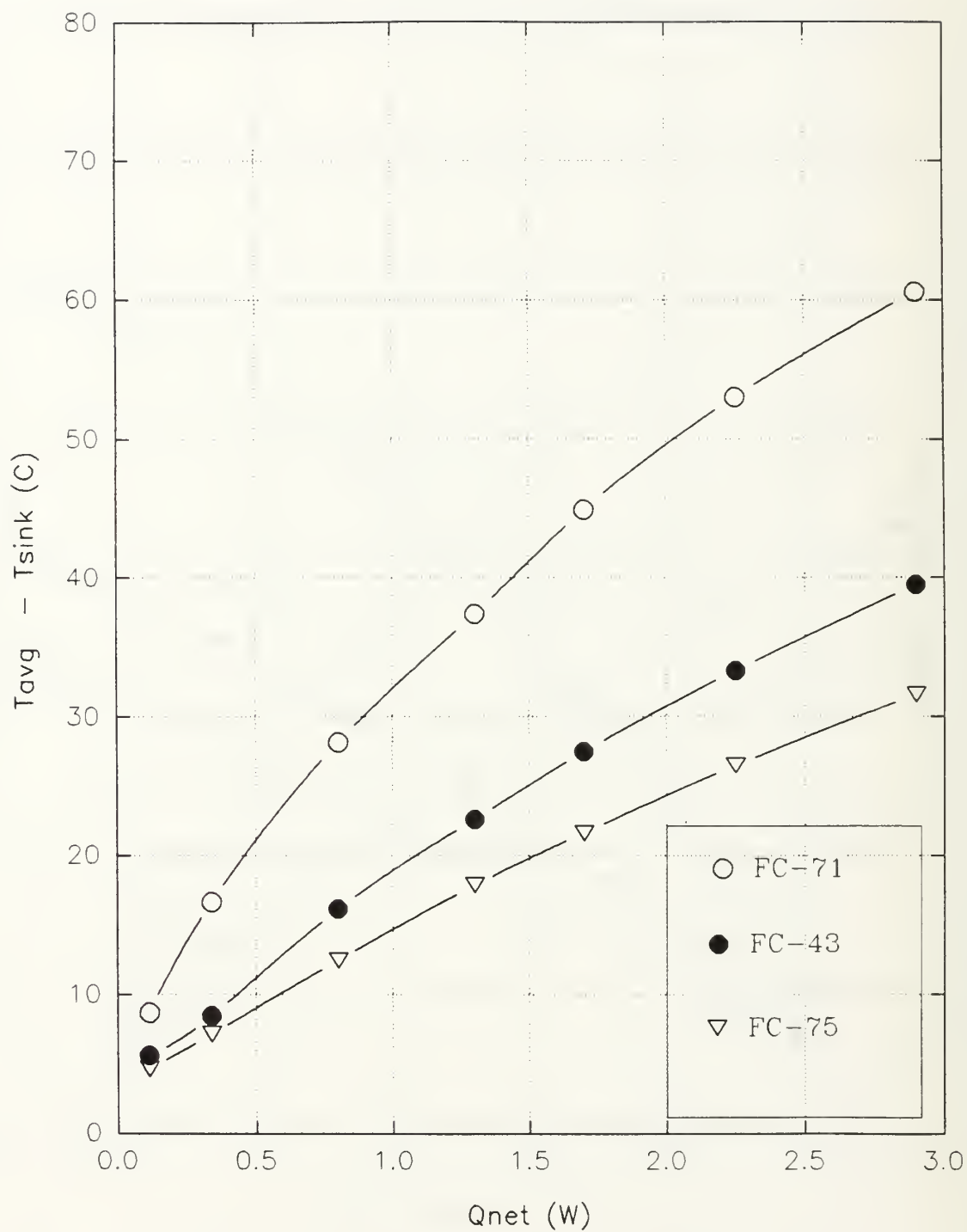


Figure 14. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 40 mm Spacing, Vertical Orientation

For the 0.115 W power level, only 0.5 °C or less separated all chip temperatures for the 7, 9, and 16 mm spacings. This is within the uncertainty of the thermocouple measurement.

C. NON-DIMENSIONAL RESULTS

1. General

The following maximums and minimums for all power levels and enclosure widths were noted:

- The maximum value of the flux based Rayleigh number, Ra_f , was 616.4×10^6 . It occurred on chip #3 at a 2.9 W power level and 7 mm spacing. The minimum Ra_f was 1.417×10^6 on chip #7 at 0.115 W and a 40 mm spacing.
- The maximum temperature based Rayleigh number, Ra , was 31.11×10^6 . It occurred on chip #3 under the same conditions as the Ra_f maximum listed above. The minimum Ra was 190,000 on chip #7 at 0.115 W and a 40 mm spacing.
- The maximum Nu was 28.88 on chip #4 at 2.9 W and a 28 mm spacing. The minimum Nu of 5.40 was noted on chip #7 at 0.115 W and a 7 mm spacing.
- The maximum uncertainty in Ra and Nu were 6.90% and 7.39%, respectively. Both values were calculated on chip #7 at a power of 0.115 W and a spacing of 9 mm.

Both Aytar and Matthews plotted the array averaged Nu as a function of either the array averaged Ra_f or Ra . When the data taken at 0.115 W was omitted, it was found that these plots were straight lines, independent of enclosure width. This was done since the uncertainty was highest at this power level, and the resulting chip temperatures were less than or equal to the ambient temperature. Figure 15 is a similar plot for FC-71. For comparison, Nu versus Ra for all three dielectric liquids have been plotted for each enclosure width in Figure 16 through Figure 20. In these figures, it is noted that the slopes of any particular dielectric liquid are virtually identical in nature.

2. Effect of Rayleigh Number

Matthews and Aytar both observed a linear relationship when log Nu was plotted against log Ra . This relationship was of the following form:

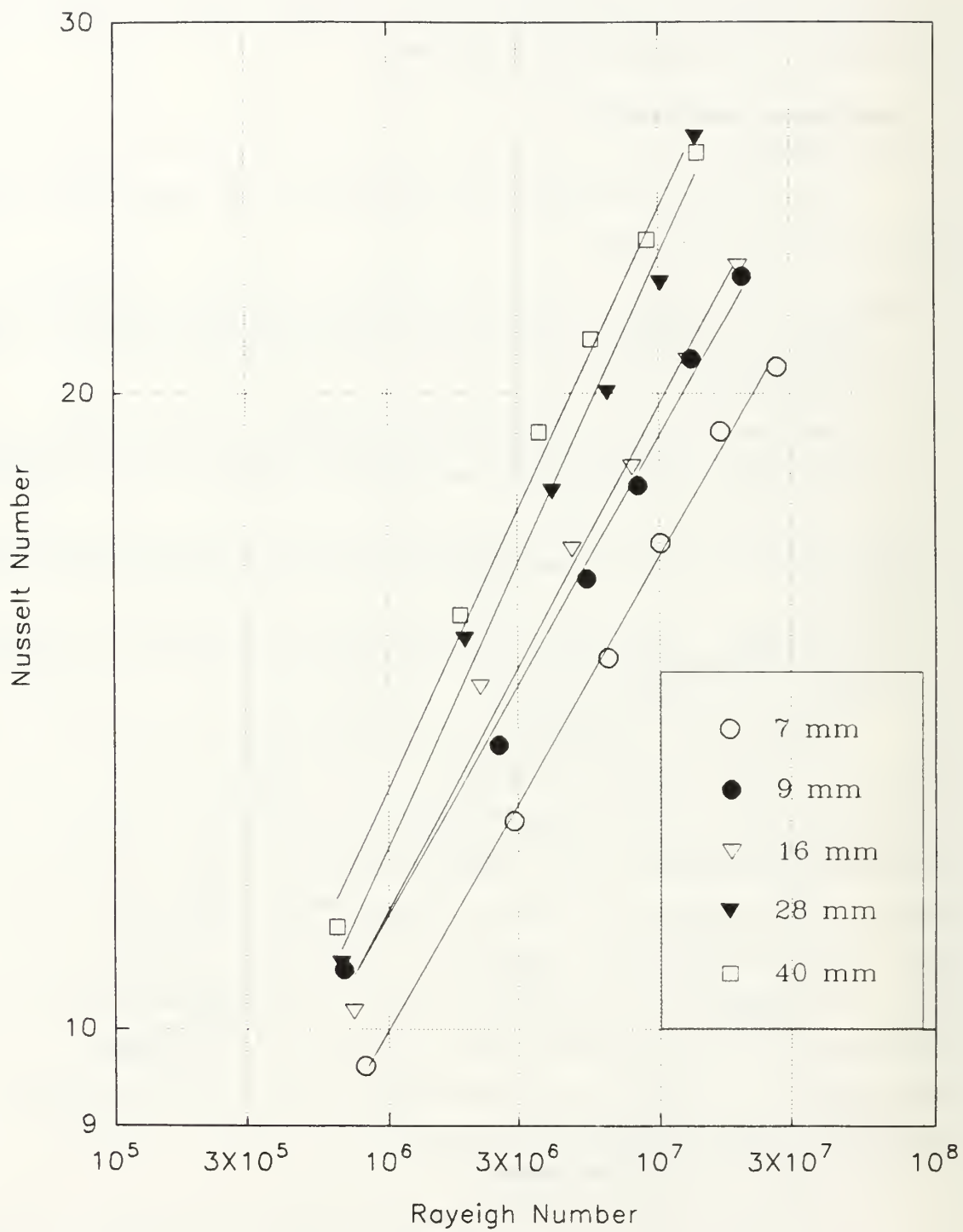


Figure 15. Nu vs. Ra for FC-71, Array Averaged, Vertical Orientation, and all Enclosure Widths

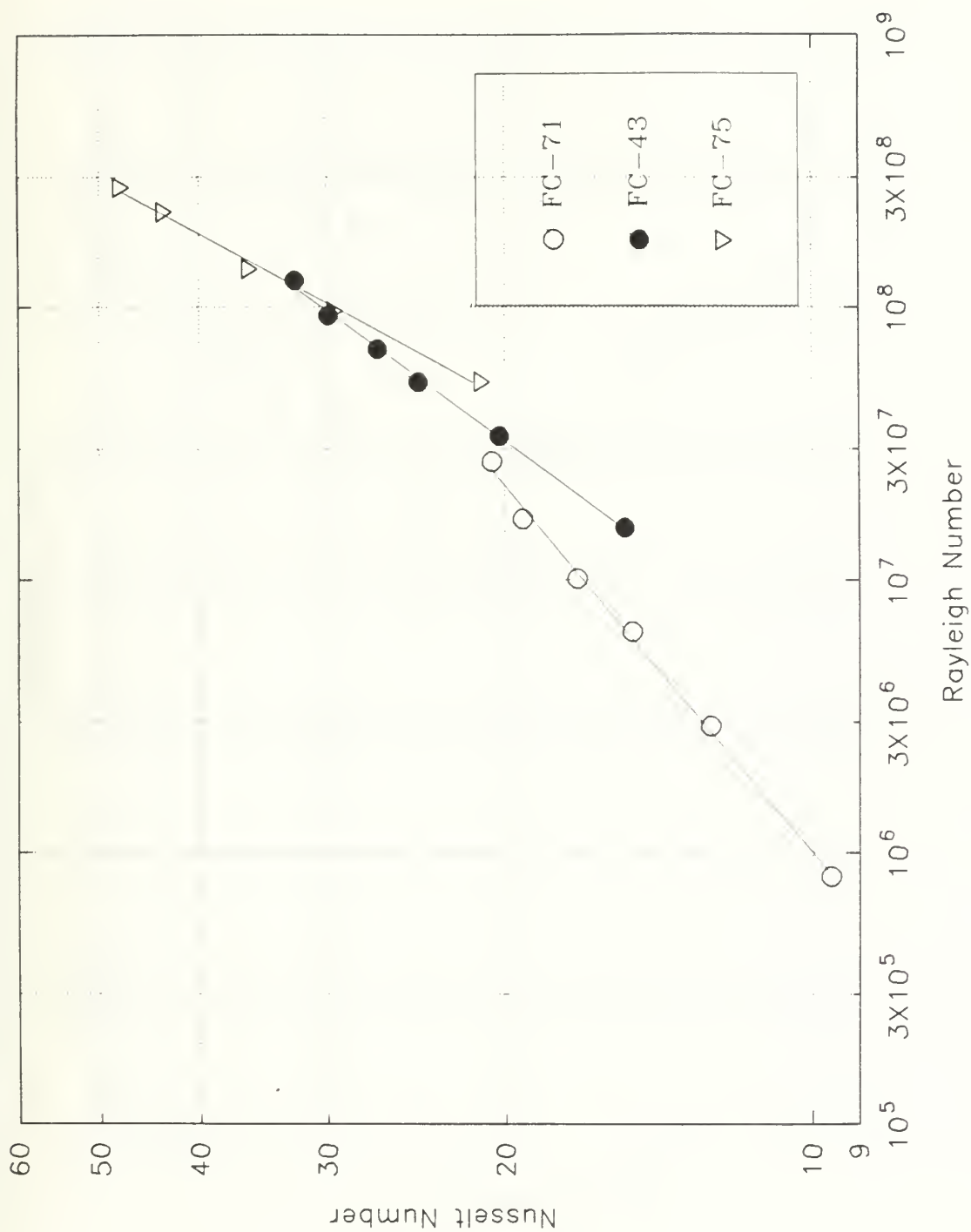


Figure 16. Nu vs. Ra for FC-71, 43, and 75, Array Averaged, Vertical Orientation, and 7 mm Enclosure Width

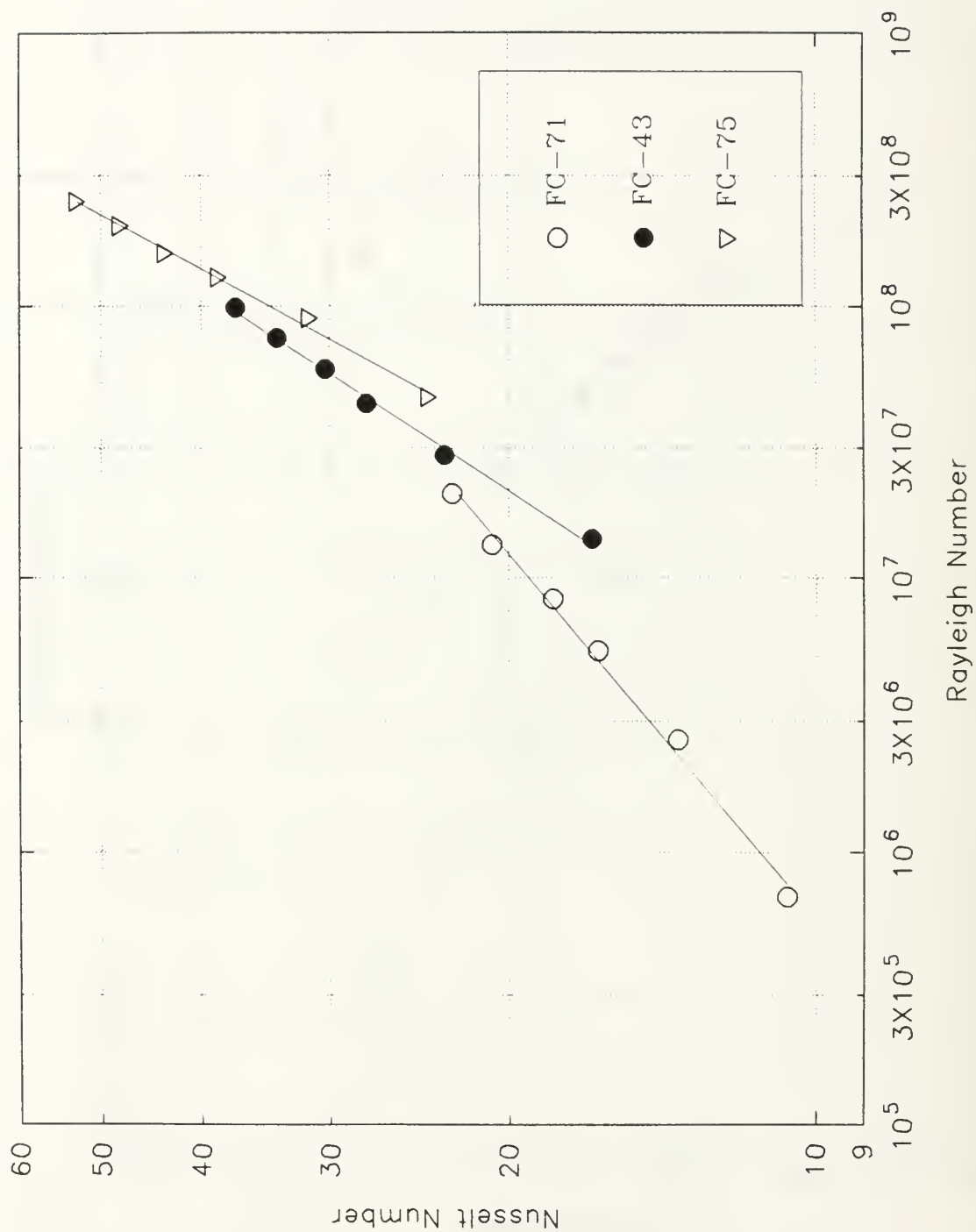


Figure 17. Nu vs. Ra for FC-71, 43, and 75, Array Averaged, Vertical Orientation, and 9 mm Enclosure Width

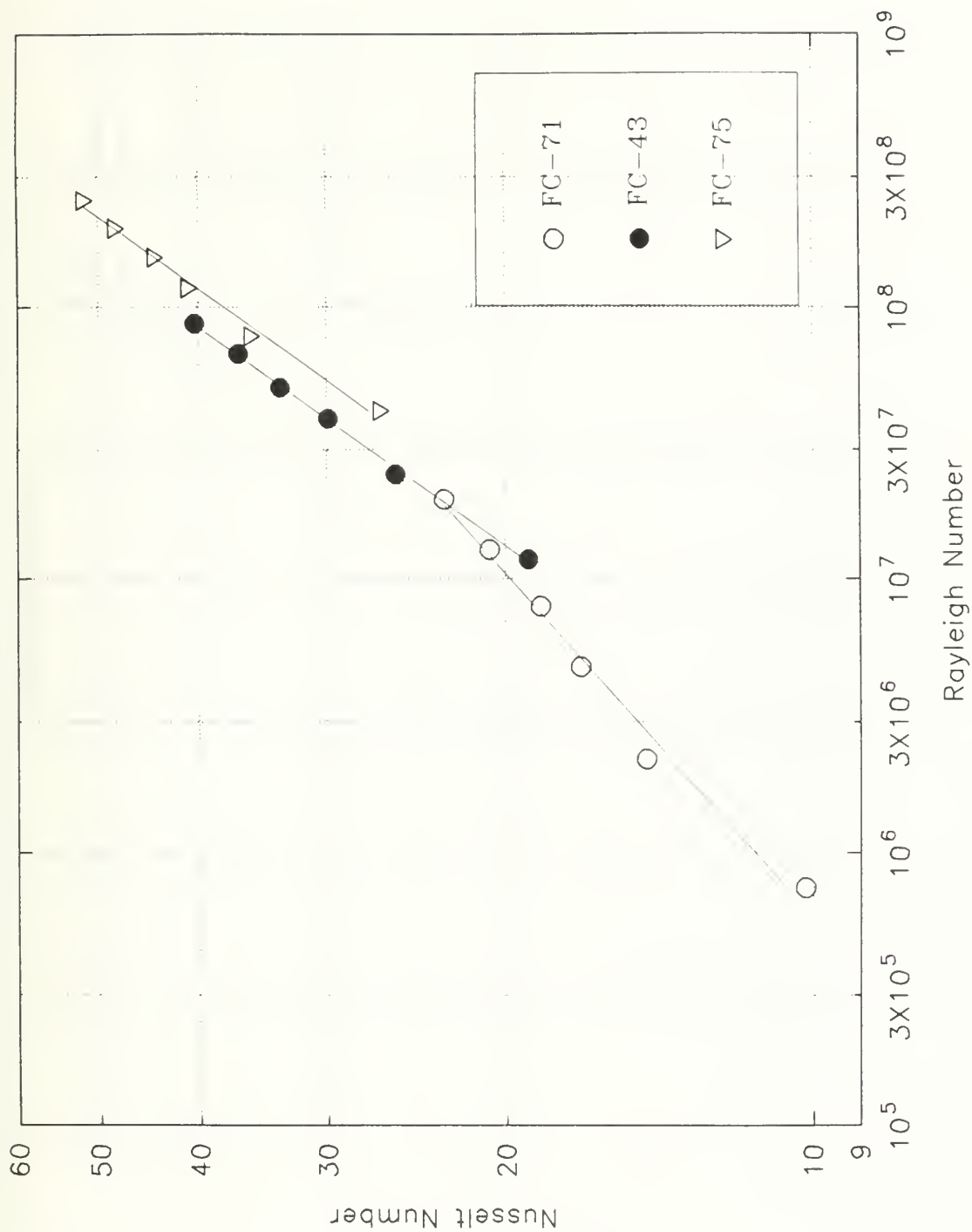


Figure 18. Nu vs. Ra for FC-71, 43, and 75, Array Averaged, Vertical Orientation, and 16 mm Enclosure Width

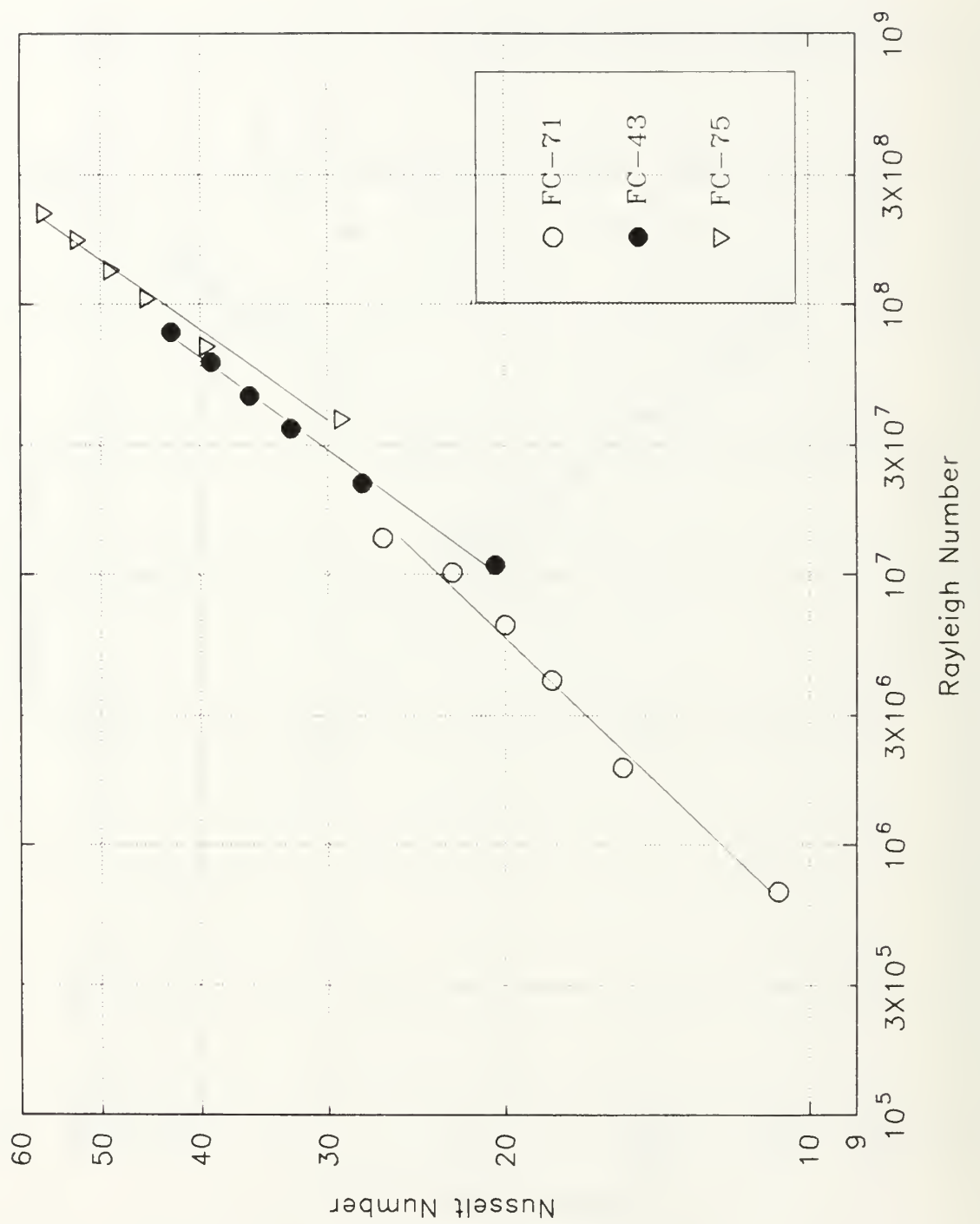


Figure 19. Nu vs. Ra for FC-71, 43, and 75, Array Averaged, Vertical Orientation, and 28 mm Enclosure Width

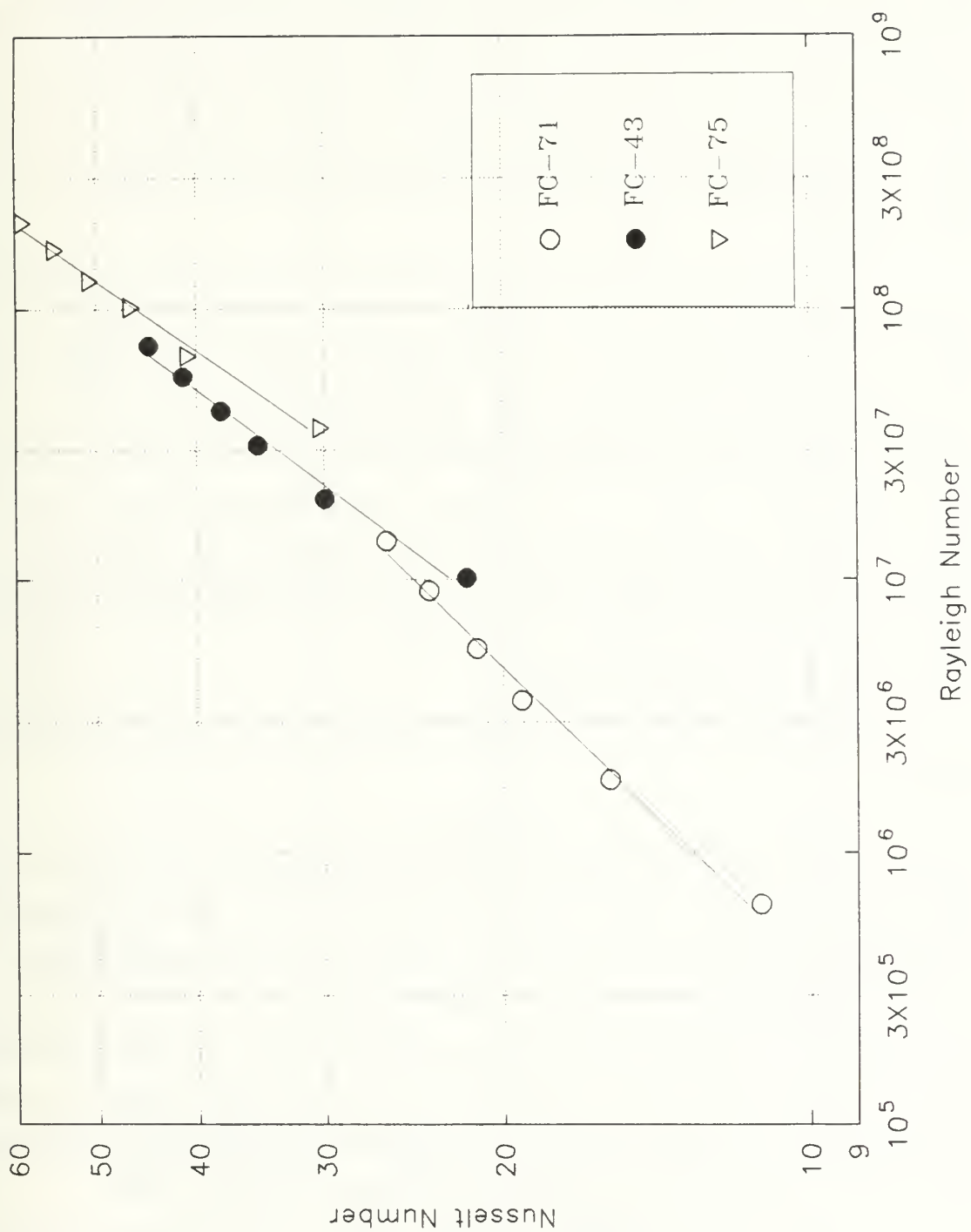


Figure 20. Nu vs. Ra for FC-71, 43, and 75, Array Averaged, Vertical Orientation, and 40 mm Enclosure Width

$$Nu = c_1 Ra^{b_1}$$

where c_1 and b_1 are constants. Both row averaged and array averaged values were calculated and plotted for each spacing. The curve fit software TABLECURVE (1990) was used to find the coefficients of the above equation, and the software package SIGMAPLOT (1989) was used to produce the graphs. Again, for consistency and correlation accuracy, the data taken at 0.115 W was omitted.

Figure 21 through Figure 25 are the FC-71 data array averaged Nu versus Ra plots for the five spacings. The corresponding curve fit equation is also included. For completeness, Figure 26 through 40 are similar plots for the spacings using row averaged data. The value of the constant b_1 was found to vary from 0.225 to 0.280 for the array averaged data. The average value of b_1 was 0.249. Corresponding values of 0.381 and 0.371 were reported by Matthews for FC-75 and FC-43, respectively. The constant c_1 varied from 0.255 to 0.514 for the same array averaged data, and the average value of c_1 was 0.383.

3. Effect of Enclosure Width

Enclosure width effects on Nu were accounted for in the following equation:

$$Nu = c_2 X^{b_2}$$

where b_2 is a constant and c_2 is the Ra dependence. X is the non-dimensional enclosure width, which is the ratio of the actual spacing to the maximum spacing of 40 mm. The constant c_2 can be represented by:

$$c_2 = c_1 Ra^{b_1}$$

The constant b_2 was derived from a plot of Nu vs. X for the five spacings, which is shown in Figure 41. The values for Nu were taken from the curve fit equations for each spacing's array averaged data using an average Ra of 4×10^6 . The resulting value for the exponent b_2 was 0.165.

Combining the above results gives the following general correlation for FC-71:

$$Nu = 0.383 Ra^{0.249} X^{0.165}$$

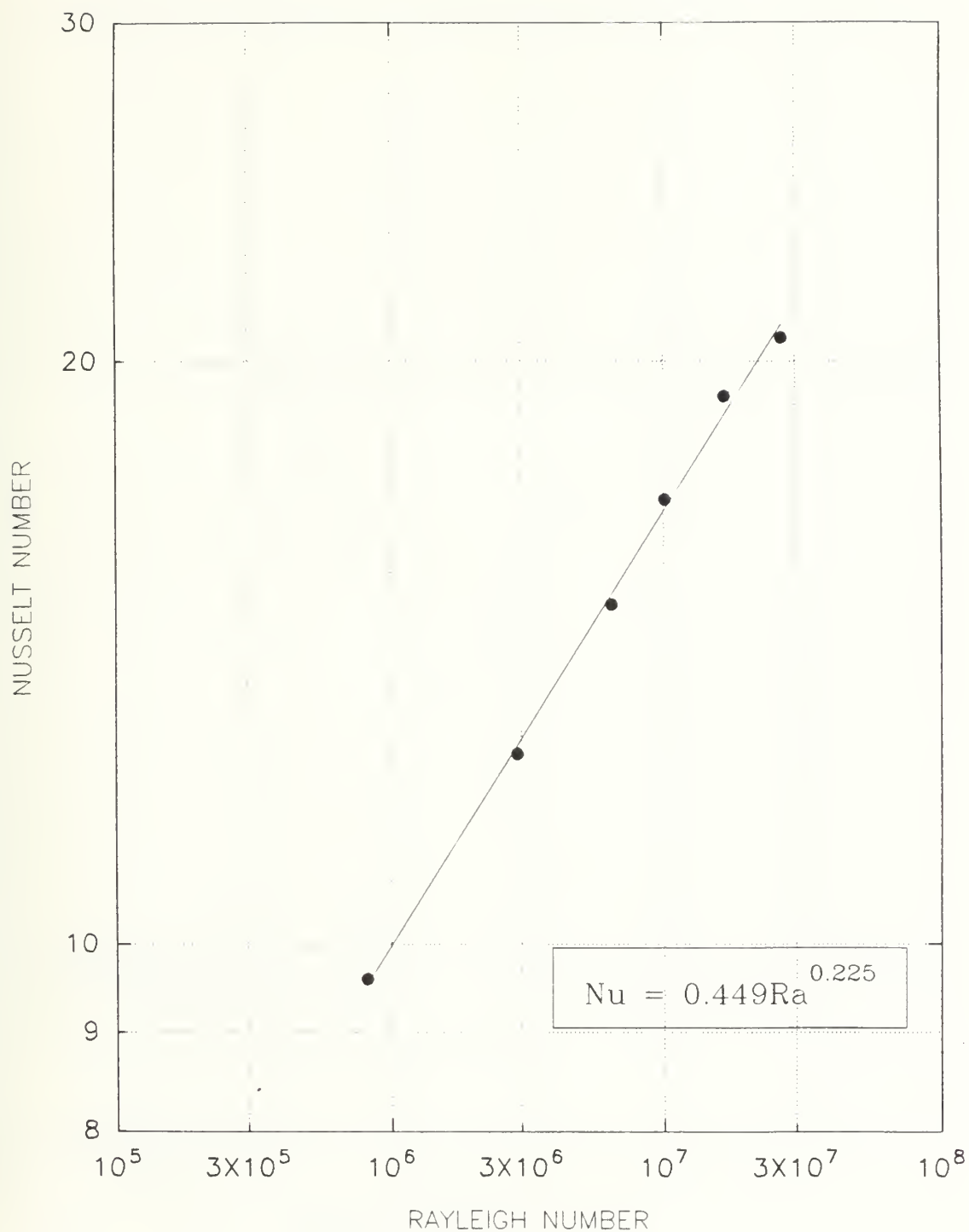


Figure 21. Nu vs. Ra for FC-71, Array Averaged, Vertical Orientation, and 7 mm Spacing

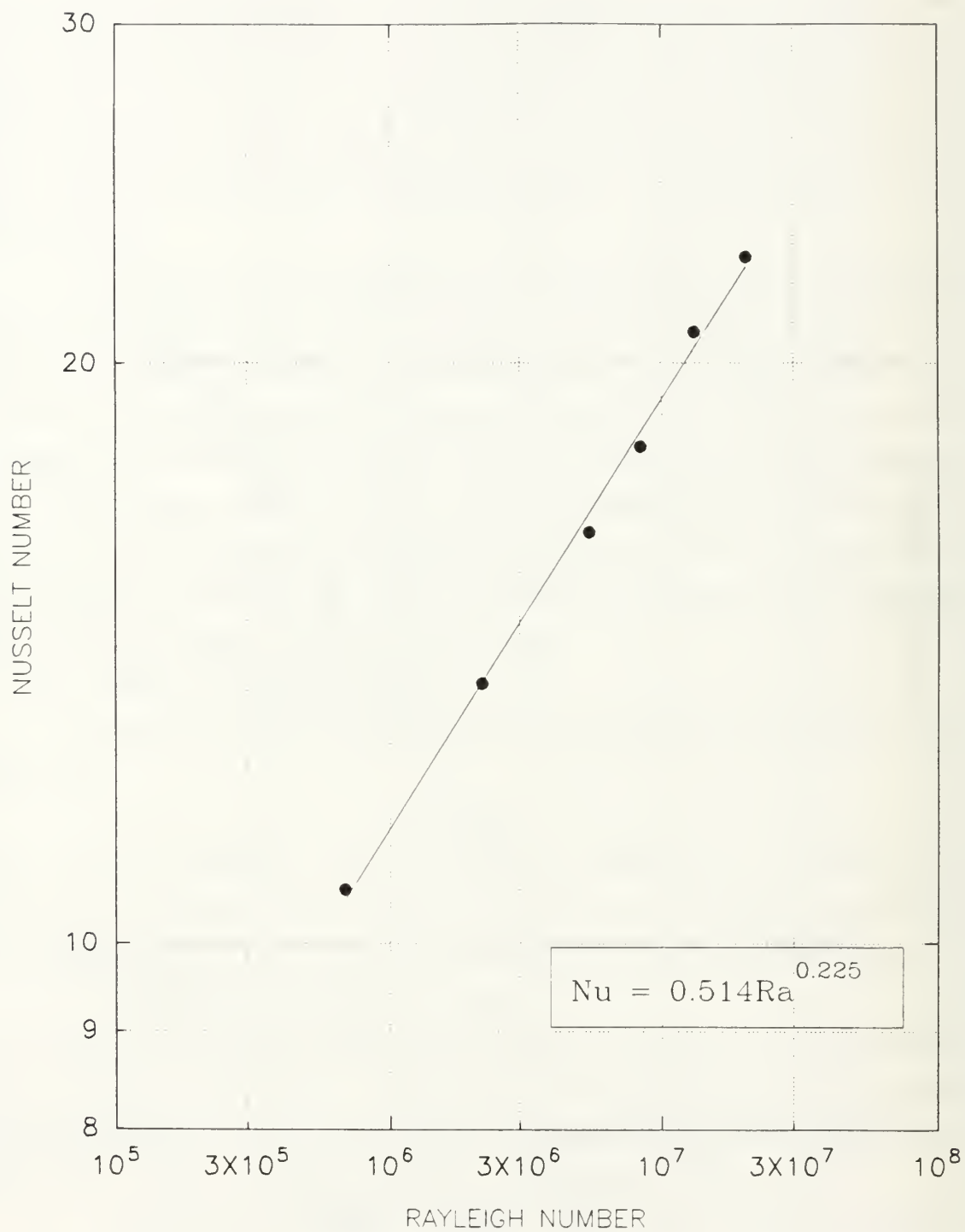


Figure 22. Nu vs. Ra for FC-71, Array Averaged, Vertical Orientation, and 9 mm Spacing



Figure 23. Nu vs. Ra for FC-71, Array Averaged, Vertical Orientation, and 16 mm Spacing

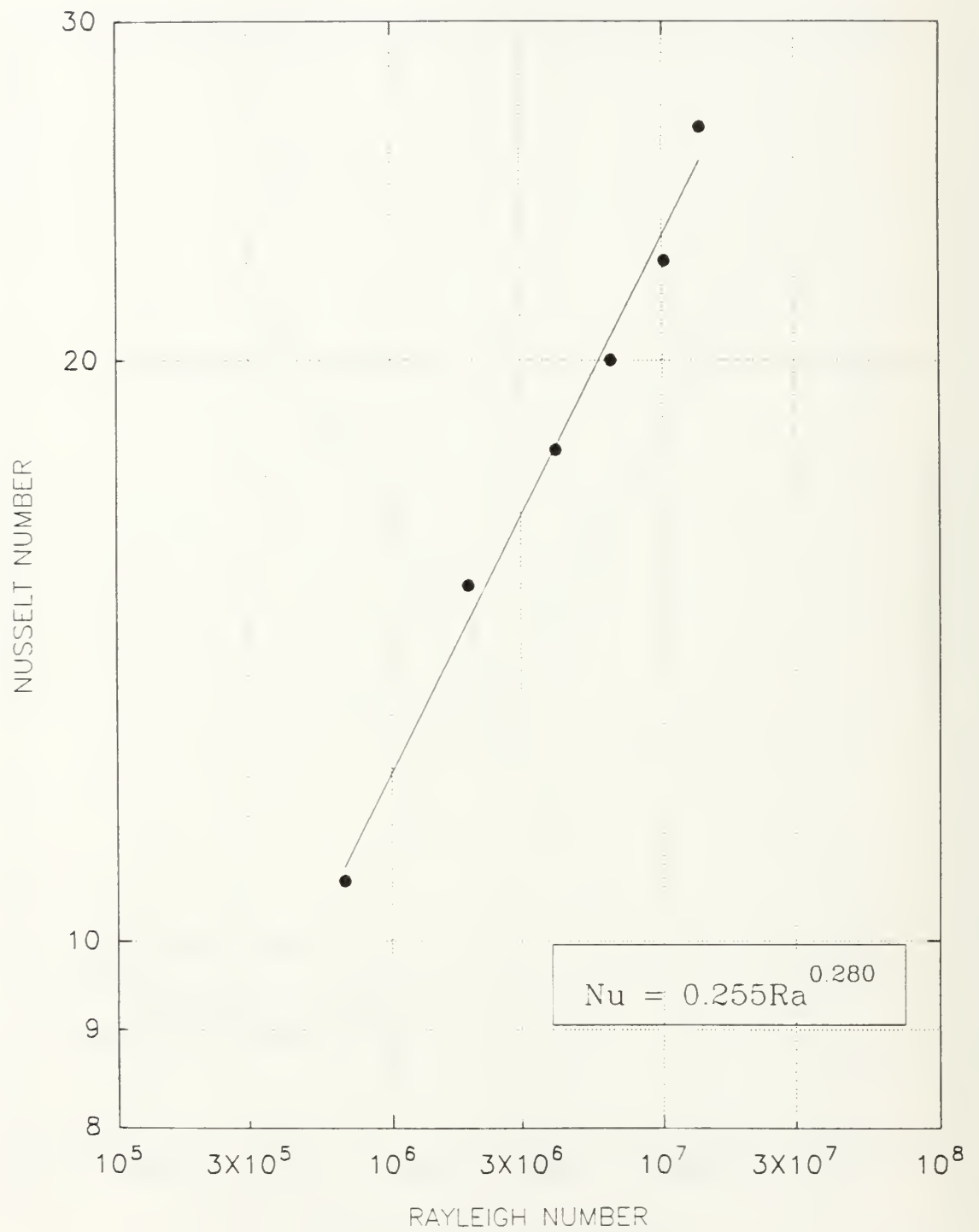


Figure 24. Nu vs. Ra for FC-71, Array Averaged, Vertical Orientation, and 28 mm Spacing

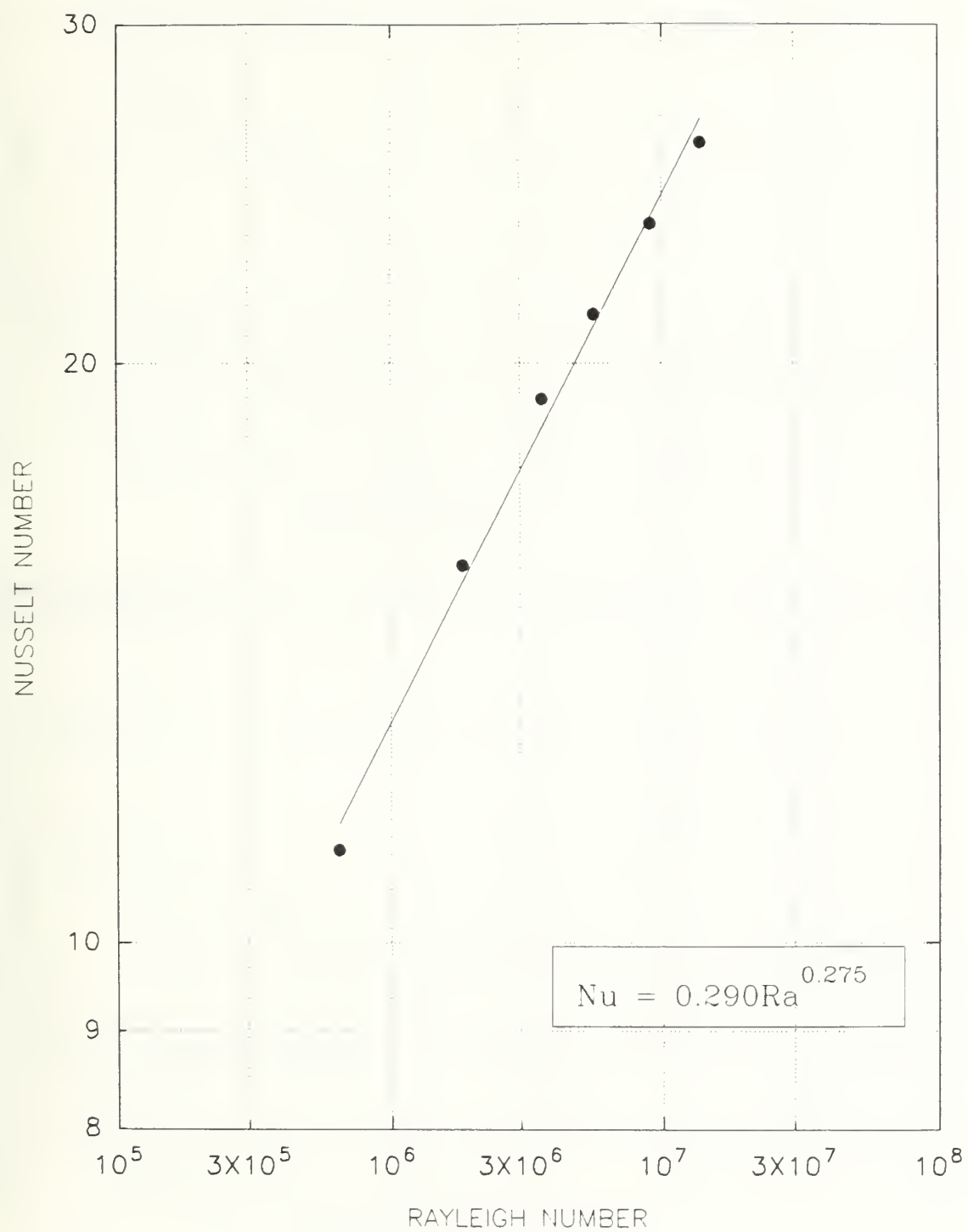


Figure 25. Nu vs. Ra for FC-71, Array Averaged, Vertical Orientation, and 40 mm Spacing

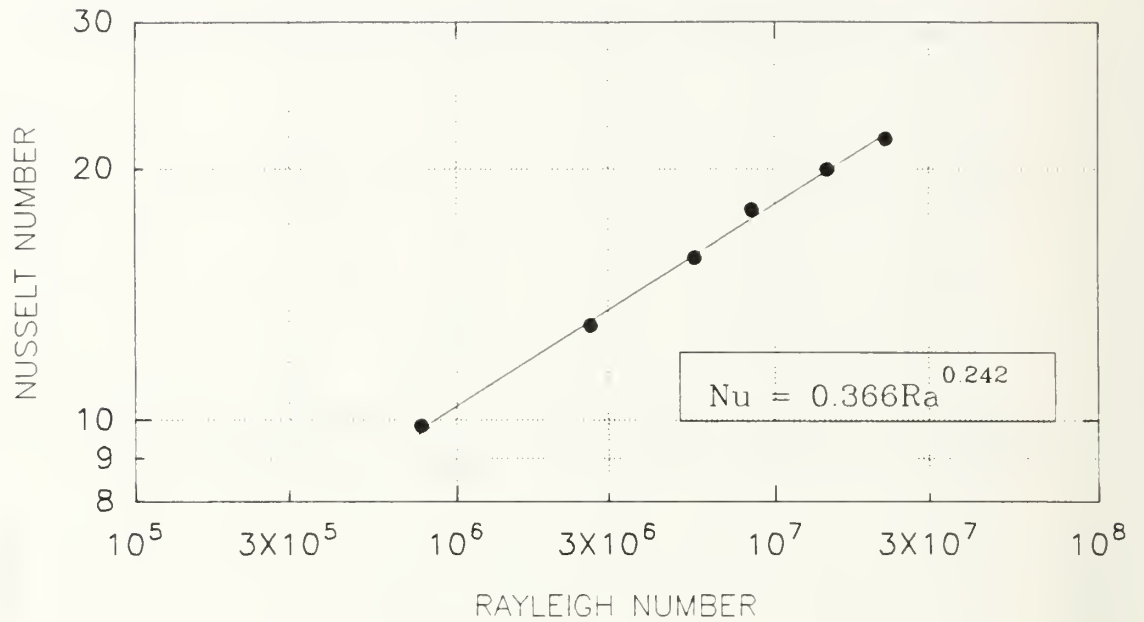


Figure 26. Nu vs. Ra for FC-71, Bottom Row Averaged, Vertical Orientation, and 7 mm Spacing

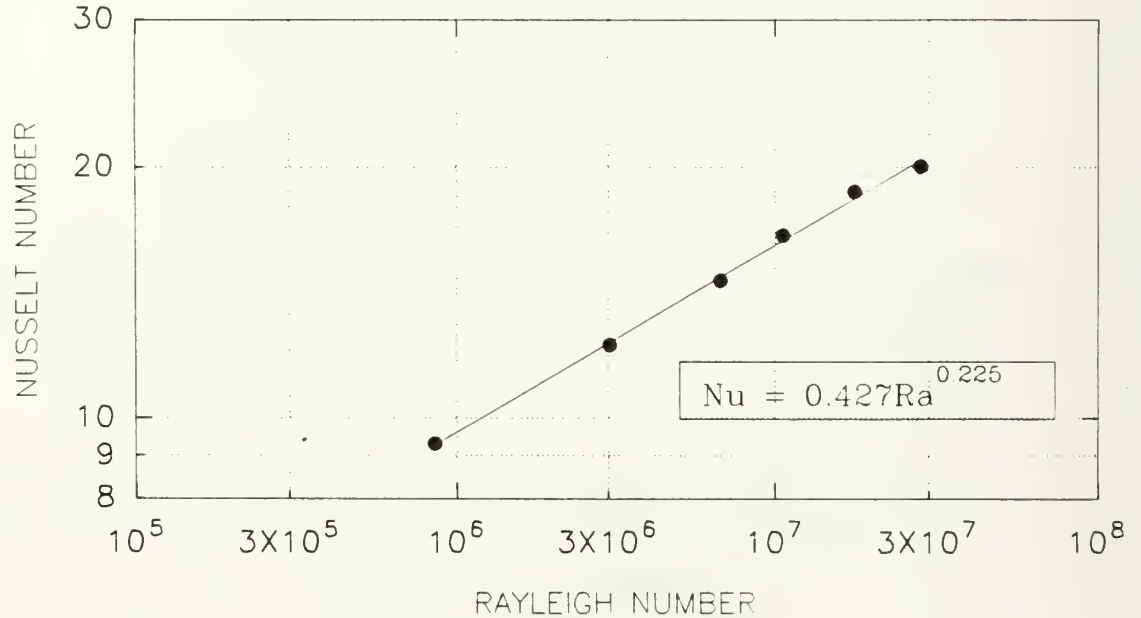


Figure 27. Nu vs. Ra for FC-71, Middle Row Averaged, Vertical Orientation, and 7 mm Spacing

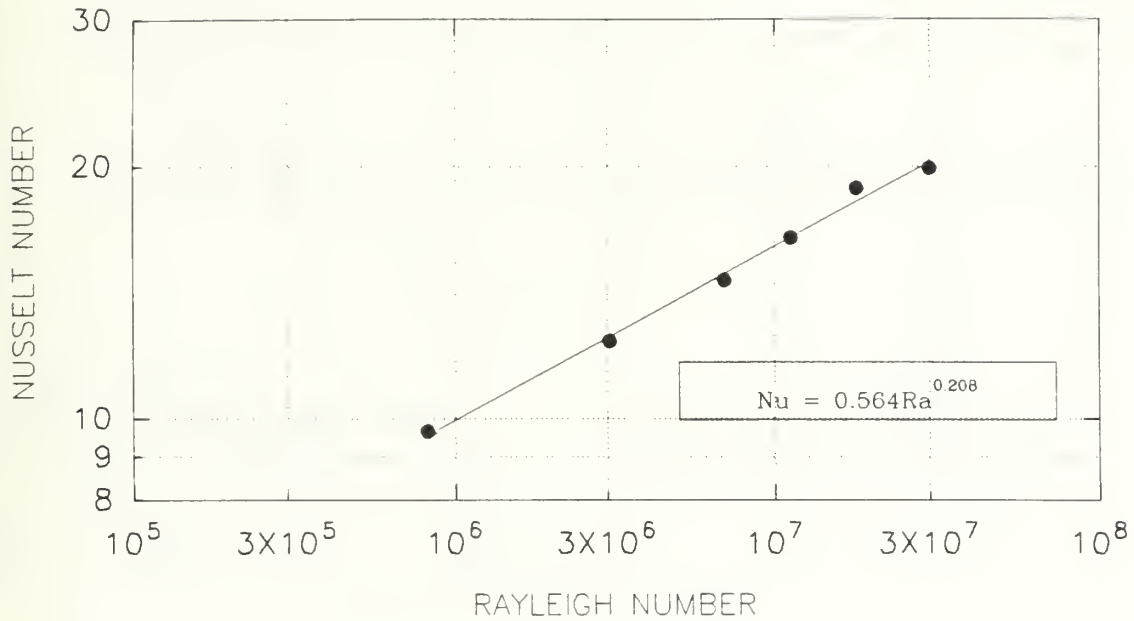


Figure 28. Nu vs. Ra for FC-71, Top Row Averaged, Vertical Orientation, and 7 mm Spacing

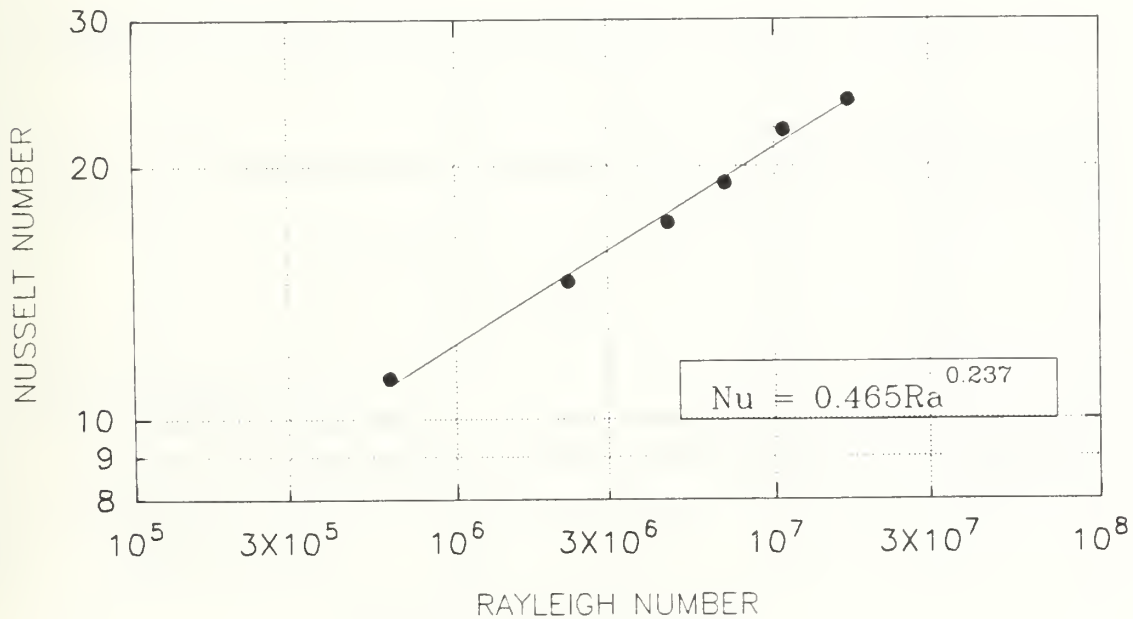


Figure 29. Nu vs. Ra for FC-71, Bottom Row Averaged, Vertical Orientation, and 9 mm Spacing

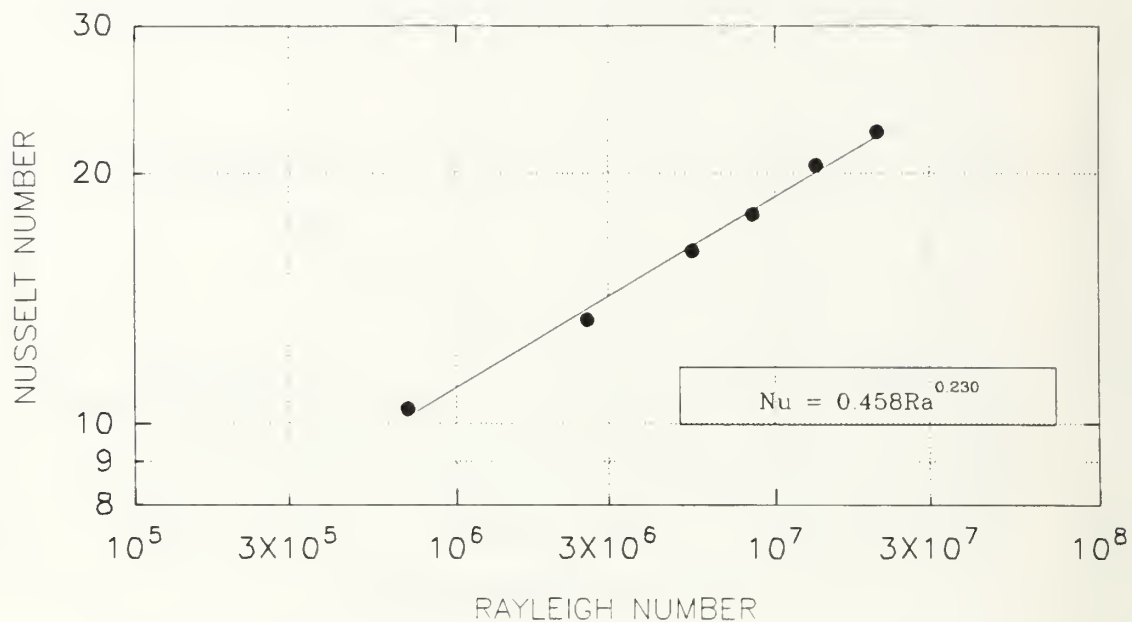


Figure 30. Nu vs. Ra for FC-71, Middle Row Averaged, Vertical Orientation, and 9 mm Spacing

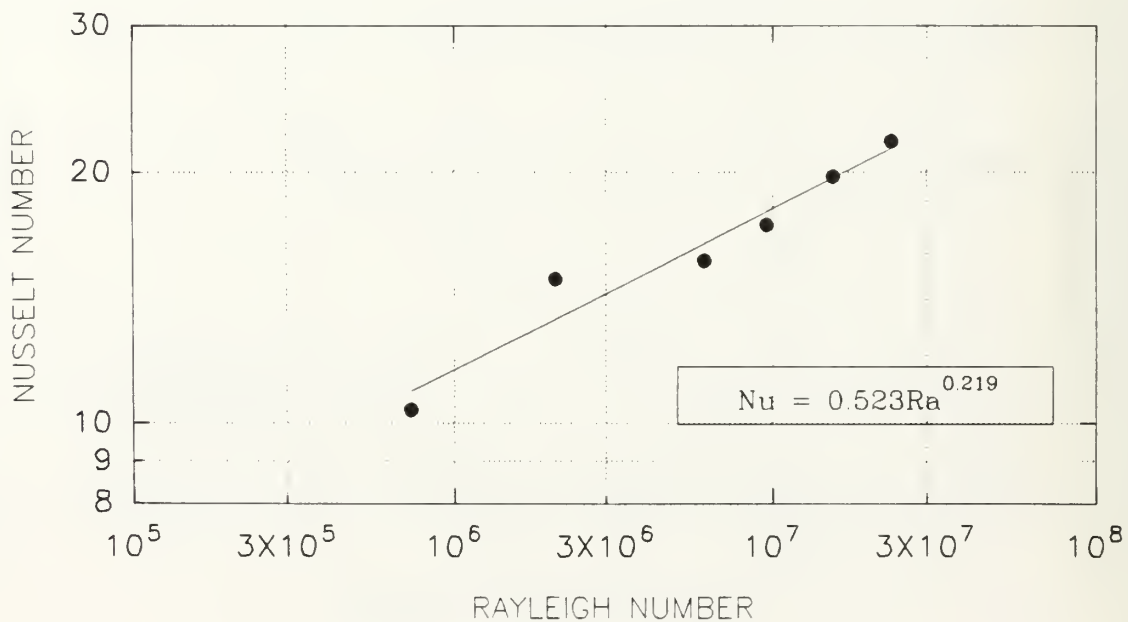


Figure 31. Nu vs. Ra for FC-71, Top Row Averaged, Vertical Orientation, and 9 mm Spacing

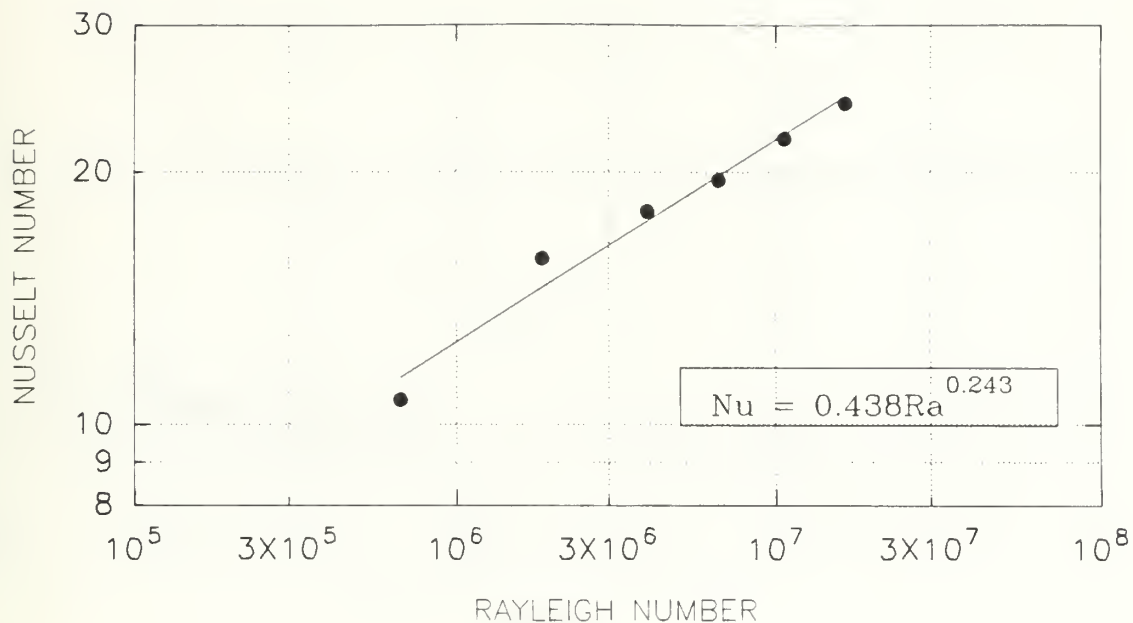


Figure 32. Nu vs. Ra for FC-71, Bottom Row Averaged, Vertical Orientation, and 16 mm Spacing

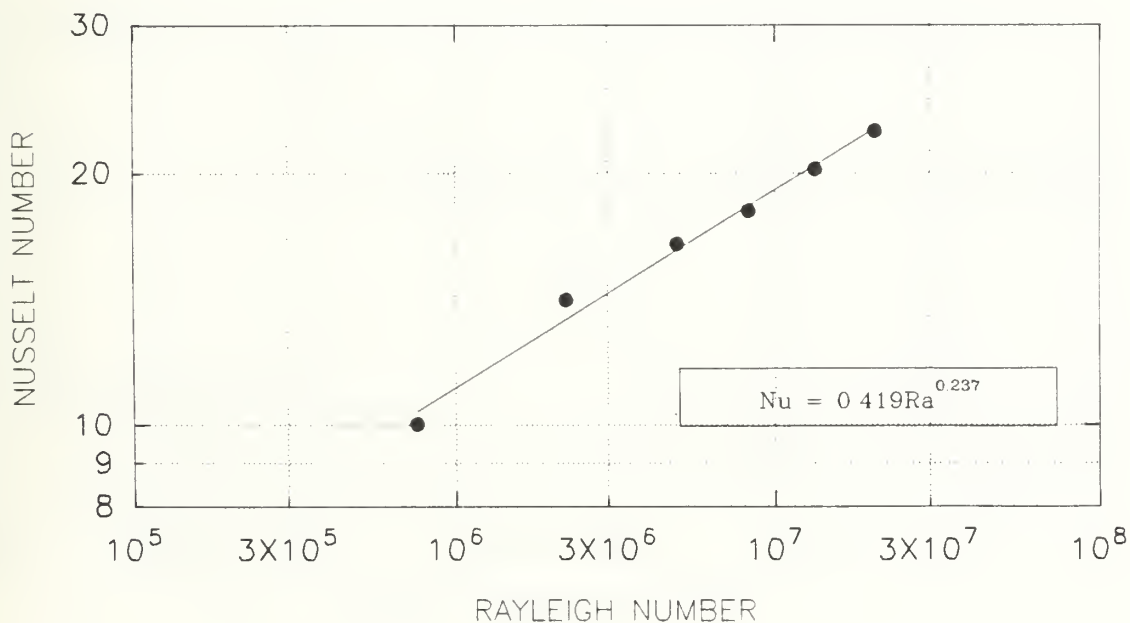


Figure 33. Nu vs. Ra for FC-71, Middle Row Averaged, Vertical Orientation, and 16 mm Spacing

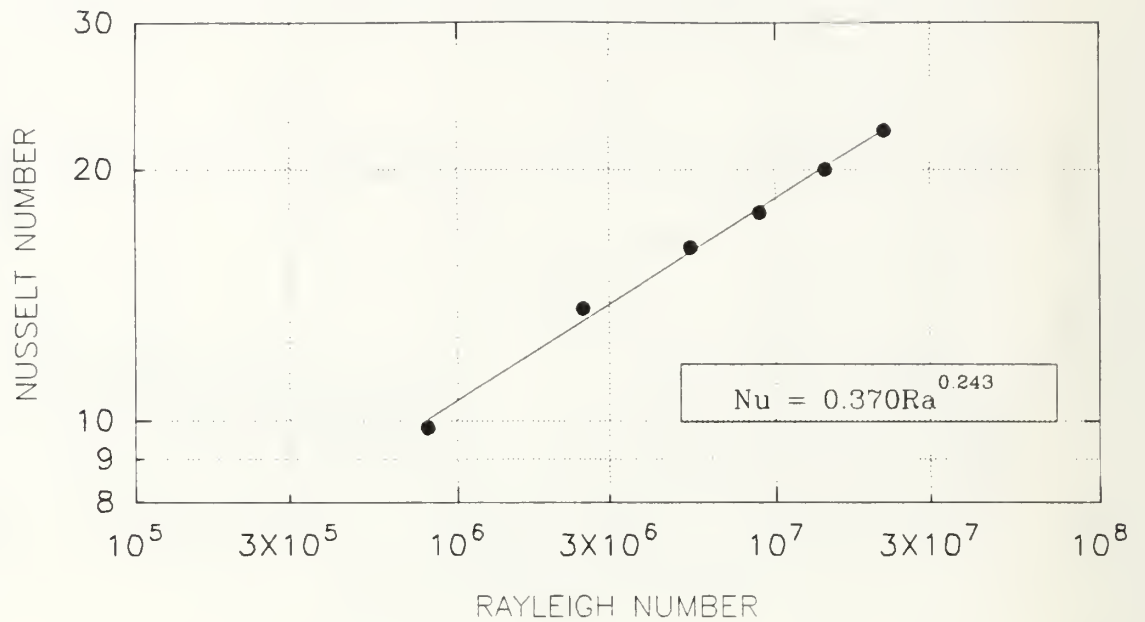


Figure 34. Nu vs. Ra for FC-71, Top Row Averaged, Vertical Orientation, and 16 mm Spacing

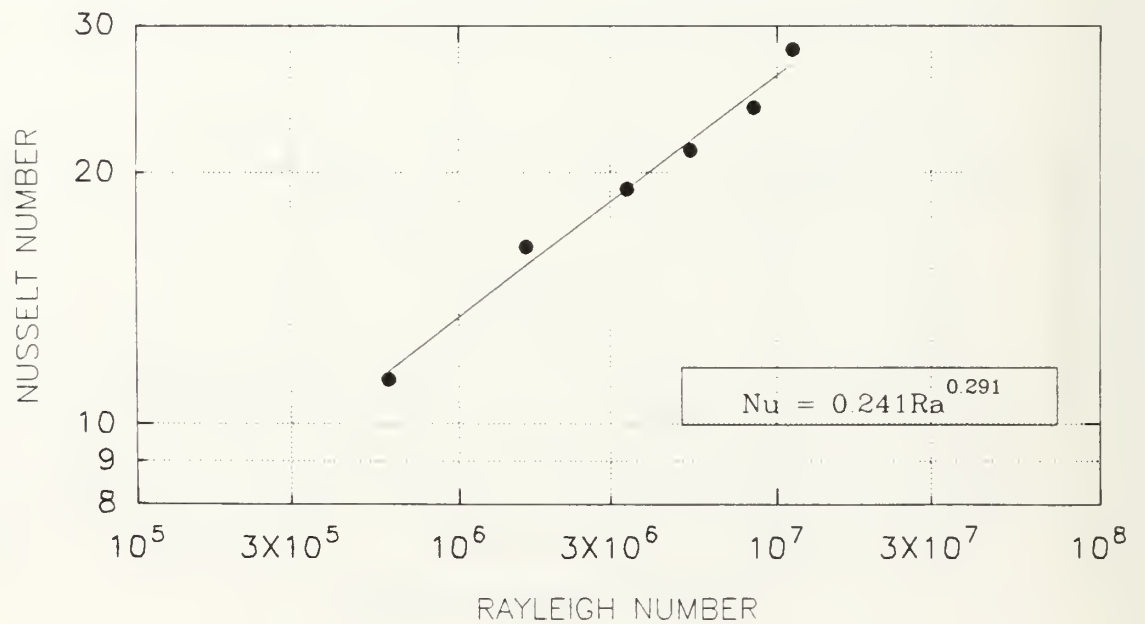


Figure 35. Nu vs. Ra for FC-71, Bottom Row Averaged, Vertical Orientation, and 28 mm Spacing

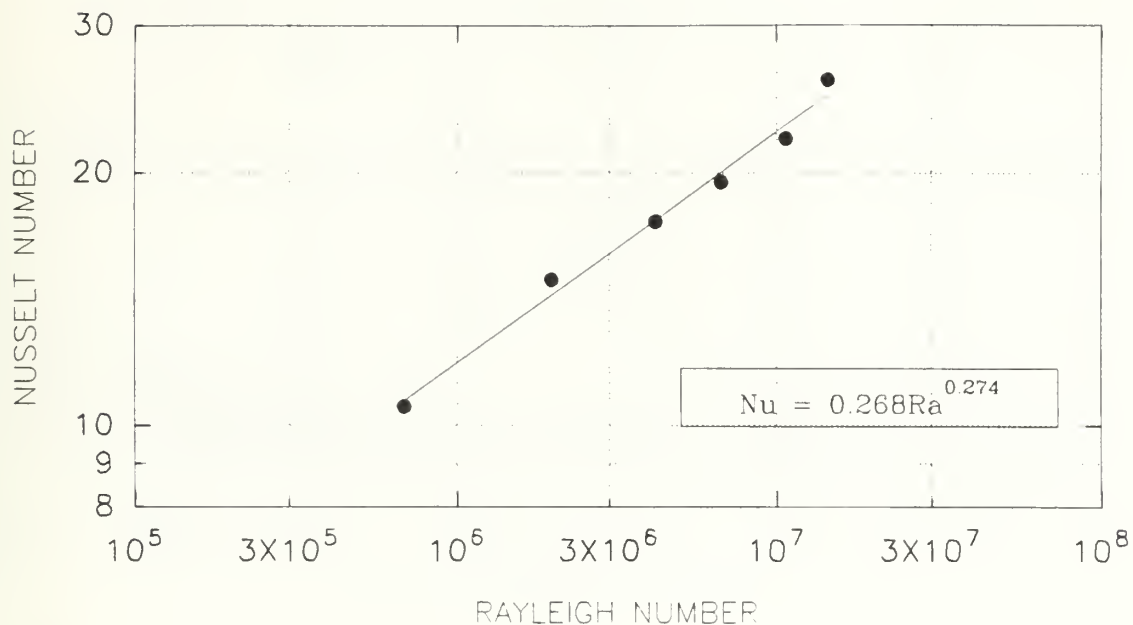


Figure 36. Nu vs. Ra for FC-71, Middle Row Averaged, Vertical Orientation, and 28 mm Spacing

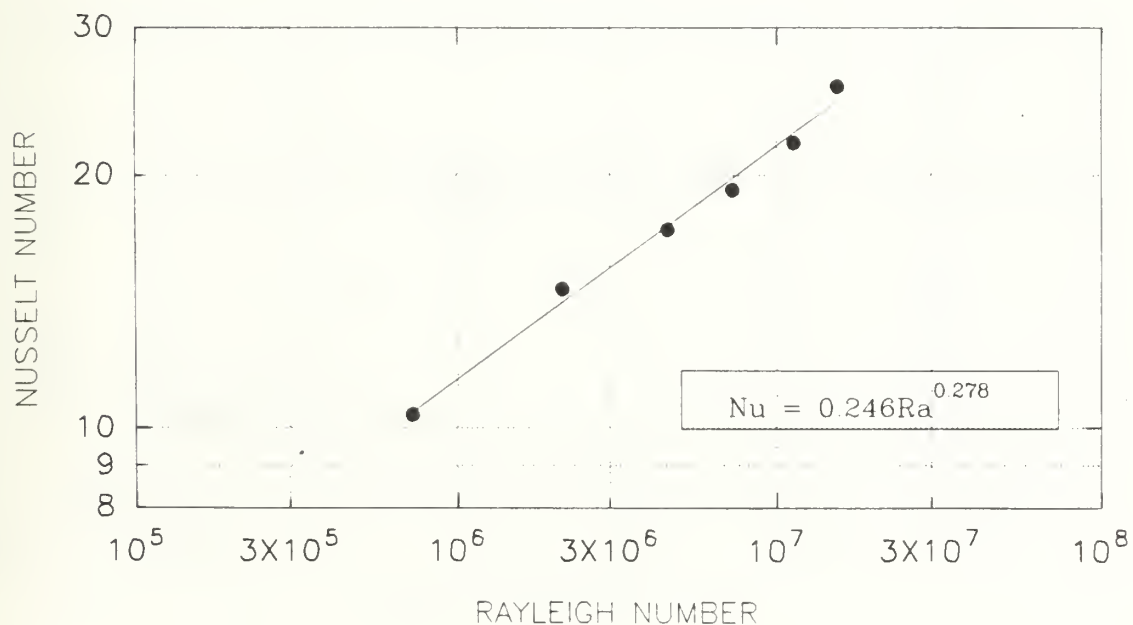


Figure 37. Nu vs. Ra for FC-71, Top Row Averaged, Vertical Orientation, and 28 mm Spacing

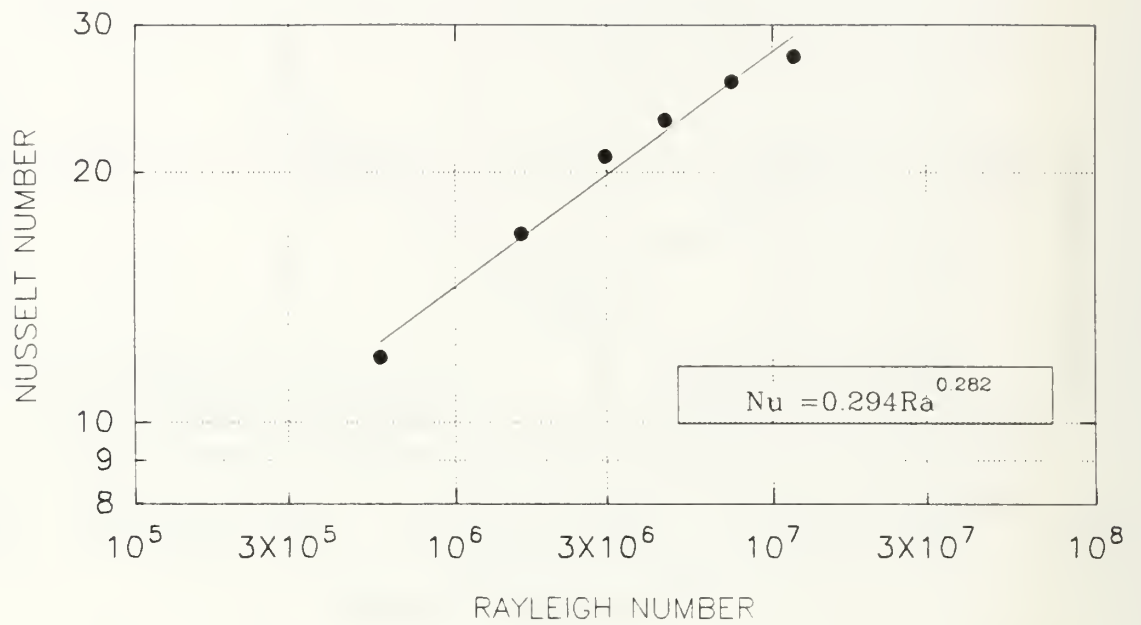


Figure 38. Nu vs. Ra for FC-71, Bottom Row Averaged, Vertical Orientation, and 40 mm Spacing

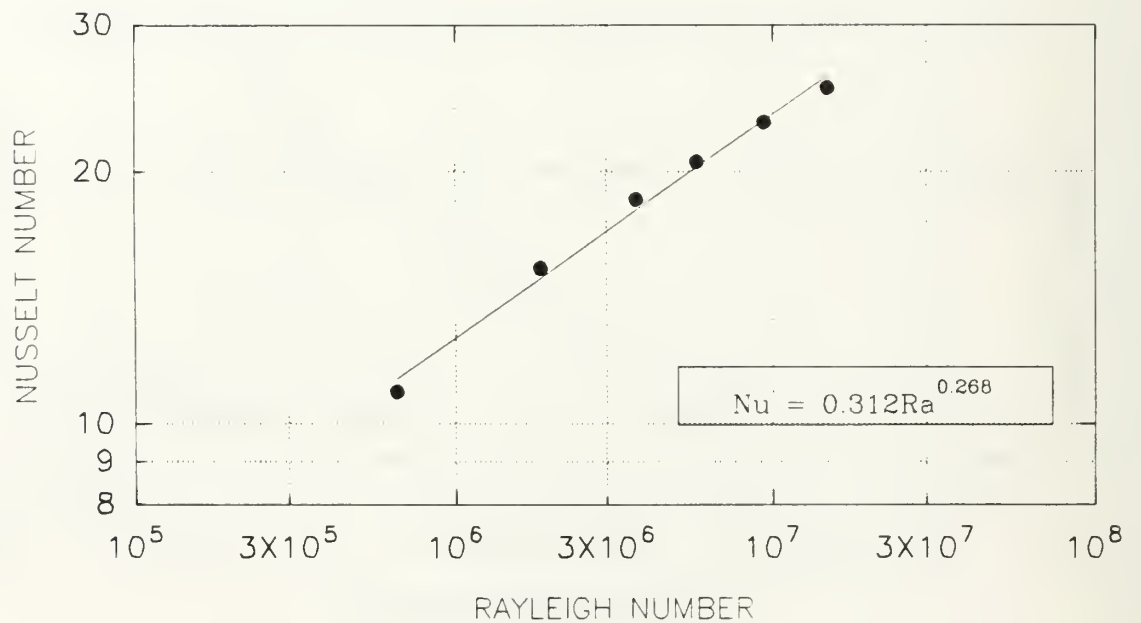


Figure 39. Nu vs. Ra for FC-71, Middle Row Averaged, Vertical Orientation, and 40 mm Spacing

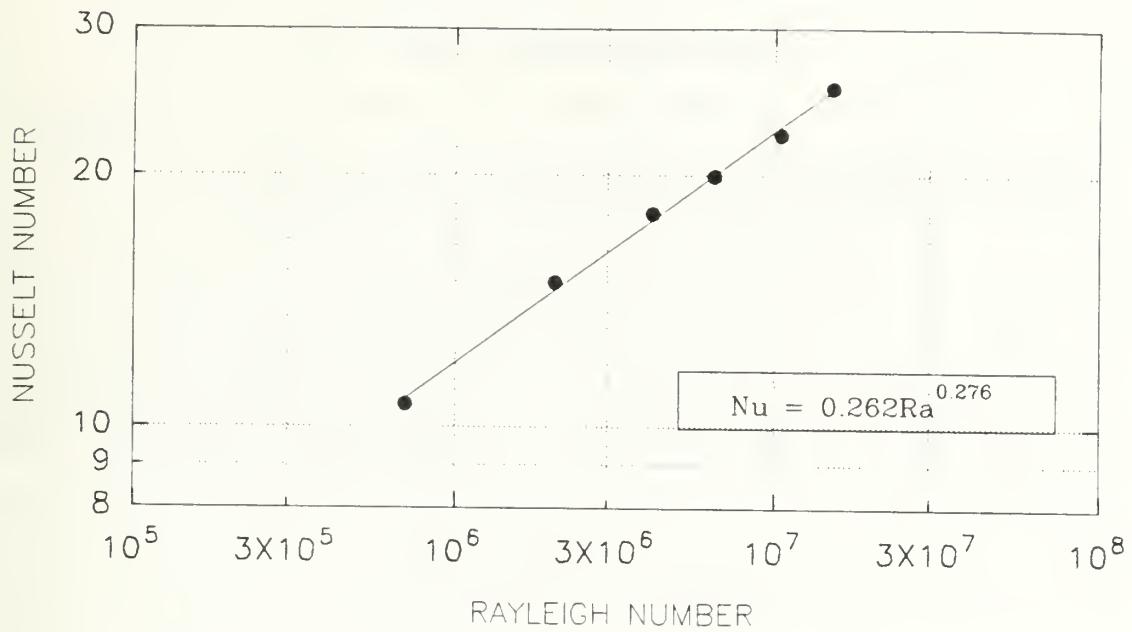


Figure 40. Nu vs. Ra for FC-71, Top Row Averaged, Vertical Orientation, and 40 mm Spacing

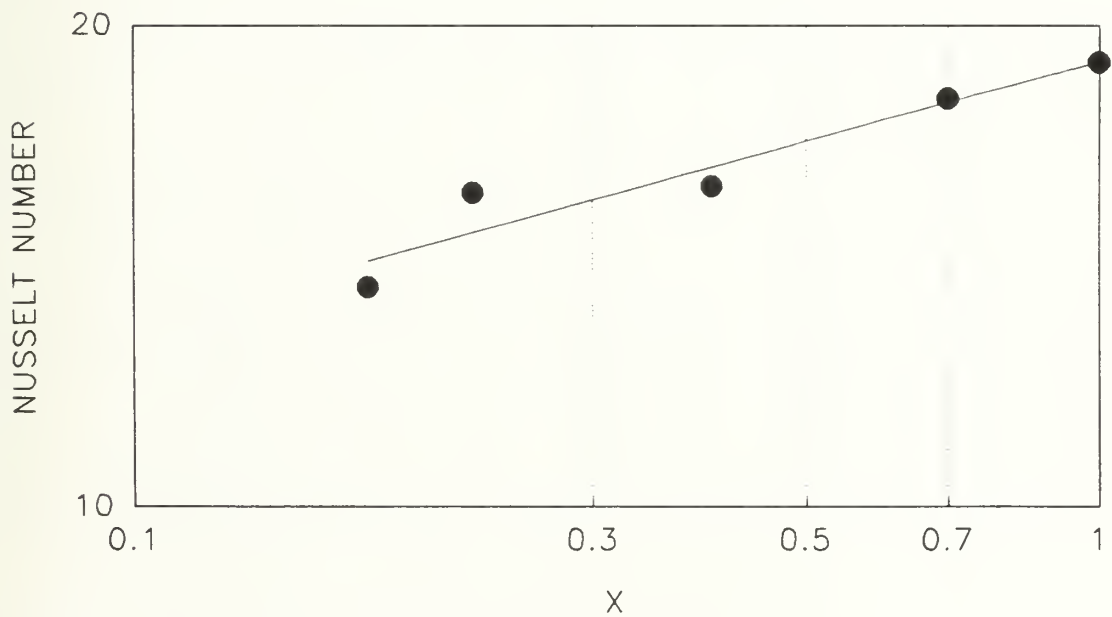


Figure 41. Nu vs. X for FC-71, Array Averaged, Vertical Orientation, and all Enclosure Widths

It is valid over the ranges:

$$9 \times 10^5 < Ra < 2 \times 10^7$$

$$0.175 < X < 1.0$$

However, this correlation was found to be accurate only to within 11% of the array averaged curve fit equations. In order to achieve improved accuracy, representative values of Ra were chosen for the general correlation and each spacing's correlation. The Nu number results were then compared, and a trial and error approach was used to come up with a better value for c_1 . When the value for c_1 was changed from 0.383 to 0.435, the accuracy improved to 4%. Therefore, a better correlation for FC-71 is:

$$Nu = 0.435Ra^{0.249}X^{0.165}$$

It is valid over the same ranges as listed above.

VI. NSWC CIRCUIT BOARD RESULTS

A. GENERAL

The next logical step in the dielectric liquid natural convection studies was to replace simulated electronic components with the actual electronic devices. A three by three array of 8.9 mm square thermal evaluation devices were tested on a Plexiglas circuit board assembly of the same dimensions as the previous experiment. All three dielectric liquids were tested. Additionally, the same enclosure, spacers, and data acquisition equipment were used again. Due to manufacturing tolerances, the enclosure widths this time were 8, 10, 16, 28, and 40 mm. Power levels were chosen to match those of the previous studies, but they were limited to the 125 °C maximum temperature of the temperature sensing elements (TSEs) inside each chip. Therefore, TSE temperatures were purposely limited to about 100 °C to meet this criterion in conjunction with the TSE's calibration range. For FC-43 and FC-75, power levels of 0.34, 0.57, 0.8, 1.3, and 1.48 W/chip were chosen. For the typically hotter FC-71, the power levels were 0.34, 0.57, 0.8, and 1.0 W/chip. Note that the 0.115 W power level was completely eliminated due to lessons learned from the previous experiments.

A total of 70 data runs were recorded and analyzed for the three dielectric liquids. The analysis of the dimensional results is similar to Matthews' and Aytar's work, but the non-dimensional analysis is somewhat different. After the FC-75 and FC-43 data runs were complete, it was realized that plots of Nu versus Grashof number, Gr, were linear, independent of Pr or power level. Therefore, CALC2 was altered to compute Gr in parallel with Ra. Average Pr was also calculated. The data, stored on floppy disk with the ACQ2 program, was then re-run with the modified CALC2 program. The extensive non-dimensional data thus obtained was used to produce a correlation for Nu. The correlation is of the following form:

$$Nu = aGr^{b1}X^{b2}$$

where Nu is based on the chip dimension in the direction of gravity. X is again the non-dimensional enclosure width, and a, b1, and b2 are constants.

B. DIMENSIONAL RESULTS

1. FC-71

As before, $T_{avg} - T_{sink}$ was plotted against Q_{net} for all spacings. As it can be seen in Figure 42, the data is linear for all spacings. The maximum chip temperature of 95.4 °C occurred in chip #6 at a power level of 1.0 W and a spacing of 8 mm. For comparison purposes, the chip temperature data was also row averaged like the previous experiments. However, recall that temperature data for the top and bottom rows of TSEs was based on a single chip each.

For all spacings and power levels, the order of average row temperatures were always top > middle > bottom. As in the previous study, this pattern indicated the dominance of the buoyant forces over the viscous forces in the fluid. The maximum temperature difference between the top and middle rows was 1.4 °C. For the middle and bottom rows, it was 5.4 °C. These differences were noted at a power level of 1.0 W and a spacing of 8 mm.

The thermocouples installed on the alumina substrate surface and the back of the Plexiglas circuit board assembly also indicated a temperature gradient. The upper two substrate temperatures were within 0.5 °C of each other; the same could be said for the lower two temperature readings. The upper pair of readings were always higher than the two substrate readings below. Similarly, when the nine backside temperatures were grouped by row, the magnitude order was always top > middle > bottom.

2. FC-43

Results for FC-43 were very similar to FC-71. Figure 43 shows the $T_{avg} - T_{sink}$ versus Q_{net} plots for all spacings. It is noted that the slope of the FC-43 data is smaller than its FC-71 counterpart. This

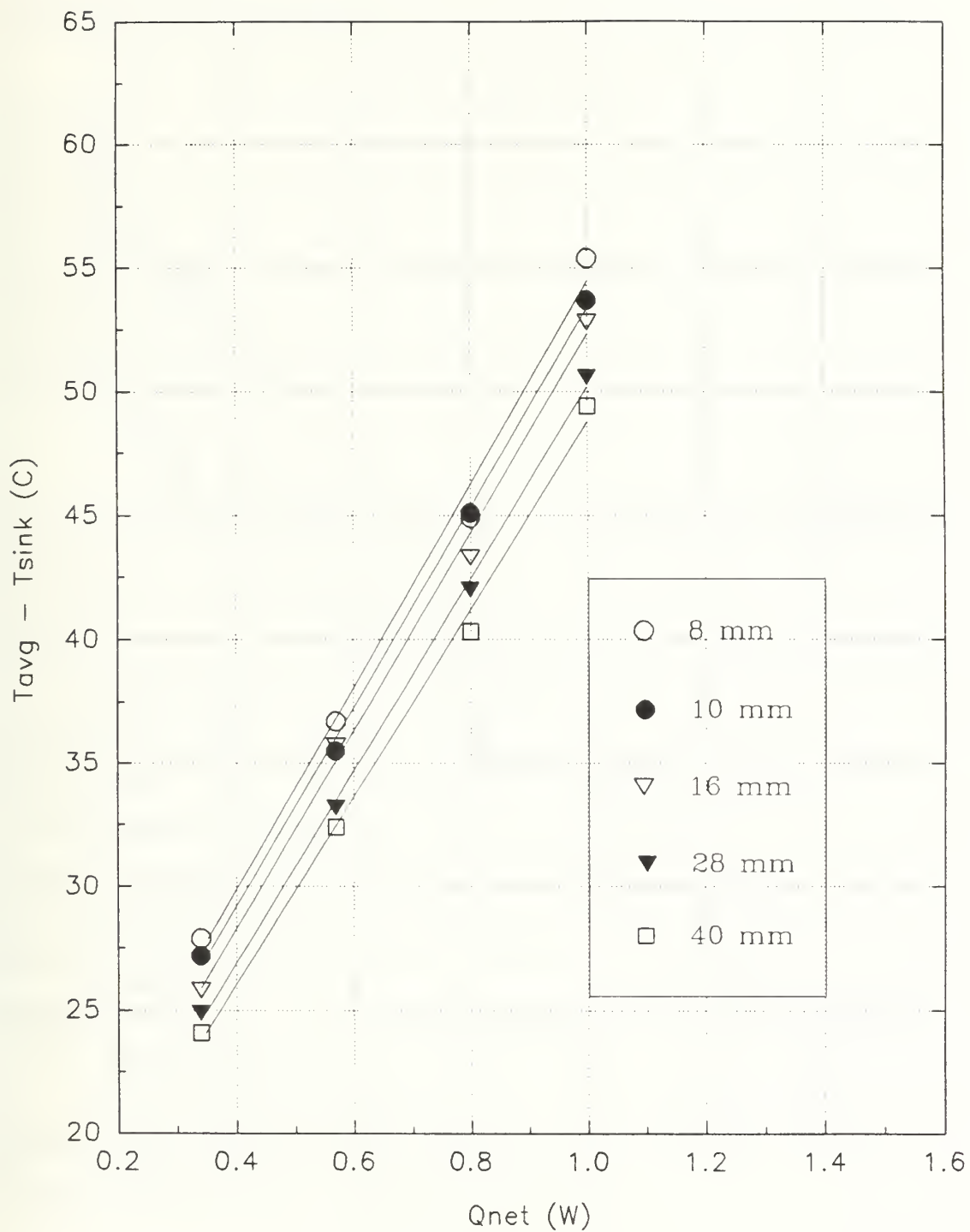


Figure 42. Array Average Temperature vs. Net Power for FC-71, NSWC Circuit Board

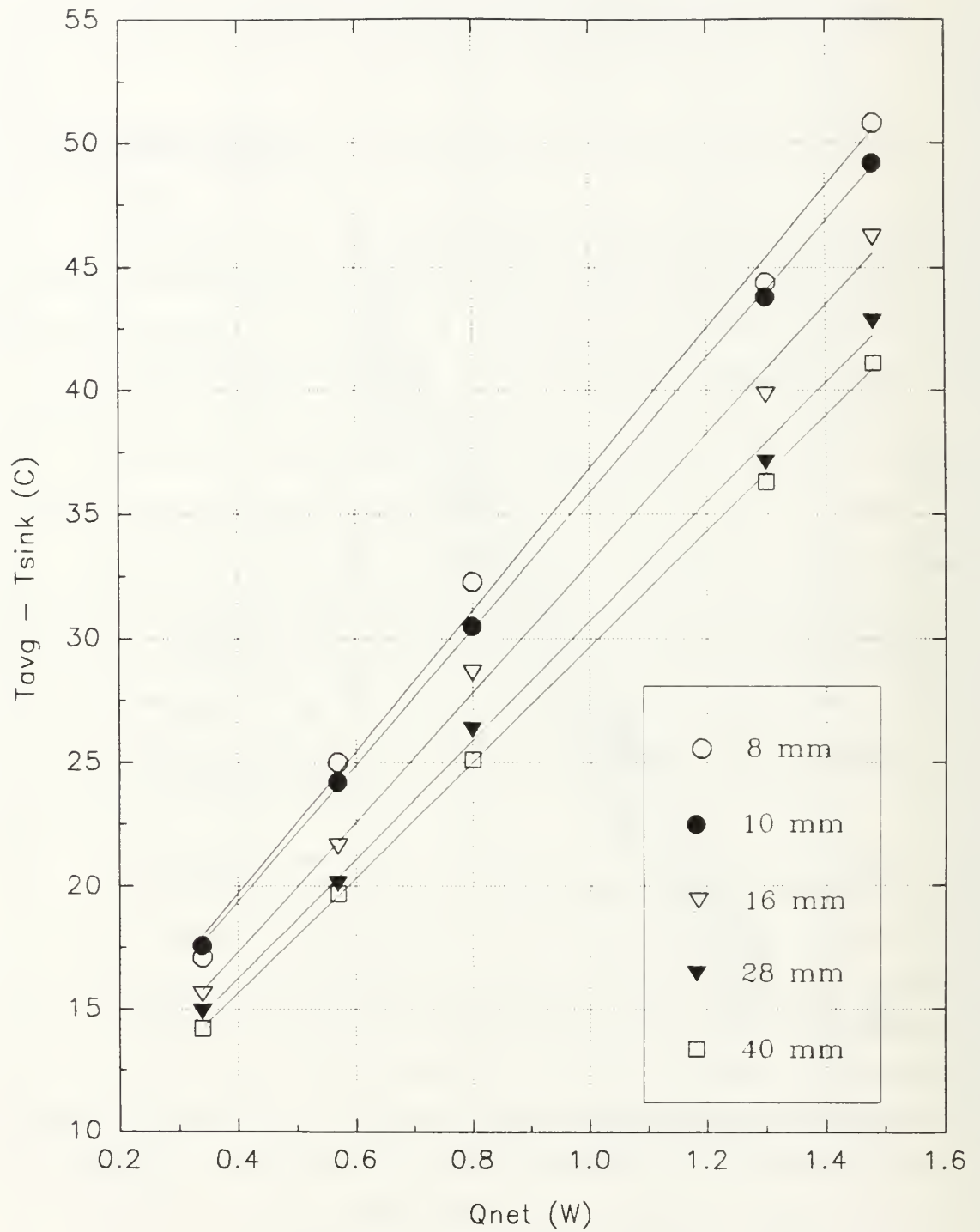


Figure 43. Array Average Temperature vs. Net Power for FC-43, NSWC Circuit Board

indicates that FC-43 is a more efficient heat transfer medium for the chips at a given power level.

The average row temperature order remained the same as the previous fluid. The maximum chip temperature of 101.9 °C was noted on chip #6 at a spacing of 8 mm and a power level of 1.48 W. Maximum temperature differences were 2.1 °C between the top and middle rows and 4.9 °C between the middle and bottom rows. As before, these differences occurred at the highest power level, 1.48 W, and minimum spacing, 8 mm. Finally, the remaining substrate and circuit board assembly temperatures exhibited the same trends as described for the FC-71.

3. FC-75

The third fluid, FC-75, exhibited almost identical results to FC-43. In Figure 44, the $T_{avg} - T_{sink}$ versus Q_{net} plots are again linear, and the slope of the FC-75 data is the smallest of the three liquids studied. The maximum chip temperatures achieved was only 92.3 °C. As expected, it occurred on chip #6 under maximum power and minimum spacing conditions.

As previously noted, the average row temperature data was again always top > middle > bottom. However, the maximum differences between rows was small. Only 0.4 °C separated the top and middle rows, and 3.3 °C was the difference between the middle and bottom rows. These row averaged temperature differences were noted at a spacing of 8 mm and a power level of 1.48 W. The established pattern of top > middle > bottom was again observed for row wise substrate surface and circuit board assembly temperatures.

4. FC-71, FC-43, and FC-75 as a Group

For comparison purposes, Figure 45 through Figure 49 are plots of $T_{avg} - T_{sink}$ versus Q_{net} for all three fluids. This means of presenting the data highlights the following conclusions:

- For a given power level, the dielectric fluid FC-75 convects heat away from the chips more efficiently than either FC-43 or FC-71.

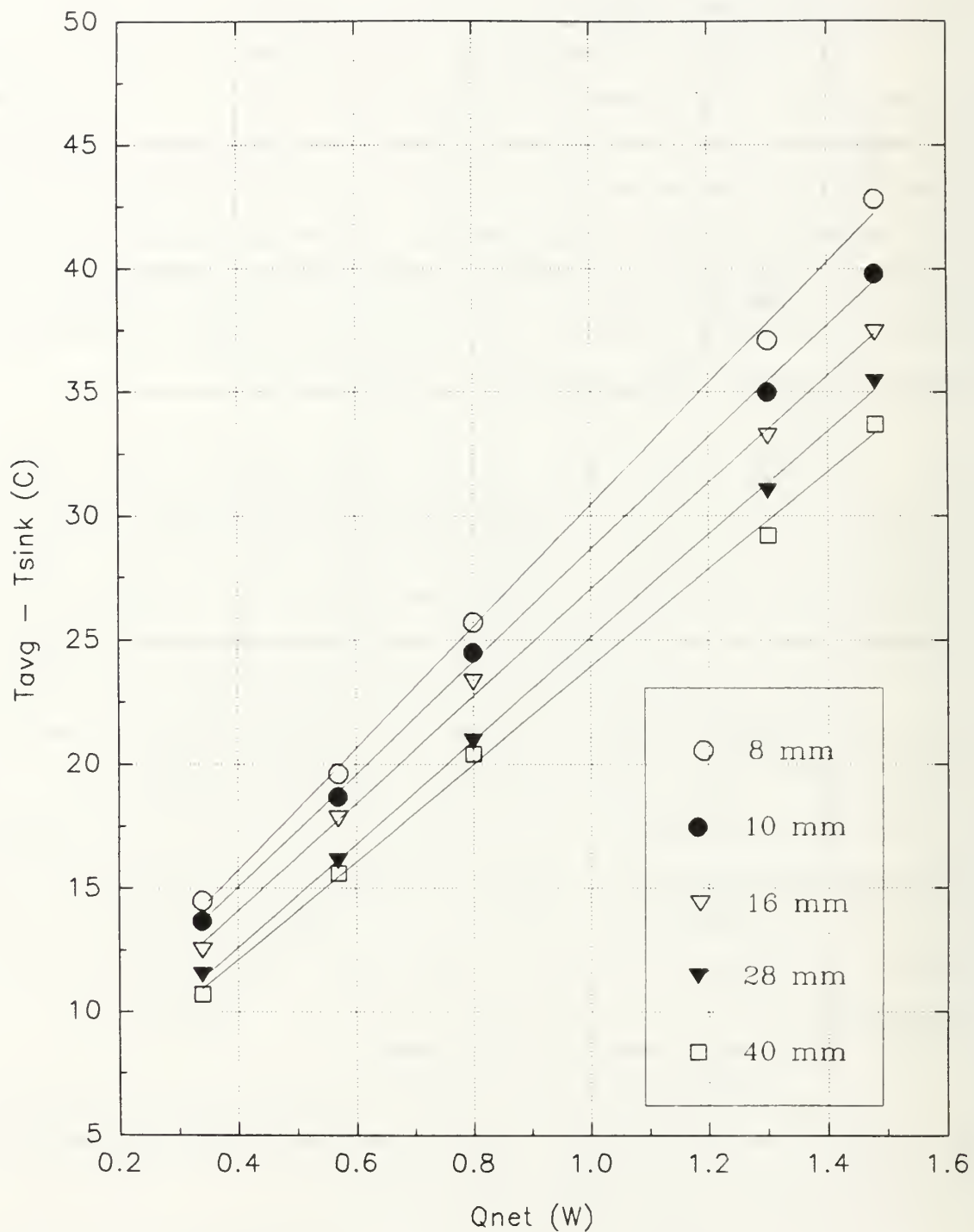


Figure 44. Array Average Temperature vs. Net Power for FC-75, NSWC Circuit Board

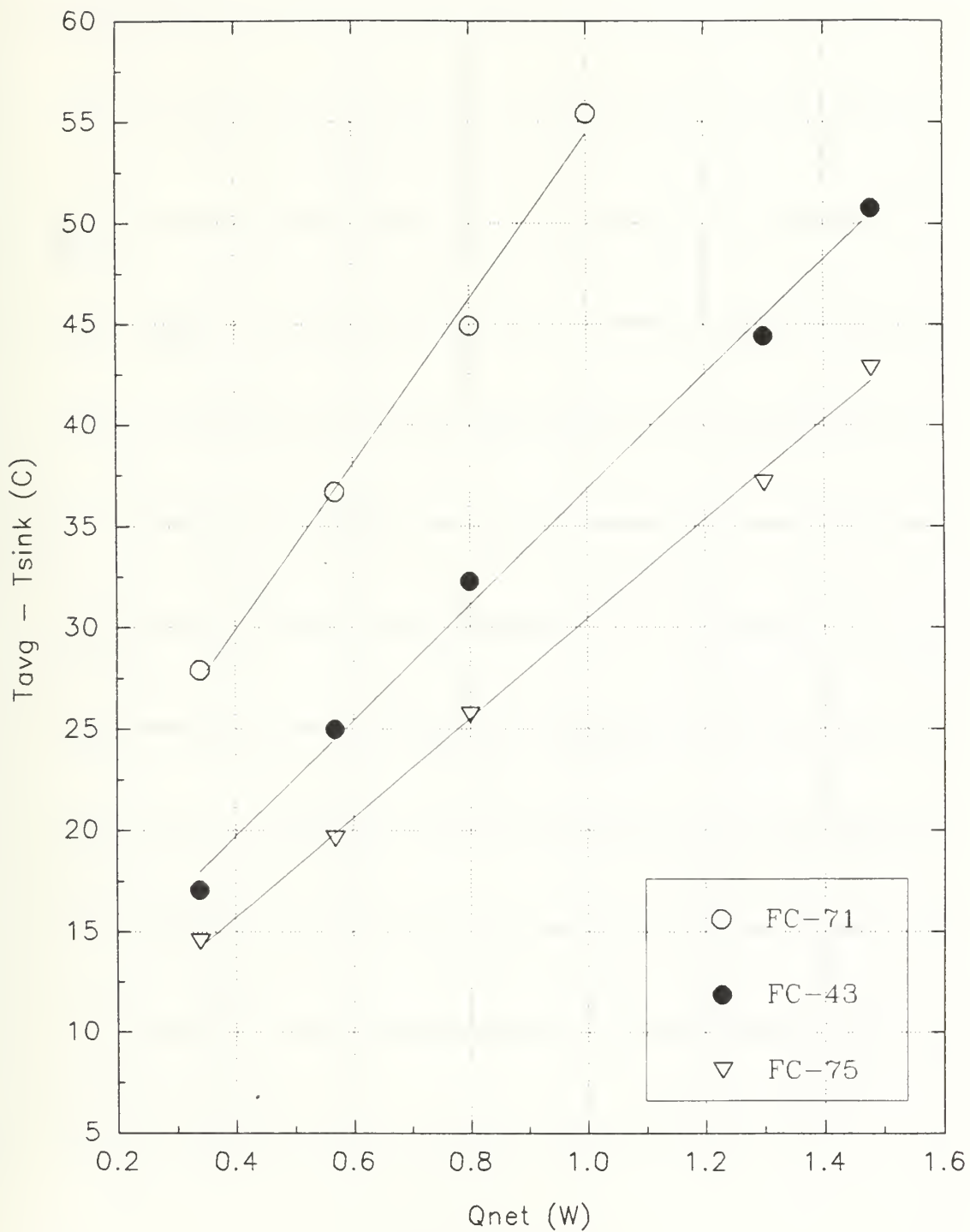


Figure 45. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 8 mm Spacing, NSWC Circuit Board

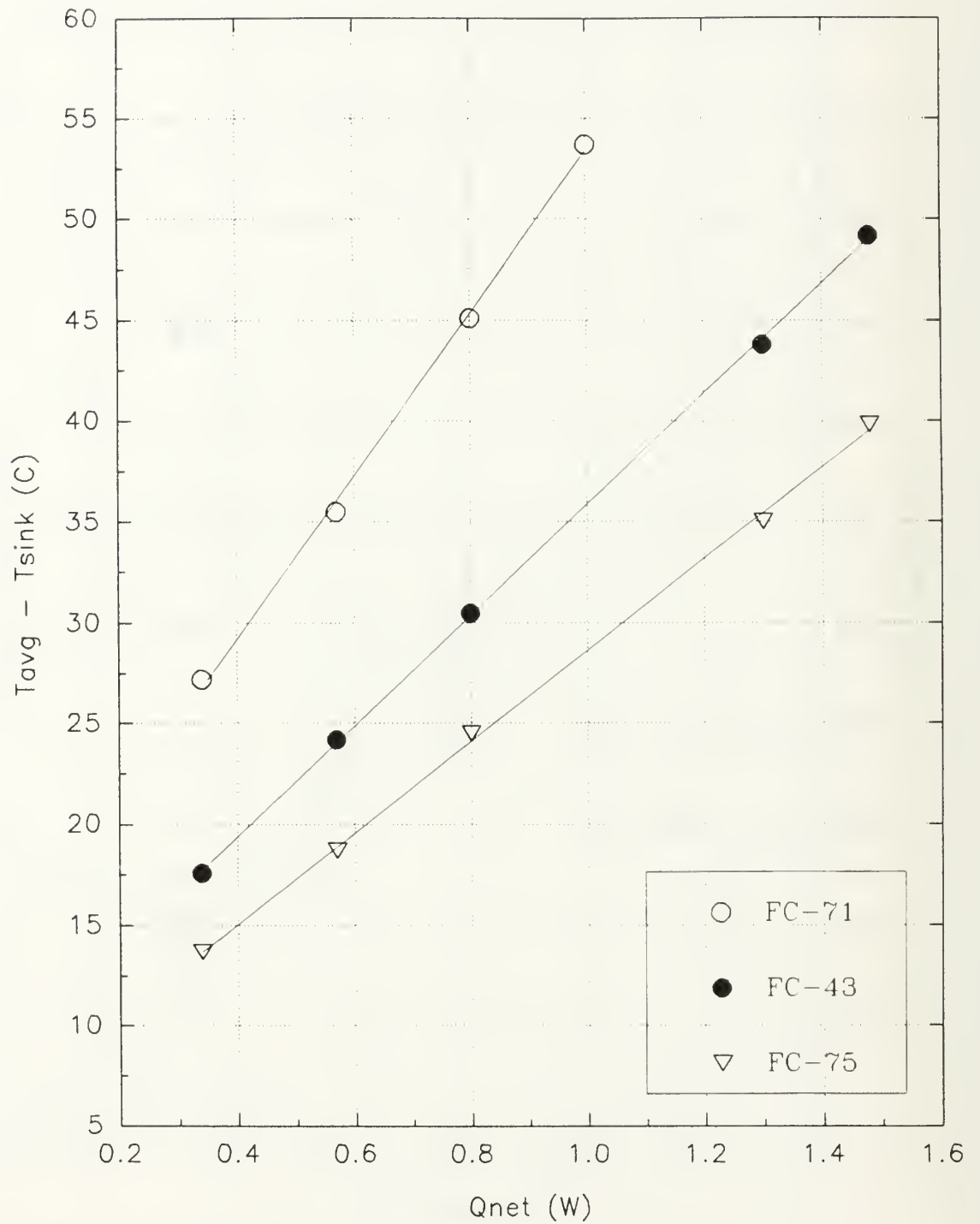


Figure 46. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 10 mm Spacing, NSWC Circuit Board

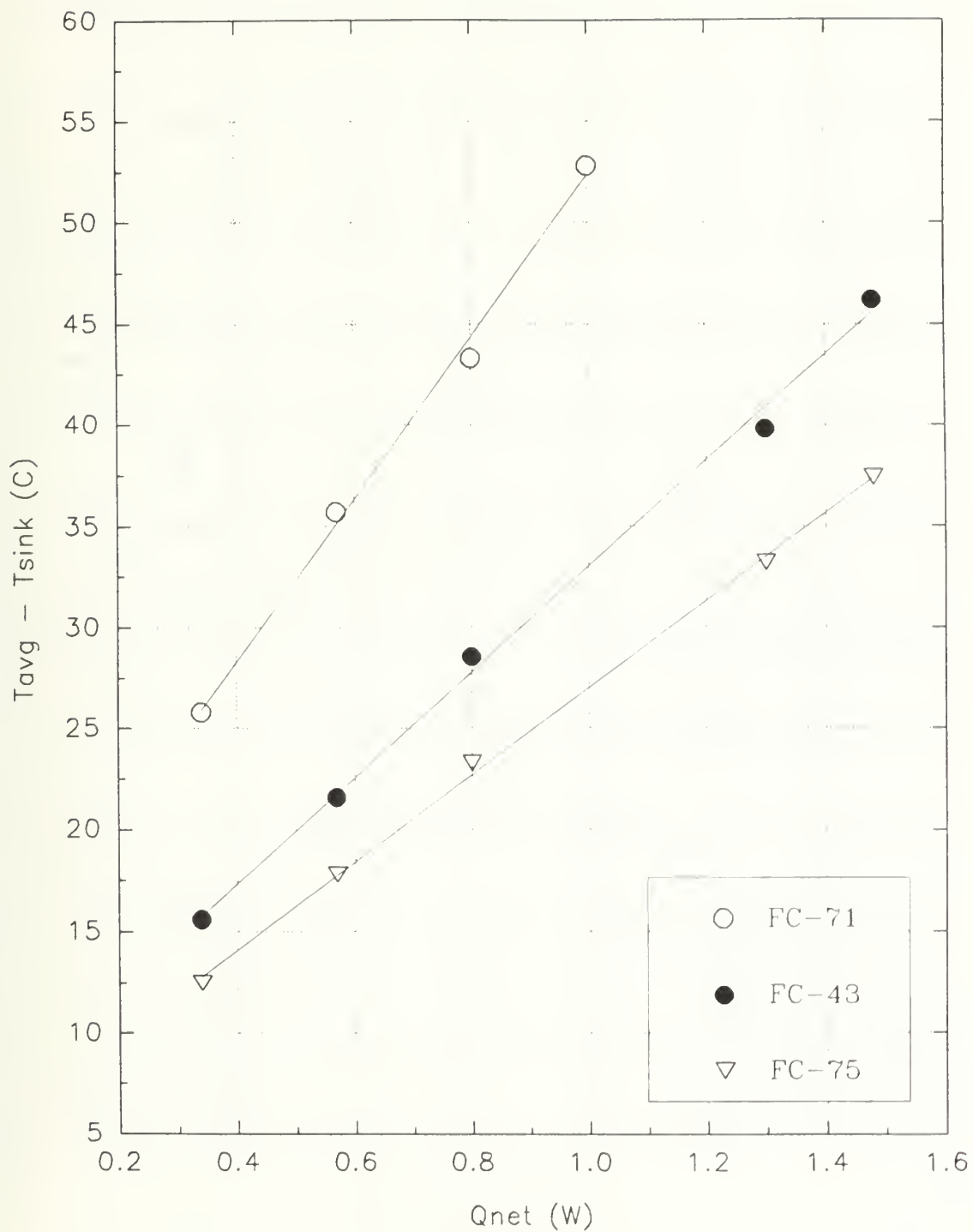


Figure 47. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 16 mm Spacing, NSWC Circuit Board

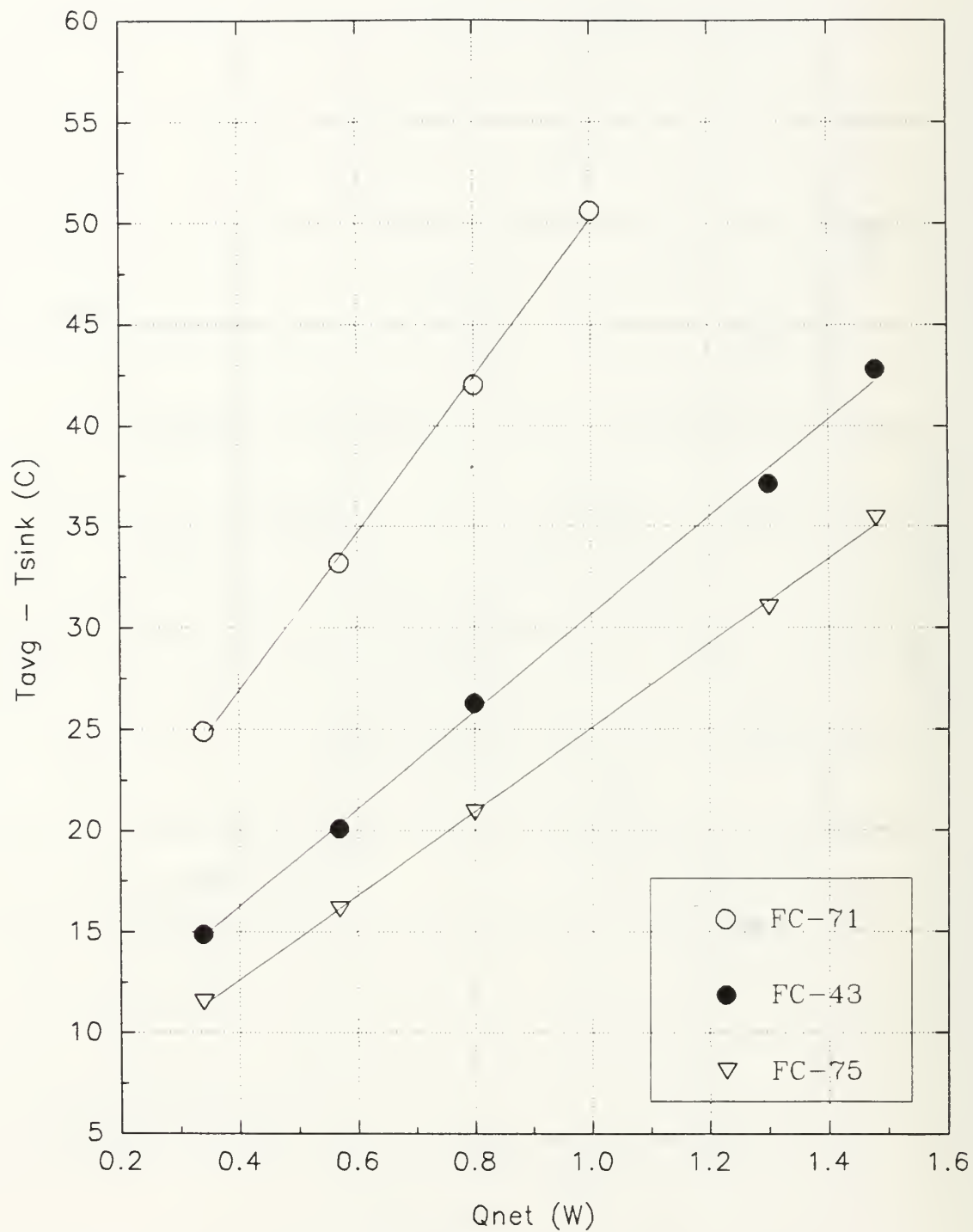


Figure 48. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 28 mm Spacing, NSWC Circuit Board

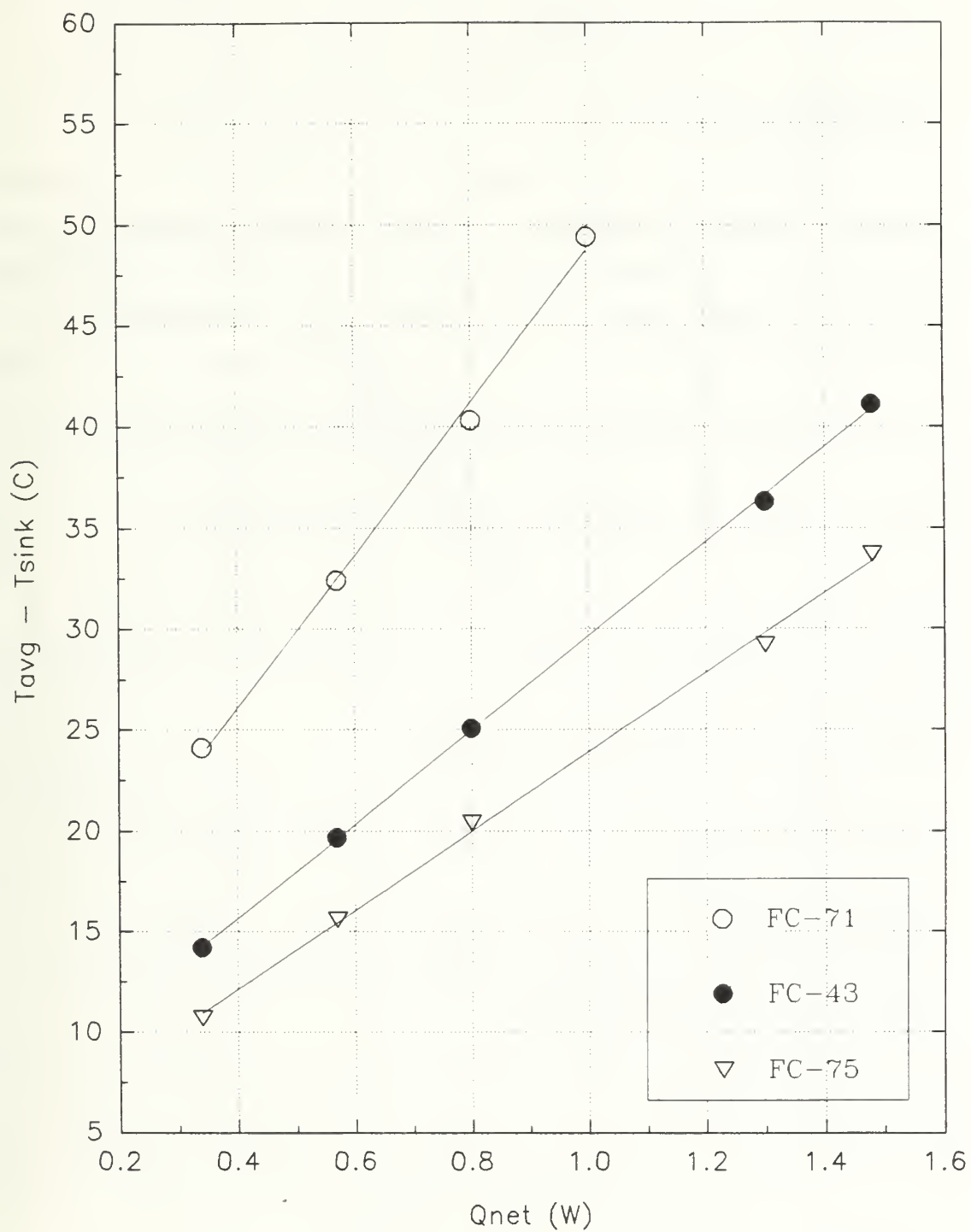


Figure 49. Array Average Temperature vs. Net Power for FC-71, 43, and 75, 40 mm Spacing, NSWC Circuit Board

- Operation with FC-71 leads to the highest chip temperatures. This would be a major disadvantage in the selection of the best dielectric liquid for immersion cooling.

C. NON-DIMENSIONAL RESULTS

1. General

The analysis of the non-dimensional data for the three dielectric liquids was grouped together. This was done because each liquid behaved similarly for a given power level or spacing. The maximum Nu and Gr were always noted at the highest power level for a given liquid, which was 1.0 W for the FC-71 and 1.48 W for the FC-43 or FC-75. Similarly, the minimum Nu and Gr always occurred at the minimum power level of 0.34 W for all liquids. Additionally, the maximum uncertainty in Nu and Gr was observed at this minimum power level and a minimum spacing of 8 mm.

Specifically, the following extremes were noted as described above:

- Maximum Nu at a spacing of 40 mm: 16.37 for FC-71, 31.04 for FC-43, and 41.44 for FC-75.
- Maximum Gr at a spacing of 8 mm: 3450 for FC-71, 2.478×10^5 for FC-43, and 1.224×10^6 for FC-75.
- Minimum Nu at a spacing of 8 mm: 9.81 for FC-71, 17.19 for FC-43, and 21.32 for FC-75.
- Minimum Gr at a spacing of 40 mm: 59 for FC-71, 1.030×10^4 for FC-43, and 1.112×10^5 for FC-75.
- Maximum uncertainties in Nu and Gr, respectively: 3.4% and 2.0% for FC-71, 4.2% and 3.3% for FC-43, and 4.7% and 3.8% for FC-75.

2. Effect of Grashof Number

As previously stated, the final goal of the non-dimensional analysis was to produce an empirical correlation for Nu. Figure 50 through Figure 54 are plots of Nu versus Gr for each individual spacing. When all three dielectric liquids are plotted together as shown, a linear relationship is realized.

This relationship was assumed to be of the form:

$$Nu = c_1 Gr^{b_1}$$

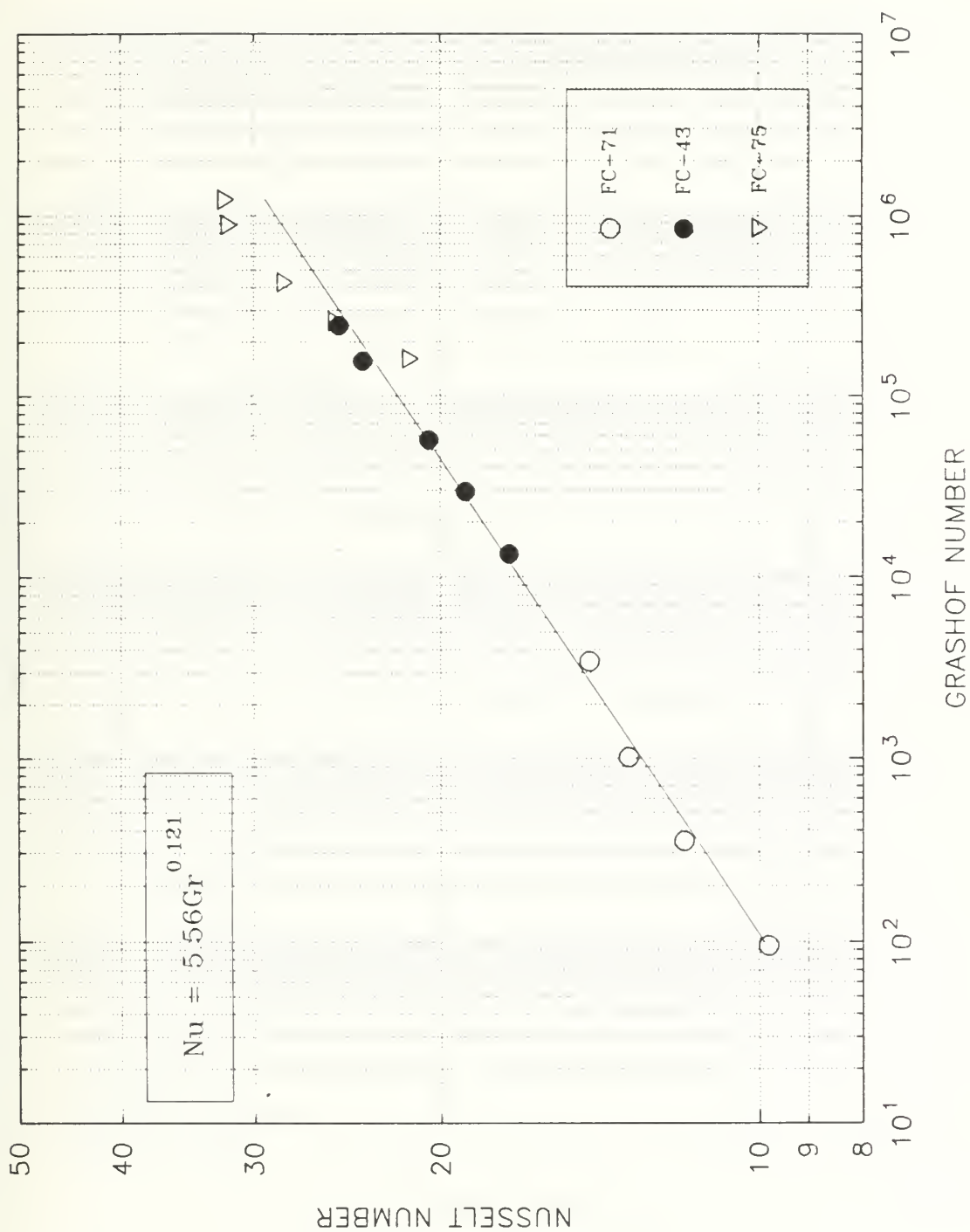
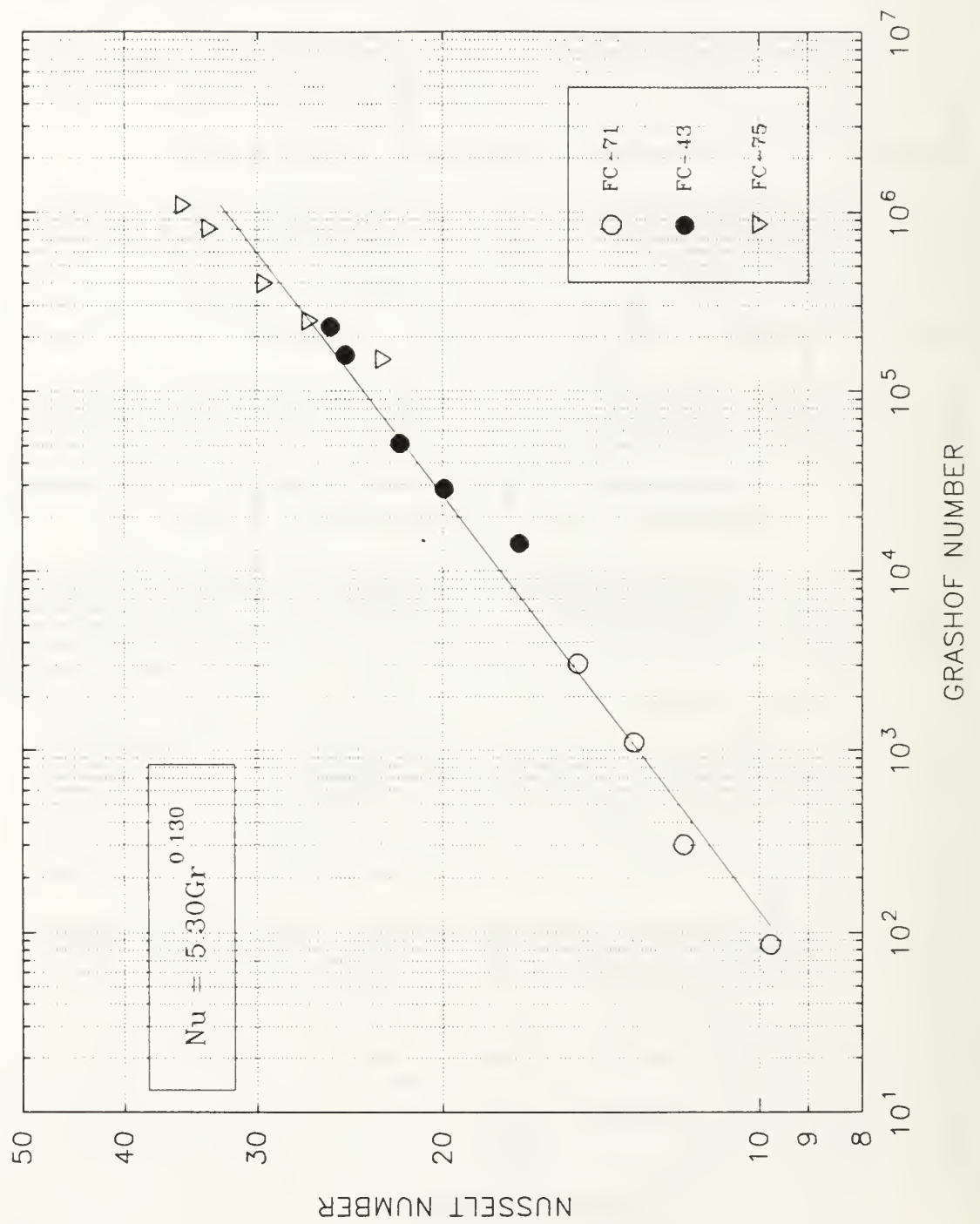


Figure 50. Nu vs. Gr for FC-71, 43, and 75, Array Averaged, NSWC Circuit Board, and 8 mm Enclosure Width



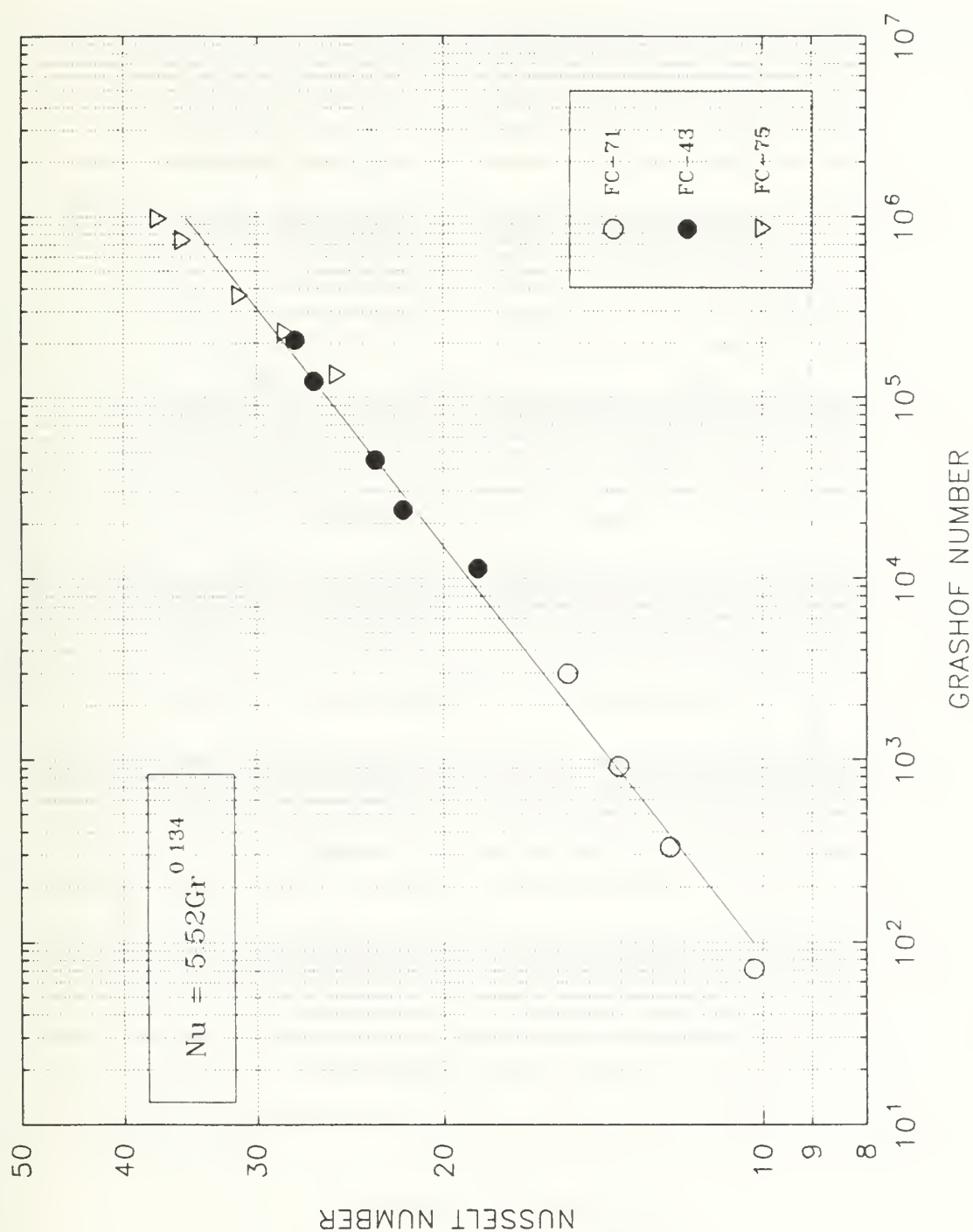


Figure 52. Nu vs. Gr for FC-71, 43, and 75, Array Averaged, NSWC Circuit Board, and 16 mm Enclosure Width

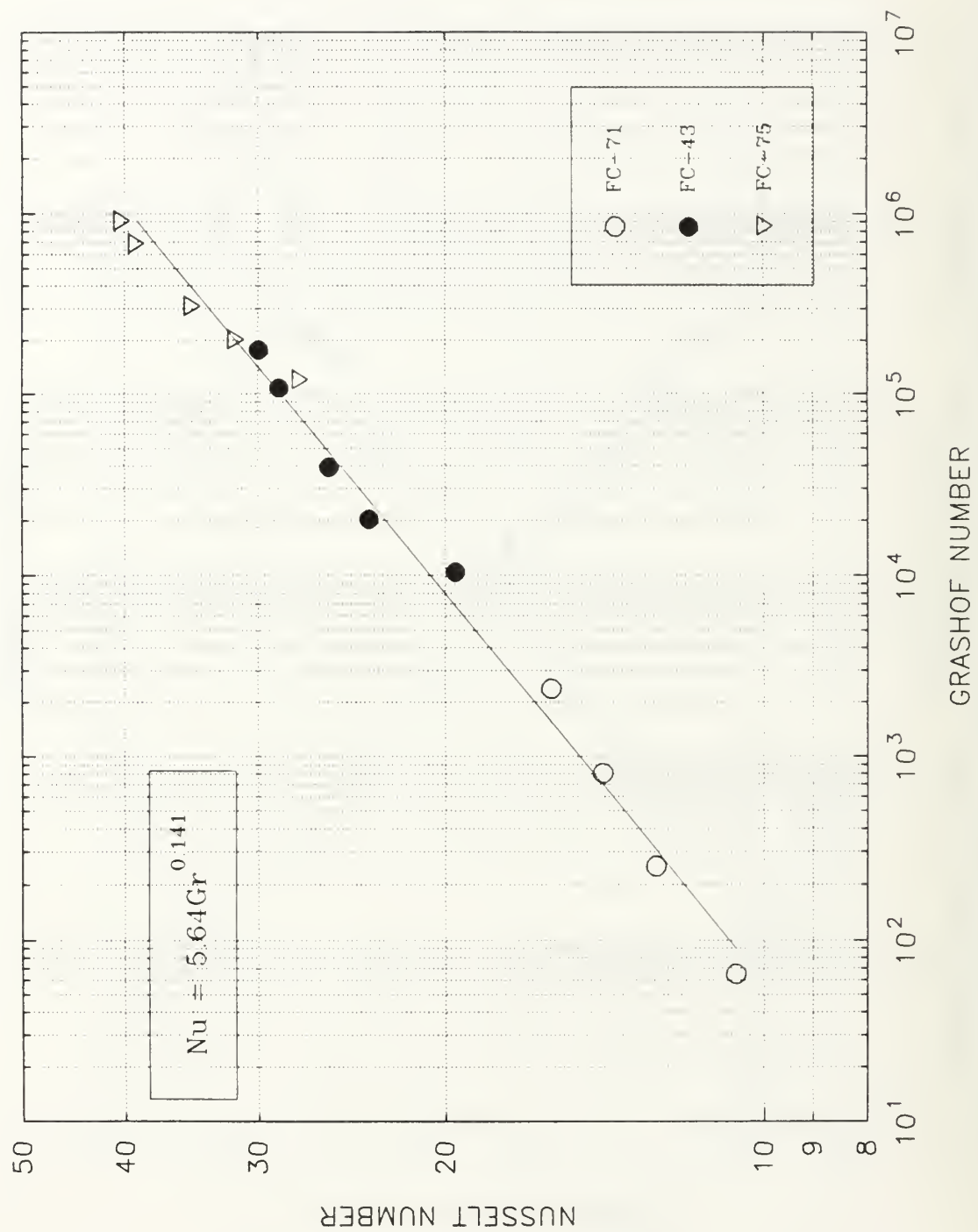


Figure 53. Nu vs. Gr for FC-71, 43, and 75, Array Averaged, NSWC Circuit Board, and 28 mm Enclosure Width

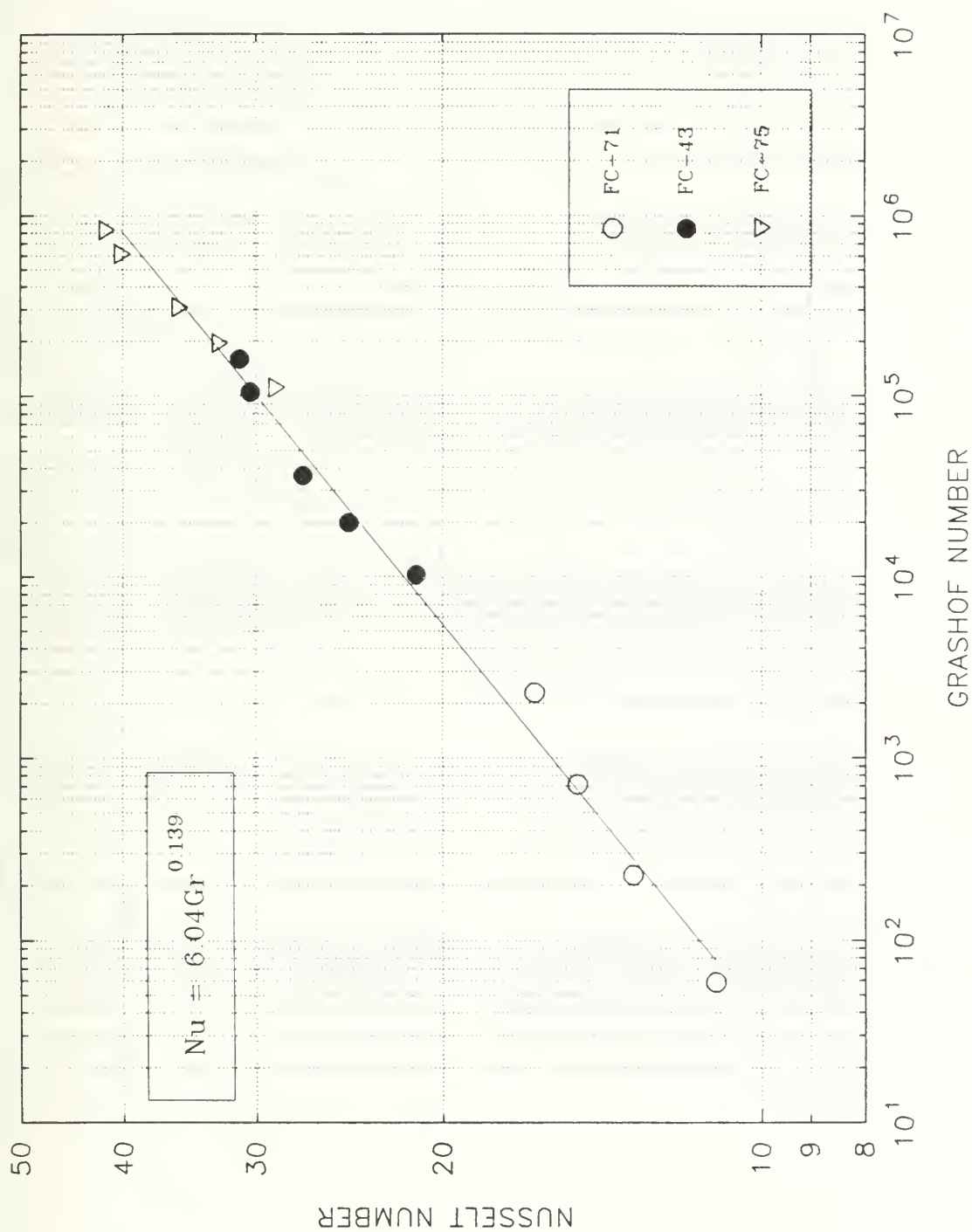


Figure 54. Nu vs. Gr for FC-71, 43, and 75, Array Averaged, NSWC Circuit Board, and 40 mm Enclosure Width

where c_1 and b_1 are constants. Again, TABLECURVE was used to find the constants in the above equation. The constant c_1 varied from 5.30 to 6.04, and the average was equal to 5.61. The constant b_1 was found to vary from 0.121 to 0.141, with the average being equal to 0.133.

3. Effect of Enclosure Width

Enclosure width effects were accounted for by the following equation:

$$Nu = c_2 X^{b_2}$$

where b_2 is a constant and c_2 is the Gr dependence. X is again the non-dimensional enclosure width. Similar to the previous study, the constant c_2 can be represented by:

$$c_2 = c_1 Gr^{b_1}$$

The constant b_2 was derived from a plot of Nu versus X for the five spacings, which is shown in Figure 55. The values for Nu were derived

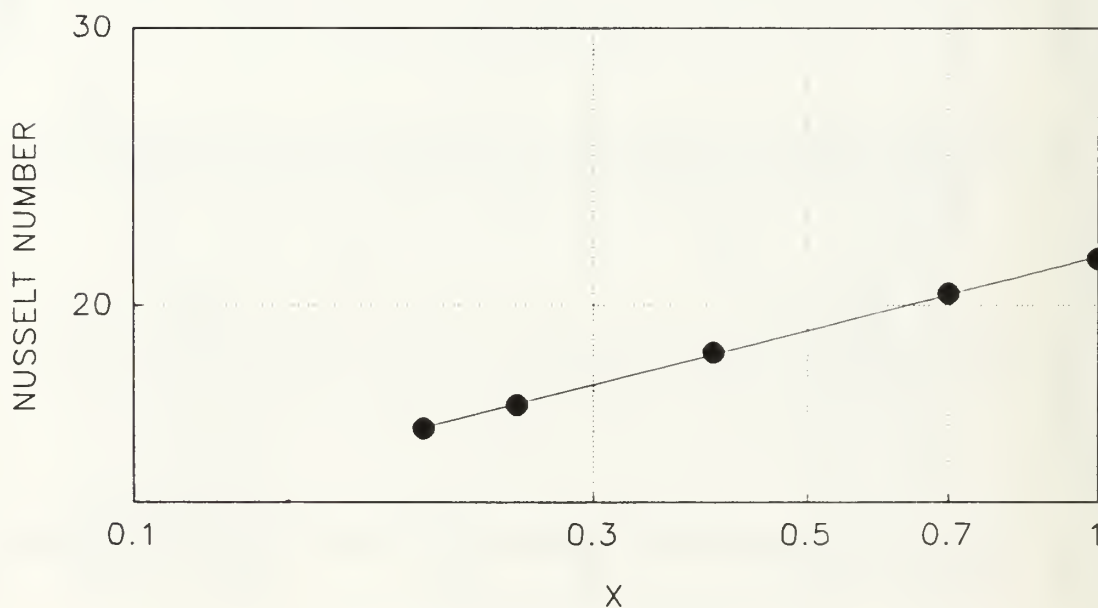


Figure 55. Nu vs. X for FC-71, 43, and 75, Array Averaged, NSW Circuit Board, and all Enclosure Widths

from the curve fit equations for each spacing's data using an average Gr of 9×10^3 . The resulting value for the exponent b2 was 0.154.

Combining the above results gives the following general correlation for the three dielectric liquids:

$$Nu = 5.61Gr^{0.133}X^{0.154}$$

It is valid over the ranges:

$$1 \times 10^2 < Gr < 8 \times 10^6$$

$$0.20 < X < 1.0$$

Five representative Grashof numbers per spacing were selected to determine values for Nu from the respective curve fit equations. These results were then compared to the Nusselt number produced from the correlation. The average difference in Nu for the 25 data points was 12.3%. Similar to the previous study, a trial and error approach was used to improve the accuracy of the general correlation. When the value for c_1 was changed from 5.61 to 6.40, the agreement improved to less than 2%. Therefore, a refined correlation for the three dielectric liquids is:

$$Nu = 6.40Gr^{0.133}X^{0.154}$$

It is valid over the ranges:

$$1 \times 10^2 < Gr < 8 \times 10^6$$

$$0.20 < X < 1.0$$

VII. CONCLUSIONS

Two studies of the natural convection heat transfer of heated protrusions immersed in dielectric liquids were conducted. The first study used a three by three array of computer chip sized aluminum blocks immersed in FC-71. The other study used a three by three array of 8.9 mm square thermal evaluation devices. Three fluids, FC-71, FC-43, and FC-75 were evaluated. Both studies used an insulated Plexiglas enclosure with a top mounted heat exchanger. Spacers were used to vary the enclosure width, and the maximum spacing was 40 mm. Conclusions from the two studies are as follows:

1. For the first study, an empirical correlation for Nusselt number was developed. It took into account variations in Rayleigh number and non-dimensional enclosure width, X . The correlation was based on array averaged data. It is listed below:

$$Nu = 0.435Ra^{0.24}X^{0.165}$$

$$9 \times 10^5 < Ra < 2 \times 10^7$$

$$0.175 < X < 1.0$$

The maximum uncertainty in the Nusselt number was 7.4%, and the correlation was accurate to within 4% of the array averaged data.

2. When the FC-71 data was combined with Matthews' FC-43 and FC-75 data, several generalizations could be made. For array averaged plots of $T_{avg} - T_{sink}$ versus Q_{net} , the order of temperatures for the three fluids was FC-71 > FC-43 > FC-75. Additionally, the temperatures increased as power level increased or enclosure width was decreased. For log-log plots of Nu versus Ra , each liquid exhibited a linear relationship. The FC-71 data had the highest Nu and Ra numbers for each spacing.

3. For the electronic chip study, a general correlation was developed for Nusselt number from the combined data of the three dielectric fluids. This correlation took into account variations in Grashof number and non-dimensional enclosure width, X . As before, it was based on array averaged data. The correlation is as follows:

$$Nu = 6.40Gr^{0.133}X^{0.154}$$

$$1 \times 10^2 < Gr < 8 \times 10^6$$

$$0.20 < X < 1.0$$

The maximum uncertainty in the Nusselt number was 4.7%, and the correlation was accurate to within 2% of the array averaged data.

4. Plots of $T_{avg} - T_{sink}$ versus Q_{net} for the three liquids again showed that the order of temperatures was FC-71 > FC-43 > FC-75 for each spacing. For all three liquids, the order of the averaged temperature data was always top > middle > bottom. This same order was also exhibited by the row averaged Plexiglas substrate back and alumina substrate surface temperatures.

5. Overall, the best liquid for natural convection heat transfer was FC-75. The best liquid is defined as the one which produced the lowest component temperatures for a given power level or spacing. Lower chip temperatures equate to longer chip lives.

VIII. RECOMMENDATIONS

The following recommendations are made for further research:

1. Manufacture additional NSWC circuit boards that have every chip wired individually. Use these boards to assemble a large array equivalent in size to a typical mainframe computer circuit board.
2. Test the above array in a suitably sized enclosure filled with FC-75. Produce an empirical correlation for the Nusselt number.
3. In parallel with the experimental work, produce computer models which can also be used to predict heat transfer characteristics. Compare these results with the experimental results and modify the programs as necessary.

APPENDIX A. COMPUTER PROGRAM ACQUIRE

```

10  | FILE ACQUIRE
20  |
30  | EDITED BY BY LOOF R. THOMPSON
40  | 1/12/82. FROM ORIGINALS OF FAMUK,
50  | BENEDICT, TORRES, AYTAH AND MATTHEWS.
60  |
70  | READ FILE "READ_ME"
80  |
90  COM /C:/ D(7)
100 DIM Emf(76),Power(5),T(76),Rp(8)
110 |
120 | CORRELATION FACTORS TO CONVERT EMF TO DEGREES CELSIUS. SOURCE:
130 | IHP APPLICATION NOTE 290, P. 8, NBS POLYNOMIAL COEFFICIENTS FOR
140 | TYPE T (COPPER-CONSTANTAN) THERMOCOUPLES.
150 DATA 0.10086091,25727.9,-767345.6,78025596,
160 DATA -9247486595,6.98E11,-2.66E13,3.94E14
170 | RESISTANCES SERIES TO HEATERS
180 DATA 2.0,2.0,2.0,2.0,2.0,2.0,2.0,2.0
190 |
200 READ D(+)
210 READ Rp(+)
220 PRINTER IS 701
230 BEEP
240 |
250 INPUT "ENTER THE INPUT MODE: 0=SYS, 1=FILE",Im
260 |
270 IF Im=1 THEN
280 BEEP
290 INPUT "ENTER THE NAME OF THE FILE TO BE READ",Oldfile$
300 |
310 PRINT USING "15X","THESE RESULTS ARE STORED IN FILE : ",10A";Oldfile$
320 ELSE
330 BEEP
340 INPUT "ENTER THE NAME OF THE NEW FILE",Newfile$
350 PRINT USING "10X","THESE RESULTS ARE STORED IN FILE: ",10A";Newfile$
360 END IF
370 PRINT
380 |
390 INPUT "FLOW VIZ? Y/N",Ans$
400 |
410 INPUT "ENTER THE BATH TEMP",B$
420 PRINT USING "15X","BATH TEMP WAS: ",10A";B$
430 |
440 IF Ans$="Y" THEN PRINT USING "15X","THIS RUN WAS RECORDED WITH FLOW VIZ",
10A"
450 |
460 INPUT "ENTER THE WALL SPACING",Wall$
470 PRINT USING "15X","SPACING WAS: ",10A";Wall$
480 |
490 INPUT "ENTER THE TYPE OF LIQUID USED",Liquid$

```



```

500 PRINT USING "15X,";"THE FLUORINERT USED WAS: ",10A";Liquid$
510 IF Im=1 THEN ASSIGN @File TO Oldfiles$
520 !
530 IF Im=0 THEN
540 CREATE BDAT Newfiles$,5
550 ASSIGN @File TO Newfiles$
560 END IF
570 !
580 ! READ DATA
590 !
600 IF Im=0 THEN
610 OUTPUT 709;"AF AF00 AL79"
620 OUTPUT 722;"F1 R1 T1 Z0 FLO"
630 !
640 FOR I=0 TO 78
650 OUTPUT 709;"AS"
660 WAIT 1
670 ENTER 722;Emf(I)
680 IF Emf(I) < 0. THEN
690 Emf(I) = -Emf(I)
700 END IF
710 BEEP
720 NEXT I
730 !
740 ! CORRECTION FOR OFFSET IN HP 3497A DAS ICE
750 ! POINT REFERENCE
760 FOR I=0 TO 19
770 Emf(I) = Emf(I) - 5.5E-6
780 NEXT I
790 FOR I=20 TO 39
800 Emf(I) = Emf(I) - 1.05E-5
810 NEXT I
820 FOR I=40 TO 59
830 Emf(I) = Emf(I) - 5.0E-6
840 NEXT I
850 FOR I=60 TO 76
860 Emf(I) = Emf(I) - 2.55E-5
870 NEXT I
880 !
890 OUTPUT @File;Emf(*)
900 !
910 ELSE
920 ENTER @File;Emf(*)
930 END IF
940 !
950 OUTPUT 709;"TD"
960 !
970 FOR I=0 TO 60
980 Sum=0.
990 FOR J=0 TO 7
1000 Sum=Sum+D(J)*Emf(I)^J
1010 NEXT J
1020 T(I)=Sum
1030 NEXT I
1040 !
1050 FOR I=71 TO 76
1060 Sum=0

```

```

1070 FOR J=0 TO 7
1080 Sum=Sum+D(J)+Emf(I)/J
1090 NEXT J
1100 T(I)=Sum
1110 NEXT I
1120 I
1130 PRINT USING "15X","VOLTMETER READING WAS: ",D.DDDD;Emf(61)
1140 PRINT USING "15X","AMBIENT TEMP WAS: ",D.D.D;T(76)
1150 PRINT
1160 I POWER CALCULATIONS
1170 I
1180 J=1
1190 Volt=Emf(61)
1200 I
1210 FOR I=62 TO 70
1220 Power(J)=Emf(I)*(Volt-Emf(I))/Rp(I-62)
1230 J=J+1
1240 NEXT I
1250 I
1260 BEEP
1270 BEEP
1280 I
1290 PRINT USING "10X","ALL TEMPERATURES ARE IN DEGREES CELSIUS""
1300 I
1310 PRINT
1320 I
1330 PRINT USING "12X","CENTER TOP RIGHT LEFT BOTTOM
BACK""
1340 PRINT
1350 PRINT USING "1X","CHIP NO1: ",6(DDD.DD,5X);T(0),T(1),T(2),T(3),T(4),T(5)
1360 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(1)
1370 PRINT
1380 PRINT USING "1X","CHIP NO2: ",6(DDD.DD,5X);T(6),T(7),T(8),T(9),T(10),T(11)
1390 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(2)
1400 PRINT
1410 PRINT USING "1X","CHIP NO3: ",6(DDD.DD,5X);T(12),T(13),T(14),T(15),T(16),T(17)
1420 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(3)
1430 PRINT
1440 PRINT USING "1X","CHIP NO4: ",6(DDD.DD,5X);T(18),T(19),T(20),T(21),T(22),T(23)
1450 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(4)
1460 PRINT
1470 PRINT USING "1X","CHIP NO5: ",6(DDD.DD,5X);T(24),T(25),T(26),T(27),T(28),T(29)
1480 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(5)
1490 PRINT
1500 PRINT USING "1X","CHIP NO6: ",6(DDD.DD,5X);T(30),T(31),T(32),T(33),T(34),T(35)
1510 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(6)
1520 PRINT
1530 PRINT USING "1X","CHIP NO7: ",6(DDD.DD,5X);T(36),T(37),T(38),T(39),T(40),T(41)
1540 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(7)
1550 PRINT

```

```

1560 PRINT USING "1X","CHIP NO8: ",6(DDD.DD,5X);T(42),T(43),T(44),T(45),T(46)
,T(47)
1570 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(8)
1580 PRINT
1590 PRINT USING "1X","CHIP NO9: ",6(DDD.DD,5X);T(48),T(49),T(50),T(51),T(52)
,T(53)
1600 PRINT USING "5X","POWER (WATTS): ",D.DDD;Power(9)
1610 !
1620 PRINT
1630 PRINT
1640 !
1650 PRINT USING "5X","HEAT EXCHANGERS TEMPERATURES:      RIGHT      CENTER      LE
FT""
1660 PRINT USING "10X","BOTTOM IS INSULATED""
1670 PRINT USING "10X","TOP: ",24X,3(DD.DD,5X);T(54),T(55),T(56)
1680 PRINT
1690 PRINT
1700 !
1710 PRINT USING "5X","BACK PLANE TEMPERATURES ARE :""
1720 PRINT
1730 PRINT USING "10X","T(57): ",2X,DD.DD;T(57)
1740 PRINT USING "10X","T(58): ",2X,DD.DD;T(58)
1750 PRINT USING "10X","T(59): ",2X,DD.DD;T(59)
1760 PRINT USING "10X","T(60): ",2X,DD.DD;T(60)
1770 PRINT USING "10X","T(71): ",2X,DD.DD;T(71)
1780 PRINT USING "10X","T(72): ",2X,DD.DD;T(72)
1790 PRINT USING "10X","T(73): ",2X,DD.DD;T(73)
1800 PRINT USING "10X","T(74): ",2X,DD.DD;T(74)
1810 PRINT USING "10X","T(75): ",2X,DD.DD;T(75)
1820 BEEP
1830 PRINTER IS 1
1840 !
1850 ASSIGN @File TO *
1860 END

```

APPENDIX B. COMPUTER PROGRAM CALCDIEL

```

10  !*****
20  ! PROGRAM CalcDiel  *
30  !*****
40  !
50  ! MODIFIED BY LCDR R. THOMPSON 1/13 AND 4/15/92.
60  ! FROM ORIGINALS OF PAMUK, BENEDICT, TORRES,
70  ! AYTAZ AND MATTHEWS.
80  !
90  !*****
100 ! THIS PROGRAM ANALYZES THE DATA READ FROM *
110 ! A DATA FILE DESIGNATED BY THE OPERATOR.IT*
120 ! REDUCES THE DATA TO CALCULATIONS OF NET *
130 ! POWER, RAYLEIGH AND NUSSELT NUMBERS. THE*
140 ! UNCERTAINTY ANALYSIS IS ALSO INCLUDED.  *
150 !*****
160 !
170 ! VARIABLES USED ARE:
180 ! EMF   : VOLTAGE FROM THE THERMOCOUPLES.
190 ! POWER : POWER DISSIPATED BY THE HEATERS
200 ! T(I)  : TEMPERATURE CONVERTED FROM THERMO-
210 !        COUPLE VOLTAGE
220 ! Tavg  : IS THE AVERAGE TEMPERATURE OF THE
230 !        CHIP. IT IS OBTAINED MULTIPLYING
240 !        THE TEMPERATURE FOUND IN EACH FACE
250 !        BY THE AREA AND DIVIDING BY THE TO-
260 !        TAL AREA
270 ! Ts    : CHIP BACK SURFACE TEMPERATURE
280 ! Tfilm : FILM TEMPERATURE OF THE DIELECTRIC
290 ! Qnet  : ELECTRIC POWER MINUS CONDUCTION LOSSES
300 ! Tsink : AVERAGE OF THE 3 THERMOCOUPLES IN
310 !        THE UPPER HEAT EXCHANGER
320 ! Nu1   : LENGTH BASED NUSSELT NUMBER
330 ! Nu2   : AREA-PERIMETER BASED NUSSELT NUMBER
340 ! D...  : UNCERTAINTY OF A VARIABLE (EXCEPT
350 !        Dliq and Delt)
360 ! OTHER VARIABLES ARE SELF-EXPLANATORY
370 !*****
380 !
390 ! COM /Co/ D(7)
400 !
410 ! DIM Emf(76),Power(9),T(76),Tavg(9),Ts(9)
420 ! DIM Tfilm(9),Qnet(9),H(9),K(9),Rho(9),Cp(9)
430 ! DIM N(9),Nu1(9),Ra1(9),Delt(9),Alfa(9),Pr(9)
440 ! DIM Gr1(9),Beta(9),Dpower(9),Ra2(9)
450 ! DIM Gr2(9),Raf1(9),Raf2(9),Nu2(9)
460 ! DIM Rowra1(3),Rownu1(3)
470 !
480 ! CORRELATION FACTORS TO CONVERT Emf TO DEGREES CELSIUS. SOURCE:
490 ! HP APPLICATION NOTE 290, P. 8, NBS POLYNOMIAL COEFFICIENTS FOR
500 ! TYPE T (COPPER-CONSTANTAN) THERMOCOUPLES.
510 ! DATA 0.10086091,25727.9,-767345.8,78025596,

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```

520 DATA -9247486589,6.98E11,-2.66E13,3.94E14
530 !
540 READ D(*)
550 ! PRECISION RESISTOR VALUE IN OHMS
560 Rp=2.0
570 !
580 PRINTER IS 701
590 BEEP
600 BEEP
610 !
620 INPUT "ENTER THE NAME OF THE FILE CONTAINING DATA",Oldfile$
630 !
640 PRINT USING "10X","THE RAW Emf DATA ARE FROM THE FILE:   ","10A";Oldfile$
650 !
660 INPUT "ENTER THE POWER SETTING ",Power$
670 PRINT USING "9X"," THE POWER SETTING PER CHIP WAS:   ","10A";Power$
680 !
690 INPUT "ENTER THE TYPE OF LIQUID USED",Liquid$
700 PRINT USING "10X","THE FLUORINERT USED WAS:   ","10A";Liquid$
710 INPUT "ENTER THE TYPE OF DIELECTRIC:0=FC-75,1=FC-43,2=FC-71",Dliq
720 !
730 INPUT "ENTER THE WALL SPACING",Wall$
740 PRINT USING "10X","THE DISTANCE TO THE FRONT WALL WAS:   ","10A";Wall$
750 !
760 INPUT "ENTER THE GEOMETRY TYPE: 0=HORIZONTAL,1=VERTICAL",Geo
770 INPUT "ENTER THE CHIP ORIENTATION",Chip$
780 PRINT USING "10X","THE CHIP ORIENTATION WAS:   ","10A";Chip$
790 !
800 BEEP
810 BEEP
820 ASSIGN @File TO Oldfile$
830 ENTER @File;Emf(*)
840 !
850 !*****
860 ! CONVERT Emf TO DEGREES CELSIUS *
870 !*****
880 !
890 FOR I=0 TO 60
900 Sum=0
910 FOR J=0 TO 7
920 Sum=Sum+D(J)*Emf(I)^J
930 NEXT J
940 T(I)=Sum
950 NEXT I
960 FOR I=71 TO 76
970 Sum=0
980 FOR J=0 TO 7
990 Sum=Sum+D(J)*Emf(I)^J
1000 NEXT J
1010 T(I)=Sum
1020 NEXT I
1030 !
1040 !*****
1050 ! CONVERT Emf TO POWER *
1060 !*****
1070 !
1080 J=1

```

```

1090 Volt=Emf(61)
1100 FOR I=62 TO 70
1110 Power(J)=Emf(I)*(Volt-Emf(I))/Rp
1120 J=J+1
1130 NEXT I
1140 !
1150 IF Geo=0 THEN
1160 L1=8.E-3
1170 Alef=4.8E-5
1180 Arig=4.8E-5
1190 Atop=1.44E-4
1200 Abot=1.44E-4
1210 Le=L1*1000
1220 PRINT USING "10X","HORIZONTAL LENGTH SCALE IS (MM): ",DD.D";Le
1230 ELSE
1240 L1=2.4E-2
1250 Alef=1.44E-4
1260 Arig=1.44E-4
1270 Atop=4.8E-5
1280 Abot=4.8E-5
1290 Le=L1*1000
1300 PRINT USING "10X","VERTICAL LENGTH SCALE IS (MM): ",DD.D";Le
1310 END IF
1320 Acen=1.92E-4
1330 Atot=5.76E-4
1340 !
1350 !*****
1360 !CALCULATE THE AVERAGE TEMPERATURES OF THE BLOCK FACES
1370 !
1380 Tavg(1)=(T(0)*Acen+T(1)*Atop+T(2)*Arig+T(3)*Alef+T(4)*Abot)/Atot
1390 Tavg(2)=(T(6)*Acen+T(7)*Atop+T(8)*Arig+T(9)*Alef+T(10)*Abot)/Atot
1400 Tavg(3)=(T(12)*Acen+T(13)*Atop+T(15)*Arig+T(16)*Abot+Alef*T(14))/Atot
1410 Tavg(4)=(T(18)*Acen+T(19)*Atop+T(20)*Arig+T(21)*Alef+T(22)*Abot)/Atot
1420 Tavg(5)=(T(24)*Acen+T(25)*Atop+T(26)*Arig+T(27)*Alef+T(28)*Abot)/Atot
1430 Tavg(6)=(T(30)*Acen+T(31)*Atop+T(32)*Arig+T(33)*Alef+T(34)*Abot)/(Atot)
1440 Tavg(7)=(T(36)*Acen+T(37)*Atop+T(38)*Arig+T(39)*Alef+T(40)*Abot)/Atot
1450 Tavg(8)=(T(42)*Acen+T(43)*Atop+T(44)*Arig+T(45)*Alef+T(46)*Abot)/(Atot)
1460 Tavg(9)=(T(48)*Acen+T(49)*Atop+T(50)*Arig+T(51)*Alef+T(52)*Abot)/Atot
1470 ! ACCURACY OF THERMOCOUPLES IS 0.5 DEGREES C.
1480 Dt=.5
1490 Dtsvg=.5
1500 Dtsink=.5
1510 !
1520 ! RESISTANCE AND UNCERTAINTY OF PLEXIGLASS FOUND WITH
1530 ! A CONDUCTIVITY OF 0.195 W/m.K & A LENGTH OF 12.0 MM
1540 Rc=320.51
1550 Drc=10.05
1560 !
1570 !*****
1580 ! CHIP BACK SURFACE TEMPERATURES *
1590 !*****
1600 Ts(1)=T(5)
1610 Ts(2)=T(11)
1620 Ts(3)=T(17)
1630 Ts(4)=T(23)
1640 Ts(5)=T(29)
1650 Ts(6)=T(35)

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1660 Ts(7)=T(41)
1670 Ts(8)=T(47)
1680 Ts(9)=T(53)
1690 Tssum=0
1700 FOR J=1 TO 9
1710 Tssum=Tssum+Ts(J)
1720 NEXT J
1730 !
1740 Tsavg=Tssum/9
1750 !
1760 !*****
1770 ! CONDUCTION LOSS CALCULATION *
1780 !*****
1790 Qloss1=(T(5)-T(57))/Rc
1800 Qloss2=(T(11)-T(58))/Rc
1810 Qloss3=(T(17)-T(59))/Rc
1820 Qloss4=(T(23)-T(60))/Rc
1830 Qloss5=(T(29)-T(71))/Rc
1840 Qloss6=(T(35)-T(72))/Rc
1850 Qloss7=(T(41)-T(73))/Rc
1860 Qloss8=(T(47)-T(74))/Rc
1870 Qloss9=(T(53)-T(75))/Rc
1880 Qloss=(Qloss1+Qloss2+Qloss3+Qloss4+Qloss5+Qloss6+Qloss7+Qloss8+Qloss9)/9.
1890 Dqloss=(Qloss)*((Dt/T(57))^2)+(Drc/Rc)^2)^.5
1900 !
1910 !*****
1920 ! AVERAGE SINK TEMPERATURE CALCULATION *
1930 !*****
1940 !
1950 Tsink=(T(54)+T(55)+T(56))/3.
1960 !
1970 PRINT USING "10X, "AVERAGE SINK TEMPERATURE (C): ",DDD.DD";Tsink
1980 !
1990 PRINT
2000 !
2010 ! TWO CHARACTERISTIC LENGTHS WILL BE USED TO CALCULATE NUSSELT NUMBERS:
2020 ! L1 IS BASED ON THE VERTICAL OR HORIZONTAL DIMENSION OF THE CHIP
2030 ! L2 IS BASED ON THE SUMMATION OF THE AREAS DIVIDED BY THE PERIMETER
2040 !
2050 L2=(2.*(6.*24./60.)+2.*(8.*6./28.)+8.*24./64.)*.001
2060 !
2070 PRINT USING "9X, "CHIP Qnet(W) Tavg-Ts Nu1 Nu2 %UNC IN N
    U" ",10A"
2080 PRINT
2090 !
2100 !*****
2110 ! CALCULATION OF NET POWER, Nu, Ra AND UNCERTAINTIES *
2120 !*****
2130 !
2140 FOR J=1 TO 9
2150 !
2160 ! Dpower IS BASED ON ACCURACY OF THE VOLTMETER AND THE PRECISION RESISTORS
    .
2170 ! Dv FOR THE VOLTAGE DROPS ACROSS THE CHIP HEATERS OR PRECISION RESISTORS
2180 ! IS 5E-6 V. Drp=.05 OHMS.
2190 Dv=5.E-6
2200 Drp=.05

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```

2210 Dpower(J)=Power(J)*((Dv/Emf(J+61))^2+(Dv/(Volt-Emf(J+61)))^2+(Drp/Rp)^2)^.
5
2220 !
2230 ! CALCULATION OF Qnet
2240 Qnet(J)=Power(J)-Qloss
2250 Dqnet=(Dpower(J)^2+Dqloss^2)^.5
2260 !
2270 ! CALCULATION OF Tfilm
2280 Tfilm(J)=(Tavg(J)+Tsink)/2
2290 !
2300 ! CALCULATION OF A DELTA TEMPERATURE
2310 Delt(J)=Tavg(J)-Tsink
2320 Ddelt=(Dtavg^2+Dtsink^2)^.5
2330 !
2340 ! CALCULATION OF CONVECTION COEFFICIENT
2350 H(J)=Qnet(J)/(Atot*Delt(J))
2360 Dh=H(J)*((Dqnet/Qnet(J))^2+(Ddelt/Delt(J))^2)^.5
2370 !
2380 ! PHYSICAL PROPERTIES ARE TAKEN FROM THE 1985 3M PRODUCT MANUAL
2390 ! FOR FLUORINERT ELECTRONIC LIQUIDS
2400 !
2410 IF Dliq=0 THEN
2420 !
2430 ! CALCULATION OF FC-75 THERMAL CONDUCTIVITY
2440 K(J)=(.65-7.89474E-4*Tfilm(J))/10
2450 !
2460 ! CALCULATION OF FC-75 DENSITY
2470 Rho(J)=(1.825-.00246*Tfilm(J))*1000
2480 !
2490 ! CALCULATION OF FC-75 SPECIFIC HEAT
2500 Cp(J)=(.241111+3.7037E-4*Tfilm(J))*4187
2510 ! THE 4187 CONVERTS FROM CALORIES TO JOULES
2520 !
2530 ! CALCULATION OF FC-75 KINEMATIC VISCOSITY
2540 N(J)=1.4074-2.964E-2*Tfilm(J)+3.8018E-4*Tfilm(J)^2-2.7308E-6*Tfilm(J)^3+8.
1679E-9*Tfilm(J)^4
2550 ! CONVERT FROM CENTISTOKES TO m^2/s
2560 N(J)=N(J)*1.E-6
2570 !
2580 ! CALCULATION OF THE COEFFICIENT OF THERMAL
2590 ! EXPANSION [BETA]
2600 Beta(J)=.00246/(1.825-.00246*Tfilm(J))
2610 !
2620 END IF
2630 !
2640 IF Dliq=1 THEN
2650 !
2660 ! CALCULATION OF FC-43 THERMAL CONDUCTIVITY
2670 K(J)=(.0666-9.864E-6*Tfilm(J))
2680 !
2690 ! CALCULATION OF FC-43 DENSITY
2700 Rho(J)=(1.913-.00218*Tfilm(J))*1000
2710 !
2720 ! CALCULATION OF FC-43 SPECIFIC HEAT
2730 Cp(J)=(.241111+3.7037E-4*Tfilm(J))*4187
2740 !
2750 ! CALCULATION OF FC-43 KINEMATIC VISCOSITY

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2760 N(J)=8.875-.47007*Tfilm(J)+1.387E-2*Tfilm(J)^2-2.1469E-4*Tfilm(J)^3+1.3139
E-6*Tfilm(J)^4
2770 !
2780 N(J)=N(J)*1.E-6
2790 !
2800 ! CALCULATION OF COEFFICIENT OF THERMAL
2810 ! EXPANSION [BETA]
2820 Beta(J)=.00218/(1.913-.00218*Tfilm(J))
2830 !
2840 END IF
2850 !
2860 IF Dliq=2 THEN
2870 !
2880 ! CALCULATION OF FC-71 THERMAL CONDUCTIVITY
2890 K(J)=.71/10
2900 !
2910 ! CALCULATION OF FC-71 DENSITY
2920 Rho(J)=(2.002-.00224*Tfilm(J))*1000
2930 !
2940 ! CALCULATION OF FC-71 SPECIFIC HEAT
2950 Cp(J)=(.241111+3.7037E-4*Tfilm(J))*4187
2960 !
2970 ! CALCULATION OF FC-71 KINEMATIC VISCOSITY
2980 N(J)=EXP(6.8976-.1388*Tfilm(J)+1.331E-3*Tfilm(J)^2-7.041E-6*Tfilm(J)^3+1.5
23E-8*Tfilm(J)^4)
2990 !
3000 N(J)=N(J)*1.E-6
3010 !
3020 ! CALCULATION OF THE COEFFICIENT OF THERMAL
3030 ! EXPANSION [BETA]
3040 Beta(J)=.00224/(2.002-.00224*Tfilm(J))
3050 !
3060 END IF
3070 !
3080 ! CALCULATION OF THERMAL DIFFUSIVITY [ALPHA]
3090 Alfa(J)=K(J)/(Rho(J)*Cp(J))
3100 !
3110 ! CALCULATION OF PRANDTL NUMBER
3120 Pr(J)=N(J)/Alfa(J)
3130 !
3140 ! CALCULATION OF NUSSELT NUMBERS
3150 Nu1(J)=H(J)*L1/K(J)
3160 Nu2(J)=H(J)*L2/K(J)
3170 Dnu1=Nu1(J)*(Dh/H(J))
3180 Pernu1=(Dnu1/Nu1(J))*100
3190 !
3200 ! CALCULATION OF GRASHOF NUMBERS
3210 Gr1(J)=9.81*Beta(J)*(L1^3)*Delt(J)/N(J)^2
3220 Gr2(J)=9.81*Beta(J)*(L2^3)*Delt(J)/N(J)^2
3230 Dgr1=Gr1(J)*(Ddelt/Delt(J))
3240 !
3250 ! CALCULATION OF RAYLEIGH NUMBERS
3260 Ra1(J)=Gr1(J)*Pr(J)*1.E-6
3270 Ra2(J)=Gr2(J)*Pr(J)*1.E-6
3280 Dra1=Ra1(J)*(Dgr1/Gr1(J))
3290 Pernra1=(Dra1/Ra1(J))*100
3300 !

```

```

3310  ! CALCULATION OF FLUX BASED RAYLEIGH NUMBERS
3320  Raf1(J)=((9.81*Beta(J)*L1^4*Qnet(J))/(K(J)*N(J)*Alfa(J)*Atot))*1.E-6
3330  !
3340  Raf2(J)=((9.81*Beta(J)*L2^4*Qnet(J))/(K(J)*N(J)*Alfa(J)*Atot))*1.E-6
3350  !
3360  ! DATA AND UNCERTAINTY OUTPUT
3370  !
3380  PRINT USING "10X,D,1X,4(4X,DDD.DD,),6X,DDD.DD";J,Qnet(J),Delt(J),Nu1(J),Nu
2(J),Pernu1
3390  PRINT
3400  !
3410  PRINT USING "12X,""TEMP BASED RAYLEIGH NUMBER * E-6 IS: "",DDDDDD.DDD";Ra1
(J)
3420  PRINT USING "12X,""FLUX BASED RAYLEIGH NUMBER * E-6 IS: "",DDDDDD.DDD";Raf
1(J)
3430  PRINT USING "12X,""AVERAGE TEMPERATURE:"",DDD.DD";Tavg(J)
3440  PRINT
3450  PRINT USING "6X,""UNC IN THE NUSSELT NUMBER (Nu1) IS: "",
DD.DD";Dnu1
3460  PRINT USING "6X,""TEMP BASED RAYLEIGH NUMBER * E-6 IS: "",DDD
D.DDD";Ra1(J)
3470  PRINT USING "6X,""UNC IN THE TEMP BASED RAYLEIGH NUMBER * E-6 IS: "",
DD.DD";Dra1
3480  PRINT USING "6X,""%UNC IN THE TEMPERATURE BASED RAYLEIGH NUMBER IS: "",D
DD.DD";Perra1
3490  PRINT
3500  NEXT J
3510  !
3520  Ra1sum=0.
3530  Raf2sum=0.
3540  Nu1sum=0.
3550  Nu2sum=0.
3560  Qnetsum=0.
3570  Deltsum=0.
3580  FOR J=1 TO 9
3590    Ra1sum=Ra1(J)+Ra1sum
3600    Raf2sum=Raf2(J)+Raf2sum
3610    Nu1sum=Nu1(J)+Nu1sum
3620    Nu2sum=Nu2(J)+Nu2sum
3630    Qnetsum=Qnet(J)+Qnetsum
3640    Deltsum=Delt(J)+Deltsum
3650  NEXT J
3660  Ra1(Avg)=Ra1sum/9.
3670  Raf2(Avg)=Raf2sum/9.
3680  Nu1(Avg)=Nu1sum/9.
3690  Nu2(Avg)=Nu2sum/9.
3700  Qnet(Avg)=Qnetsum/9.
3710  Delt(Avg)=Deltsum/9.
3720  !
3730  FOR J=1 TO 3
3740    Rowra1(J)=0
3750    Rownu1(J)=0
3760  NEXT J
3770  FOR J=1 TO 3
3780    Rowra1(J)=(Ra1(J)+Ra1(J+3)+Ra1(J+6))/3.0
3790    Rownu1(J)=(Nu1(J)+Nu1(J+3)+Nu1(J+6))/3.0
3800  NEXT J

```

```

3810  I
3820  PRINT
3830  PRINT
3840  PRINT
3850  PRINT USING "12X","ARRAY AVG Ra1•E-6 IS:","",DDDDD.DDD";Ra1(Avg)
3860  PRINT USING "12X","ARRAY AVG Nu1 IS:","",DDD.DD";Nu1(Avg)
3870  PRINT
3880  PRINT USING "12X","ROW 1 AVG Ra1•E-6 IS:","",DDDDD.DDD";Rowra1(1)
3890  PRINT USING "12X","ROW 1 AVG Nu1 IS:","",DDD.DD";Rownu1(1)
3900  PRINT
3910  PRINT USING "12X","ROW 2 AVG Ra1•E-6 IS:","",DDDDD.DDD";Rowra1(2)
3920  PRINT USING "12X","ROW 2 AVG Nu1 IS:","",DDD.DD";Rownu1(2)
3930  PRINT
3940  PRINT USING "12X","ROW 3 AVG Ra1•E-6 IS:","",DDDDD.DDD";Rowra1(3)
3950  PRINT USING "12X","ROW 3 AVG Nu1 IS:","",DDD.DD";Rownu1(3)
3960  PRINT
3970  PRINT USING "12X","ARRAY AVG Qnet IS:","",D.DD";Qnet(Avg)
3980  PRINT USING "12X","ARRAY AVG (TAUG-TSINK) IS:","",DD.DD";Delt(Avg)
3990  ASSIGN @File TO *
4000  END

```

APPENDIX C. COMPUTER PROGRAM ACQ2

```

10  | FILE ACQ2
20  |
30  | WRITTEN BY LCDR R. THOMPSON MAR 92.
40  |
50  | THIS PROGRAM ACQUIRES THE VOLTAGE DATA
60  | FROM THE HP3456A DVM VIA THE HP3457A DAS
70  | FOR THE 3 BY 3 ARRAY NWSC CIRCUIT BOARD.
80  | IT IS WRITTEN IN BASIC 2.0.
90  | DVM - DIGITAL VOLTMETER
100 | DAS - DATA ACQUISITION SYSTEM
110 |
120 DIM Emf(64),Power(1:9),T(64)
130 |
140 |PRECISION RESISTOR VALUE IN OHMS
150 Rp=2.0
160 |
170 | ASSIGN OUTPUT TO HP THINK JET PRINTER
180 PRINTER IS 701
190 BEEP
200 INPUT "ENTER THE INPUT MODE: 0=SYS, 1=FILE",Im
210 |
220 IF Im=1 THEN
230 BEEP
240 INPUT "ENTER THE NAME OF THE FILE TO BE READ",Oldfile$
250 |
260 PRINT USING "15X","THESE RESULTS ARE STORED IN FILE : ",10A";Oldfile$
270 ELSE
280 BEEP
290 INPUT "ENTER THE NAME OF THE NEW FILE",Newfile$
300 PRINT USING "10X","THESE RESULTS ARE STORED IN FILE: ",10A";Newfile$
310 END IF
320 PRINT
330 PRINT USING "15X","DATA TAKEN BY THOMPSON"
340 |
350 INPUT "ENTER THE BATH TEMP",B$
360 PRINT USING "15X","BATH TEMP WAS: ",10A";B$
370 |
380 INPUT "ENTER THE WALL SPACING",Wall$
390 PRINT USING "15X","SPACING WAS: ",10A";Wall$
400 |
410 INPUT "ENTER THE TYPE OF LIQUID USED",Liquid$
420 PRINT USING "15X","THE FLUORINERT USED WAS: ",10A";Liquid$
430 IF Im=1 THEN ASSIGN @File TO Oldfile$
440 |
450 IF Im=0 THEN
460 CREATE BDAT Newfile$,5
470 ASSIGN @File TO Newfile$
480 END IF
490 |
500 | READ DATA INTO HP9826 COMPUTER. 709 IS THE DAS, 722 IS THE DVM.

```

```

510 !
520 IF Im=0 THEN
530 ! AR RESETS DAS. AF IS FIRST CHANNEL, AL IS LAST CHANNEL.
540 OUTPUT 709;"AR AF00 AL13"
550 ! F1 SETS FUNCTION TO DC VOLTS. R1 SETS RANGE TO AUTO.
560 ! T1 SETS TRIGGER TO INTERNAL. Z0 SETS AUTO ZERO TO OFF.
570 ! FLO SETS FILTER TO OFF.
580 OUTPUT 722;"F1 R1 T1 Z0 FLO"
590 !
600 FOR I=0 TO 13
610 ! AS CAUSES THE DAS TO ANALOG STEP THROUGH THE CHANNELS.
620 OUTPUT 709;"AS"
630 WAIT 1
640 ! ENTER SENDS VOLTAGES FROM DVM TO DAS.
650 ENTER 722;Emf(I)
660 BEEP
670 NEXT I
680 OUTPUT 709;"AR AF41 AL64"
690 FOR I=41 TO 64
700 OUTPUT 709;"AS"
710 WAIT 1
720 ENTER 722;Emf(I)
730 BEEP
740 NEXT I
750 OUTPUT @File;Emf(*)
760 !
770 ELSE
780 ENTER @File;Emf(*)
790 END IF
800 !
810 ! AR RESETS DAS
820 OUTPUT 709;"AR"
830 !
840 ! CONVERT THERMOCOUPLE VOLTAGES TO TEMPERATURE IN DEGREES CELSIUS.
850 ! THERMOCOUPLES CALIBRATED AGAINST PLATINUM RESISTANCE THERMOMETER
860 ! MARCH 2-3, 1992. TEMP. RANGE 10-100 DEG. C.
870 ! CALIBRATION CURVE FIT BY TABLECURVE SOFTWARE.
880 ! NOTE: CAL. CURVE VOLTAGES ARE MILLIVOLTS, Emf(I)'S ARE IN MICROVOLTS.
890 A=.24577483
900 B=24.896086
910 C=-.075219169
920 FOR I=11 TO 13
930 T(I)=A+B*Emf(I)*1.E+3+C*(1.E+3*Emf(I))^3
940 NEXT I
950 FOR I=41 TO 59
960 T(I)=A+B*Emf(I)*1.E+3+C*(1.E+3*Emf(I))^3
970 NEXT I
980 !
990 ! CHIP TSE'S (THE TRANSISTORS) ALSO CALIBRATED SAME TIME AS ABOVE.
1000 ! TEMP. (DEG. C) VS. BASE-EMITTER (Vbe) VOLTAGE CALIBRATION CURVES
1010 ! PRODUCED USING TABLECURVE SOFTWARE.
1020 T(60)=577.58074-575.54353*Emf(60)
1030 T(61)=575.66889-574.05057*Emf(61)
1031 T(62)=577.64556-575.50862*Emf(62)
1032 T(63)=576.98331-574.69738*Emf(63)
1033 T(64)=578.28560-575.99793*Emf(64)
1050 !

```

```

1060 PRINT USING "15X","VOLTAGE SUPPLY WAS: ",DD.DD";Emf(10)
1070 PRINT USING "15X","AMBIENT TEMP WAS: ",DD.D";T(50)
1080 PRINT
1090 ' POWER CALCULATIONS
1100 '
1110 FOR I=1 TO 3
1120   Power(I)=Emf(1)+Emf(5)/(3+Rp)
1130 NEXT I
1140 J=4
1150 FOR I=1 TO 3
1160   Power(J)=Emf(I)+Emf(I+5)/Rp
1170   J=J+1
1180 NEXT I
1190 FOR I=7 TO 9
1200   Power(I)=Emf(4)+Emf(9)/(3+Rp)
1210 NEXT I
1220 '
1230 BEEP
1240 '
1250 PRINT USING "10X","ALL TEMPERATURES ARE IN DEGREES CELSIUS"
1260 '
1270 PRINT
1280 '
1290 PRINT USING "10X","CHIP          CHIP LID          POWER(W)""
1300 PRINT
1310 PRINT
1320 PRINT USING "1X","CHIP NO1:  N/A          N/A          ",D.DDD";Powe
r(1)
1330 PRINT USING "12X","          (AVG)""
1340 PRINT
1350 PRINT USING "1X","CHIP NO2:  ",2(DDD.DD,10X),D.DDD";T(60),T(45),Power(2)
1360 PRINT USING "12X","          (AVG)""
1370 PRINT
1380 PRINT USING "1X","CHIP NO3:  N/A          N/A          ",D.DDD";Powe
r(3)
1390 PRINT USING "12X","          (AVG)""
1400 PRINT
1410 PRINT USING "1X","CHIP NO4:  ",2(DDD.DD,10X),D.DDD";T(61),T(46),Power(4)
1420 PRINT
1430 PRINT
1440 PRINT USING "1X","CHIP NO5:  ",2(DDD.DD,10X),D.DDD";T(62),T(47),Power(5)
1450 PRINT
1460 PRINT
1470 PRINT USING "1X","CHIP NO6:  ",2(DDD.DD,10X),D.DDD";T(63),T(48),Power(6)
1480 PRINT
1490 PRINT
1500 PRINT USING "1X","CHIP NO7:  N/A          N/A          ",D.DDD";Powe
r(7)
1510 PRINT USING "12X","          (AVG)""
1520 PRINT
1530 PRINT USING "1X","CHIP NO8:  ",2(DDD.DD,10X),D.DDD";T(64),T(49),Power(8)
1540 PRINT USING "12X","          (AVG)""
1550 PRINT
1560 PRINT USING "1X","CHIP NO9:  N/A          N/A          ",D.DDD";Fowe
r(9)
1570 PRINT USING "12X","          (AVG)""
1580 PRINT

```



```

1590 PRINT USING "12Y","CIRCUIT BOARD SUBSTRATE SURFACE TEMPERATURES""
1600 PRINT
1610 PRINT USING "1Y","A: "" ,DDD.DD";T(41)
1620 PRINT USING "1Y","B: "" ,DDD.DD";T(42)
1630 PRINT USING "1Y","C: "" ,DDD.DD";T(43)
1640 PRINT USING "1Y","D: "" ,DDD.DD";T(44)
1650 PRINT
1660 PRINT USING "5X","HEAT EXCHANGER TEMPERATURES:      RIGHT      CENTER      LE
FT""
1670 PRINT USING "10X","BOTTOM IS INSULATED""
1680 PRINT USING "10X","TOP: "" ,24X,3(DD.DD,5X)";T(11),T(12),T(13)
1690 PRINT
1700 I
1710 PRINT USING "5X","CIRCUIT BOARD ASSEMBLY BACK TEMPERATURES ARE: ""
1720 PRINT
1730 PRINT USING "10X","CHIP NO1: "" ,2X,DD.DD";T(51)
1740 PRINT USING "10X","CHIP NO2: "" ,2X,DD.DD";T(52)
1750 PRINT USING "10X","CHIP NO3: "" ,2X,DD.DD";T(53)
1760 PRINT USING "10X","CHIP NO4: "" ,2X,DD.DD";T(54)
1770 PRINT USING "10X","CHIP NO5: "" ,2X,DD.DD";T(55)
1780 PRINT USING "10X","CHIP NO6: "" ,2X,DD.DD";T(56)
1790 PRINT USING "10X","CHIP NO7: "" ,2X,DD.DD";T(57)
1800 PRINT USING "10X","CHIP NO8: "" ,2X,DD.DD";T(58)
1810 PRINT USING "10X","CHIP NO9: "" ,2X,DD.DD";T(59)
1820 BEEP
1830 I REASSIGN PRINTER TO THE CRT
1840 PRINTER IS I
1850 ASSIGN @File TO *
1860 END

```

APPENDIX D. COMPUTER PROGRAM CALC2

```

10  ! FILE CALC2
20  !
30  ! WRITTEN BY LCDR R. THOMPSON MAR 92.
40  ! MODIFIED EARLY APR 92. BASED ON THE PROGRAM
50  ! CALCDIEL WRITTEN AND MODIFIED BY PAMUK,
60  ! BENEDICT, TORRES, AYTA, MATTHEWS AND THOMPSON.
70  !
80  ! THIS PROGRAM ANALYZES THE DATA READ FROM
90  ! A DESIGNATED "ACQ2" DATA FILE. ACQ2 DATA FILES
100 ! ARE FOR THE 3 BY 3 ARRAY NWSC CIRCUIT BOARD.
110 ! IT REDUCES THE DATA TO CALCULATIONS OF NET POWER, RAYLEIGH
120 ! AND NUSSELT NUMBER. THE UNCERTAINTY ANALYSIS IS ALSO INCLUDED.
130 ! IT IS WRITTEN IN BASIC 2.0.
140 !
150 ! VARIABLES USED ARE:
160 ! EMF   : DATA FILE VOLTAGES.
170 ! POWER : POWER DISSIPATED BY THE CHIP RESISTORS (W)
180 ! T(I)  : TEMPERATURES CONVERTED FROM THERMO-
190 !        COUPLE VOLTAGES (DEG. C)
200 ! Ts    : CIRCUIT BOARD ASSEMBLY BACK TEMPERATURE (DEG. C)
210 ! Tfilm : DIELECTRIC FILM TEMPERATURE (DEG. C)
220 ! Tavg  : AVG. TEMP. OF THE 5 CHIP TSE's (DEG. C)
230 ! Qnet  : ELECTRIC POWER MINUS CONDUCTION LOSSES (W)
240 ! Tsink : AVERAGE OF THE 3 THERMOCOUPLES IN
250 !        THE UPPER HEAT EXCHANGER (DEG. C)
260 ! Nu    : LENGTH BASED NUSSELT NUMBER
270 ! Rc    : THERMAL RESISTANCE FOR CONDUCTION (DEG. C/W)
280 ! D...  : UNCERTAINTY OF A VARIABLE (EXCEPT
290 !        Dliq AND Delt)
300 ! OTHER VARIABLES NOT SELF-EXPLANATORY ARE DEFINED IN THE PROGRAM
310 !
320 DIM Emf(64),Power(1:9),T(64),Ts(1:9),Tside(1:9),Tlid(1:9)
330 DIM Qnet(1:9),Delt(1:9),H(1:9),Nu(1:9),Ra(1:9)
340 DIM Gr(1:9),Qloss(1:9),Dh(1:9),Dpower(1:9)
350 DIM Rowgr(1:3),Rowra(1:3),Rownu(1:3)
360 !
370 ! PRECISION RESISTOR VALUE IN OHMS
380 Rp=2.0
390 ! ASSIGN OUTPUT TO HP THINKJET PRINTER
400 PRINTER IS 701
410 BEEP
420 !
430 PRINT USING "10X","DATA TAKEN BY THOMPSON""
440 INPUT "ENTER THE NAME OF THE FILE CONTAINING DATA",Oldfile$
450 !
460 PRINT USING "10X","THE RAW Emf DATA ARE FROM THE FILE:   ","10A";Oldfile$
470 !
480 INPUT "ENTER THE APPROX. POWER SETTING ",Power$
490 PRINT USING "9X"," THE APPROX. POWER SETTING PER CHIP WAS:   ","10A";Power$
500 !

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510 INPUT "ENTER THE TYPE OF LIQUID USED",Liquid$
520 PRINT USING "10X,";"THE FLUORINERT USED WAS:  ","10A";Liquid$
530 INPUT "ENTER THE TYPE OF DIELECTRIC:0=FC-75,1=FC-43,2=FC-71",Dliq
540 !
550 INPUT "ENTER THE WALL SPACING",Wall$
560 PRINT USING "10X,";"THE DISTANCE TO THE FRONT WALL WAS:  ","10A";Wall$
570 !
580 BEEP
590 ASSIGN @File TO Oldfile$
600 ENTER @File;Emf(*)
610 !
620 ! CONVERT THERMOCOUPLE VOLTAGES TO TEMPERATURE IN DEGREES CELSIUS.
630 ! THERMOCOUPLES CALIBRATED AGAINST PLATINUM RESISTANCE THERMOMETER
640 ! MARCH 2-3, 1992. TEMP. RANGE 10-100 DEG. C.
650 ! CALIBRATION CURVE FIT BY TABLECURVE SOFTWARE.
660 ! NOTE: CAL. CURVE VOLTAGES ARE MILLIVOLTS, Emf(I)S ARE IN MICROVOLTS.
670 A=.24977483
680 B=24.896088
690 C=-.079219169
700 FOR I=11 TO 13
710 T(I)=A+B*Emf(I)*1.E+3+C*(1.E+3*Emf(I))^3
720 NEXT I
730 FOR I=41 TO 59
740 T(I)=A+B*Emf(I)*1.E+3+C*(1.E+3*Emf(I))^3
750 NEXT I
760 !
770 ! CHIP TSE'S (THE TRANSISTORS) ALSO CALIBRATED SAME TIME AS ABOVE.
780 ! TEMP. (DEG. C) VS. BASE-EMITTER (Vbe) VOLTAGE CALIBRATION CURVES
790 ! PRODUCED USING TABLECURVE SOFTWARE.
800 T(60)=577.58074-575.54353*Emf(60)
810 T(61)=575.66889-574.05057*Emf(61)
820 T(62)=577.64556-575.50862*Emf(62)
830 T(63)=576.98331-574.69738*Emf(63)
840 T(64)=578.28560-575.99793*Emf(64)
850 Tavg=(T(60)+T(61)+T(62)+T(63)+T(64))/5
860 !
870 ! POWER CALCULATIONS
880 FOR I=1 TO 3
890 Power(I)=Emf(0)*Emf(5)/(3*Rp)
900 NEXT I
910 J=4
920 FOR I=1 TO 3
930 Power(J)=Emf(I)*Emf(I+5)/Rp
940 J=J+1
950 NEXT I
960 FOR I=7 TO 9
970 Power(I)=Emf(4)*Emf(9)/(3*Rp)
980 NEXT I
990 !
1000 ! BELOW LENGTHS AND AREAS BASED ON CHIP MODELED AS A SQUARE WAFER.
1010 ! ALL DIMENSIONS IN m OR SQUARE m.
1020 L1=8.89E-3
1030 Alef=1.65E-5
1040 Arig=Alef
1050 Atop=Alef
1060 Abot=Alef
1070 Alid=7.90E-5

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1080 Le=L1*1000
1090 PRINT USING "10X,";"LENGTH SCALE IS (mm): ",";D.DD";Le
1100 ! Atot IS TOTAL AREA FOR CONVECTION
1110 Atot=Alef+Arig+Atop+Abot+Alid
1120 !
1130 ! CHIP SIDE TEMPERATURES. ASSUMES IT IS THE AVERAGE OF THE TSE TEMP.
1140 ! AND THE LID TEMP., WITH CHIP TEMPS. 1,4,7 ASSUMED TO BE EQUAL AND
1150 ! CHIP TEMPS. 3,6,9 ASSUMED TO BE EQUAL.
1160 !
1170 Tside(1)=(T(46)+T(61))/2
1180 Tside(2)=(T(45)+T(60))/2
1190 Tside(3)=(T(48)+T(63))/2
1200 Tside(4)=(T(46)+T(61))/2
1210 Tside(5)=(T(47)+T(62))/2
1220 Tside(6)=(T(48)+T(63))/2
1230 Tside(7)=(T(46)+T(61))/2
1240 Tside(8)=(T(49)+T(64))/2
1250 Tside(9)=(T(48)+T(63))/2
1260 !
1270 ! CHIP LID TEMPERATURES. SAME ASSUMPTION AS CHIP SIDES.
1280 Tlid(1)=T(46)
1290 Tlid(2)=T(45)
1300 Tlid(3)=T(48)
1310 Tlid(4)=T(46)
1320 Tlid(5)=T(47)
1330 Tlid(6)=T(48)
1340 Tlid(7)=T(46)
1350 Tlid(8)=T(49)
1360 Tlid(9)=T(48)
1370 !
1380 ! CIRCUIT BOARD ASSEMBLY BACK TEMPERATURES
1390 Tssum=0
1400 FOR I=1 TO 9
1410   Ts(I)=T(I+50)
1420   Tssum=Tssum+Ts(I)
1430 NEXT I
1440 !
1450 Tsavg=Tssum/9
1460 !
1470 ! CONDUCTION LOSS CALCULATION
1480 ! ONE DIMENSIONAL CONDUCTION ASSUMED THROUGH THE FOLLOWING:
1490 ! 4.06E-4 m (0.016") OF SILICON (CHIP)
1500 ! 7.62E-4 m (0.030") OF ALUMINA (CIRCUIT BOARD)
1510 ! 1.27E-4 m (0.005") OF SILICONE RUBBER
1520 ! 1.18E-2 m (0.465") OF ACRYLIC (PLEXIGLAS)
1530 ! VALUES FOR THERMAL CONDUCTIVITY K IN W/m-DEG. C. SOURCE:
1540 ! SILICON, ALUMINA: NWSO, MR. TONY BUECHLER
1550 ! ALL OTHERS: MATERIALS ENGINEERING 1978 MATERIALS SELECTOR, VOL. 86,
1560 ! NO. 6, REINHOLD PUBLISHING, 1977, PAGES 202, 143.
1570 !
1580 ! SILICON: 168*EXP(-0.00458*T). T IS SILICON TEMP. IN DEG. C.
1590 ! ALUMINA: 16.7 SILICON RUBBER: 0.225 ACRYLIC: 0.208
1600 ! HEAT FLUX q FOUND BY STANDARD EQUATION
1610 !  $q = \text{AREA} \cdot \Delta T / (\text{SUM OF LENGTHS} / K'S)$ 
1620 !
1630 ! IT IS ASSUMED FOR CALCULATIONS THAT THE FOLLOWING TEMPS. ARE THE SAME:
1640 ! CHIP1 = CHIP4 = CHIP7

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1650 ! CHIP3 = CHIP6 = CHIP9
1660 !
1670 ! E IS AREA (SQUARE m), F & S ARE L/K VALUES
1680 E=7.90E-5
1690 F=2.42E-6
1700 G=.00458
1710 S=5.73E-2
1720 !
1730 Qloss(1)=E*(T(61)-T(51))/(F*EXP(G*T(61))+S)
1740 Qloss(2)=E*(T(60)-T(52))/(F*EXP(G*T(60))+S)
1750 Qloss(3)=E*(T(63)-T(53))/(F*EXP(G*T(63))+S)
1760 Qloss(4)=E*(T(61)-T(54))/(F*EXP(G*T(61))+S)
1770 Qloss(5)=E*(T(62)-T(55))/(F*EXP(G*T(62))+S)
1780 Qloss(6)=E*(T(63)-T(56))/(F*EXP(G*T(63))+S)
1790 Qloss(7)=E*(T(61)-T(57))/(F*EXP(G*T(61))+S)
1800 Qloss(8)=E*(T(64)-T(58))/(F*EXP(G*T(64))+S)
1810 Qloss(9)=E*(T(63)-T(59))/(F*EXP(G*T(63))+S)
1820 !
1830 Qlosssum=0
1840 FOR I=1 TO 9
1850 Qlosssum=Qlosssum+Qloss(I)
1860 NEXT I
1870 Qlossavg=Qlosssum/9
1880 !
1890 ! Rc WILL BE CALCULATED ASSUMING SILICON TEMP. IS 60 DEG. C, WHICH IS
1900 ! ABOUT MID-RANGE FOR THE EXPERIMENTS
1910 ! Rc = (1/AREA)*SUM OF LENGTHS/K'S
1920 ! ALSO ASSUME Drc IS 5% OF Rc
1930 Rc=725
1940 Drc=36.3
1950 ! Dtlid, Dtside, Dtchip, Dts, AND Dtsink BASED ON THERMOCOUPLE CALIBRATION
1960 Dtchip=.3
1970 Dts=.3
1980 Dtlid=.3
1990 Dtside=.3
2000 Dtsink=.3
2010 ! SINCE CONDUCTION TEMP. DIFFERENCE = Tchip - Ts, Dt IS THE RMS OF Dtchip
AND Dts
2020 Dt=(Dtchip^2+Dts^2)^.5
2030 Dqloss=(Qlossavg)*((Dt/(Tavg-Tsavg))^2+(Drc/Rc)^2)^.5
2040 !
2050 ! AVERAGE SINK TEMPERATURE CALCULATION
2060 Tsink=(T(11)+T(12)+T(13))/3
2070 !
2080 PRINT USING "10X,""AVERAGE SINK TEMPERATURE (C):  ",DD.DD,Tsink
2090 PRINT
2100 !
2110 ! THE CHARACTERISTIC LENGTH TO BE USED TO CALCULATE THE NUSSELT NUMBER
2120 ! IS L1, WHICH WAS PREVIOUSLY DEFINED
2130 !
2140 PRINT USING "9X,""CHIP    Qnet(W)    Delta T    Nu    %UNC IN NU"" ,10A"
2150 PRINT
2160 !
2170 ! CALCULATION OF NET POWER, Nu, Ra AND UNCERTAINTIES
2180 FOR J=1 TO 9
2190 ! Dpower IS BASED ON ACCURACY OF THE DVM AND THE PRECISION RESISTORS.
2200 ! Dv FOR VOLTAGE DROPS ACROSS THE CHIP RESISTOR OR THE PRECISION RESISTOR

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2210 ! IS SE-6 V. Drp = 0.05 OHMS.
2220 Dv=5.E-6
2230 Drp=.05
2240 FOR I=1 TO 3
2250 Dpower(I)=Power(I)*((3*Dv/Emf(0))^2+(Dv/Emf(5))^2+(Drp/Rp)^2)^.5
2260 NEXT I
2270 L=4
2280 FOR I=1 TO 3
2290 Dpower(L)=Power(L)*((Dv/Emf(L))^2+(Dv/Emf(I+5))^2+(Drp/Rp)^2)^.5
2300 L=L+1
2310 NEXT I
2320 FOR I=7 TO 9
2330 Dpower(I)=Power(I)*((3*Dv/Emf(4))^2+(Dv/Emf(9))^2+(Drp/Rp)^2)^.5
2340 NEXT I
2350 !
2360 ! CALCULATION OF Qnet
2370 Qnet(J)=Power(J)-Qloss(J)
2380 Dqnet=(Dpower(J)^2+Dqloss^2)^.5
2390 !
2400 ! CALCULATION OF Tfilm
2410 Tfilm=(Tavg+Tsink)/2
2420 ! Dtagv BASED ON TSE CALIBRATION
2430 Dtagv=.275
2440 !
2450 ! CALCULATION OF Delta T BASED ON LID AREA COMPRISING ABOUT 55% OF TOTAL
2460 ! CONVECTION AREA, AND THE CHIP SIDES 45% OF THE AREA
2470 Delt(J)=(.55*Tlid(J)+.45*Tside(J))-Tsink
2480 Ddelt=(Dtlid^2+Dtside^2+Dtsink^2)^.5
2490 !
2500 ! CALCULATION OF CONVECTION COEFFICIENT
2510 H(J)=Qnet(J)/(Atot*Delt(J))
2520 Dh(J)=H(J)*((Dqnet/Qnet(J))^2+(Ddelt/Delt(J))^2)^.5
2530 NEXT J
2540 !
2550 ! PHYSICAL PROPERTIES ARE TAKEN FROM THE 1985 3M PRODUCT MANUAL
2560 ! FOR FLUORINERT ELECTRONIC LIQUIDS
2570 !
2580 IF Dliq=0 THEN
2590 !
2600 ! CALCULATION OF FC-75 THERMAL CONDUCTIVITY
2610 K=(.65-7.89474E-4*Tfilm)/10
2620 !
2630 ! CALCULATION OF FC-75 DENSITY
2640 Rho=(1.825-.00246*Tfilm)*1000
2650 !
2660 ! CALCULATION OF FC-75 SPECIFIC HEAT
2670 Cp=(.241111+3.7037E-4*Tfilm)*4187
2680 ! THE 4187 CONVERTS FROM CALORIES TO JOULES
2690 !
2700 ! CALCULATION OF FC-75 KINEMATIC VISCOSITY
2710 N=1.4074-2.964E-2*Tfilm+3.8018E-4*Tfilm^2-2.7308E-6*Tfilm^3+8.1679E-9*Tfilm^4
2720 ! CONVERT FROM CENTISTOKES TO m^2/s
2730 N=N*1.E-6
2740 !
2750 ! CALCULATION OF THE COEFFICIENT OF THERMAL
2760 ! EXPANSION [BETA]

```



```

2770 Beta=.00246/(1.825-.00246*Tfilm)
2780 !
2790 END IF
2800 !
2810 IF Dliq=1 THEN
2820 !
2830 ! CALCULATION OF FC-43 THERMAL CONDUCTIVITY
2840 K=.0666-9.864E-6*Tfilm
2850 !
2860 ! CALCULATION OF FC-43 DENSITY
2870 Rho=(1.913-.00218*Tfilm)*1000
2880 !
2890 ! CALCULATION OF FC-43 SPECIFIC HEAT
2900 Cp=(.241111+3.7037E-4*Tfilm)*4187
2910 !
2920 ! CALCULATION OF FC-43 KINEMATIC VISCOSITY
2930 N=8.875-.47007*Tfilm+1.387E-2*Tfilm^2-2.1469E-4*Tfilm^3+1.3139E-6*Tfilm^4
2940 !
2950 N=N*1.E-6
2960 !
2970 ! CALCULATION OF COEFFICIENT OF THERMAL
2980 ! EXPANSION [BETA]
2990 Beta=.00218/(1.913-.00218*Tfilm)
3000 !
3010 END IF
3020 !
3030 IF Dliq=2 THEN
3040 !
3050 ! CALCULATION OF FC-71 THERMAL CONDUCTIVITY
3060 K=.071
3070 !
3080 ! CALCULATION OF FC-71 DENSITY
3090 Rho=(2.002-.00224*Tfilm)*1000
3100 !
3110 ! CALCULATION OF FC-71 SPECIFIC HEAT
3120 Cp=(.241111+3.7037E-4*Tfilm)*4187
3130 !
3140 ! CALCULATION OF FC-71 KINEMATIC VISCOSITY
3150 N=EXP(6.8976-.1388*Tfilm+1.331E-3*Tfilm^2-7.041E-6*Tfilm^3+1.523E-8*Tfilm^
4)
3160 !
3170 N=N*1.E-6
3180 !
3190 ! CALCULATION OF THE COEFFICIENT OF THERMAL
3200 ! EXPANSION [BETA]
3210 Beta=.00224/(2.002-.00224*Tfilm)
3220 !
3230 END IF
3240 !
3250 ! CALCULATION OF THERMAL DIFFUSIVITY [ALPHA]
3260 Alfa=K/(Rho*Cp)
3270 !
3280 ! CALCULATION OF PRANDTL NUMBER
3290 Pr=N/Alfa
3300 FOR J=1 TO 9
3310 !
3320 ! CALCULATION OF NUSSELT NUMBER

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```

3330 Nu(J)=H(J)*L1/K
3340 Dnu=Nu(J)*(Dh(J)/H(J))
3350 Pernu=(Dnu/Nu(J))*100
3360 !
3370 ! CALCULATION OF GRASHOF NUMBER
3380 Gr(J)=9.81*Beta*(L1^3)*Delt(J)/N^2
3390 Dgr=Gr(J)*(Ddelt/Delt(J))
3400 Pergr=(Dgr/Gr(J))*100
3410 !
3420 ! CALCULATION OF RAYLEIGH NUMBER
3430 Ra(J)=Gr(J)*Pr*1.E-6
3440 Dra=Ra(J)*(Dgr/Gr(J))
3450 Perdra=(Dra/Ra(J))*100
3460 !
3470 ! DATA AND UNCERTAINTY OUTPUT
3480 IF J=6 THEN
3490 PRINT
3500 PRINT
3510 PRINT
3520 PRINT
3530 PRINT
3540 PRINT
3550 PRINT USING "10X,""PAGE 2 OF FILE:  ",10A";Oldfile$
3560 PRINT
3570 PRINT USING "10X,D,1X,3(5X,DD.DD,),5X,DDD.DDD";J,Qnet(J),Delt(J),Nu(J),Per
nu
3580 ELSE
3590 PRINT
3600 PRINT USING "10X,D,1X,3(5X,DD.DD,),5X,DDD.DDD";J,Qnet(J),Delt(J),Nu(J),Per
nu
3610 END IF
3620 PRINT
3630 PRINT USING "12X,""TEMP BASED RAYLEIGH NUMBER * E-6 IS: ",DDDDDD.DDD";Ra(
J)
3640 PRINT USING "12X,""GRASHOF NUMBER IS: ",DDDDDD.DDD";Gr(
J)
3650 PRINT
3660 PRINT USING "6X,""UNC IN THE NUSSELT NUMBER IS: ",D
D.DDD";Dnu
3670 PRINT USING "6X,""UNC IN THE TEMP. BASED RAYLEIGH NUMBER * E-6 IS: ",D
D.DDD";Dra
3680 PRINT USING "6X,""UNC IN THE GRASHOF NUMBER IS: ",DDD
DDD.D";Dgr
3690 ! ALGEBRAICALLY, Perdra = Pergr
3700 PRINT USING "6X,""%UNC IN THE RAYLEIGH OR GRASHOF NUMBER IS: ",D
D.DDD";Perdra
3710 NEXT J
3720 !
3730 Resum=0
3740 Nusum=0
3750 Qnetsum=0
3760 Deltsum=0
3770 Grsum=0
3780 FOR J=1 TO 9
3790 Resum=Ra(J)+Resum
3800 Nusum=Nu(J)+Nusum
3810 Qnetsum=Qnet(J)+Qnetsum

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```

3820   Deltsum=Delt(J)+Deltsum
3830   Grsum=Gr(J)+Grsum
3840   NEXT J
3850   Raavg=Resum/9
3860   Nuavg=Nusum/9
3870   Qnetavg=Qnetsum/9
3880   Deltavg=Deltsum/9
3890   Gravg=Grsum/9
3900   !
3910   FOR J=1 TO 3
3920     Rowra(J)=0
3930     Rownu(J)=0
3940     Rowgr(J)=0
3950   NEXT J
3960   FOR J=1 TO 3
3970     Rowra(J)=(Ra(J)+Ra(J+3)+Ra(J+6))/3
3980     Rownu(J)=(Nu(J)+Nu(J+3)+Nu(J+6))/3
3990     Rowgr(J)=(Gr(J)+Gr(J+3)+Gr(J+6))/3
4000   NEXT J
4010   !
4020   PRINT
4030   PRINT USING "19X,""TDP ROW AVG Ra+E-6 IS:  "" ,DDDDD.DDD";Rowra(3)
4040   PRINT USING "19X,""TDP RDW AVG Nu IS:      "" ,DDD.DD";Rownu(3)
4050   PRINT USING "19X,""TDP RDW AVG Gr IS:      "" ,DDDDDDD.DDD";Rowgr(3)
4060   PRINT
4070   PRINT USING "19X,""MID RDW AVG Ra+E-6 IS:  "" ,DDDDD.DDD";Rowra(2)
4080   PRINT USING "19X,""MID RDW AVG Nu IS:      "" ,DDD.DD";Rownu(2)
4090   PRINT USING "19X,""MID RDW AVG Gr IS:      "" ,DDDDDDD.DDD";Rowgr(2)
4100   PRINT
4110   PRINT USING "19X,""BDT ROW AVG Ra+E-6 IS:  "" ,DDDDD.DDD";Rowra(1)
4120   PRINT USING "19X,""BDT RDW AVG Nu IS:      "" ,DDD.DD";Rownu(1)
4130   PRINT USING "19X,""BDT RDW AVG Gr IS:      "" ,DDDDDDD.DDD";Rowgr(1)
4140   PRINT
4150   PRINT USING "19X,""ARRAY AVG Qnet IS:      "" ,D.DD";Qnetavg
4160   PRINT USING "19X,""ARRAY AVG Delta T IS:    "" ,DD.DD";Deltavg
4170   PRINT USING "19X,""ARRAY AVG Ra+E-6 IS:    "" ,DDDDD.DDD";Raavg
4180   PRINT USING "19X,""ARRAY AVG Nu IS:        "" ,DDD.DD";Nuavg
4190   PRINT USING "19X,""ARRAY AVG Gr IS:        "" ,DDDDDDD.DDD";Gravg
4200   PRINT USING "19X,""ARRAY AVG Pr IS:        "" ,DDDD.D";Pr
4210   ASSIGN @File TO *
4220   END

```

APPENDIX E. UNCERTAINTY ANALYSIS

The software programs CALCDIEL and CALC2 also calculated a standard zeroth order uncertainty analysis as described in Beckwith and Marangoni (1990). The uncertainties were calculated for the Nusselt number and Rayleigh number for each component on the circuit board. A complete description of the CALCDIEL uncertainty analysis can be found in Matthews' thesis. The analysis performed by CALC2 for the NSWC circuit board is described below in expression form, with numerical results omitted for generality. The small uncertainty associated with the thermophysical properties was neglected. Other assumptions are included, where appropriate.

1. Conduction heat loss through the circuit board assembly

$$Q_{loss} = \frac{\Delta T_c}{R_c}$$

$$\delta Q_{loss} = Q_{loss} \sqrt{\left(\frac{\delta \Delta T_c}{\Delta T_c} \right)^2 + \left(\frac{\delta R_c}{R_c} \right)^2}$$

$$\Delta T_c = T_{chip} - T_s$$

$$\delta \Delta T_c = \sqrt{\delta T_{chip}^2 + \delta T_s^2}$$

$$\delta \Delta T_{chip} = \delta T_s = \pm 0.3 \text{ } ^\circ\text{C}$$

R_c was calculated for $T_{chip} = 60 \text{ } ^\circ\text{C}$. δR_c was then assumed to be 5% of R_c .

$$\delta R_c = \pm 36.3 \text{ } ^\circ\text{C/W}$$

2. Power supplied to the chip resistors

$$\delta Power = Power \sqrt{\left(\frac{\delta V_{htr}}{V_{htr}} \right)^2 + \left(\frac{\delta V_{ip}}{V_{ip}} \right)^2 + \left(\frac{\delta R_p}{R_p} \right)^2}$$

$$\delta V_{\text{htr}} = \delta V_{\text{rp}} = \pm 0.000005 \text{ V}$$

$$\delta R_p = \pm 0.05 \text{ } \Omega$$

3. Net power dissipated

$$Q_{\text{net}} = \text{Power} - Q_{\text{loss}}$$

$$\delta Q_{\text{net}} = \sqrt{\delta \text{Power}^2 + \delta Q_{\text{loss}}^2}$$

4. Temperatures for calculation of average heat transfer coefficient

$$\delta T_{\text{avg}} = \pm 0.275 \text{ } ^\circ\text{C}$$

$$\delta T_{\text{sink}} = \pm 0.3 \text{ } ^\circ\text{C}$$

$$\Delta T = (0.55T_{\text{lid}} + 0.45T_{\text{side}}) - T_{\text{sink}}$$

55% of the exposed chip surface area is the lid, and the chip sides make up the remainder. T_{side} is based on the average between the lid temperature and the indicated chip temperature.

$$\delta \Delta T = \sqrt{\delta T_{\text{lid}}^2 + \delta T_{\text{side}}^2 + \delta T_{\text{sink}}^2}$$

$$\delta T_{\text{lid}} = \delta T_{\text{side}} = \delta T_{\text{sink}} = \pm 0.3 \text{ } ^\circ\text{C}$$

5. Average heat transfer coefficient

$$h = \frac{Q_{\text{net}}}{A_{\text{tot}} \Delta T}$$

$$\delta h = h \sqrt{\left(\frac{\delta Q_{\text{net}}}{Q_{\text{net}}}\right)^2 + \left(\frac{\delta \Delta T}{\Delta T}\right)^2}$$

The small uncertainty in A_{tot} is neglected.

6. Nusselt number

$$Nu = \frac{hL}{k}$$

$$\delta Nu = Nu \sqrt{\left(\frac{\delta h}{h}\right)^2 + \left(\frac{\delta L}{L}\right)^2 + \left(\frac{\delta k}{k}\right)^2}$$

The small uncertainty in L is neglected, and δk is assumed to be negligible. Therefore

$$\delta Nu = Nu \frac{\delta h}{h}$$

7. Rayleigh number

$$Ra = GrPr$$

$$Gr = \frac{g\beta L^3 \Delta T}{\nu^2} \quad Pr = \frac{\nu}{\alpha}$$

Neglecting property and length uncertainties,

$$\delta Gr = Gr \frac{\delta \Delta T}{\Delta T}$$

$$\delta Ra = Ra \frac{\delta Gr}{Gr}$$

APPENDIX F. TSE CALIBRATION

The entire NSWC circuit board assembly was placed inside a 400 W Central Scientific Company oven for calibration. The oven had a cylindrically shaped internal volume of 0.11 m³. A platinum resistance thermometer was inserted through a small hole in the top of the oven. A Rosemont Engineering Company galvanometer and commutating bridge, accurate to ± 0.0001 ohm, was used to measure the thermometer's resistance. Temperature was then read from pre-printed NBS calibration data, accurate to ± 0.01 °C. Due to the expected nature of the dielectric liquid temperature fluctuations during the experimental runs, the calibration data was rounded to the nearest 0.1 °C.

This temperature was considered to be the reference temperature for the calibration curve. The V_{BE} voltages were read with a HP-3456A digital voltmeter, accurate to ± 0.000005 V. Data points were taken at four different oven settings, ranging from 19.5 to 95.8 °C. Additional data points were taken as necessary to ensure steady state conditions had been reached. The data points used for the TSE calibration are included in the table below:

channel	60	61	62	63	64	
TSE #	3	4	5	6	8	Temp. (°C)
V_{BE} (V)	0.96907	0.96827	0.96925	0.96954	0.96955	19.5
V_{BE} (V)	0.92845	0.92752	0.92861	0.92875	0.92892	43.6
V_{BE} (V)	0.88470	0.88367	0.88486	0.88479	0.88521	68.6
V_{BE} (V)	0.83666	0.83551	0.83683	0.83697	0.83724	95.8

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